

①  
VOLUME II  
LIGHTWEIGHT VACUUM JACKET FOR  
CRYOGENIC INSULATION

APPENDICIES TO FINAL REPORT

By

D. L. Barclay, J. E. Bell, E. W. Brogen & J. W. Straayer

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J. R. Barber, Project Manager

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15. Supplementary Notes  J. R. BARBER, PROJECT MANAGER					
16. Abstract  <p>This program demonstrated the feasibility of producing a Lightweight Vacuum Jacket using state-of-the-art technology and materials. Design and analytical studies were made on a full scale, i.e., 4.57 m (15 ft.) inside diameter with a volume of 65.1 m<sup>3</sup> (2300 ft.<sup>3</sup>), oms fuel (LH<sub>2</sub>) tank. Preliminary design details were completed for the tank assembly which included an optimized vacuum jacket and MLI system. A half scale LH<sub>2</sub> test model was designed and fabricated. A force/stiffness (F/S) proof test was conducted on the vacuum jacket. The vacuum jacket designed and fabricated on this program achieved a vacuum leak rate of 1 x 10<sup>-5</sup> atmosphere ml of helium per second, sustained approximately 1500 hours of vacuum pressure and experienced 29 vacuum pressure cycles prior to failure.</p> <p style="text-align: center;">* KEY WORDS</p> <p style="text-align: center;">Evaluated MLI System, Vacuum Jacket Space Shuttle Orbiter Oms Fuel (LH<sub>2</sub>) Tank Force/Stiffness (F/S) Proof Test Vacuum Annulus Preconditioning</p>					
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## INTRODUCTION

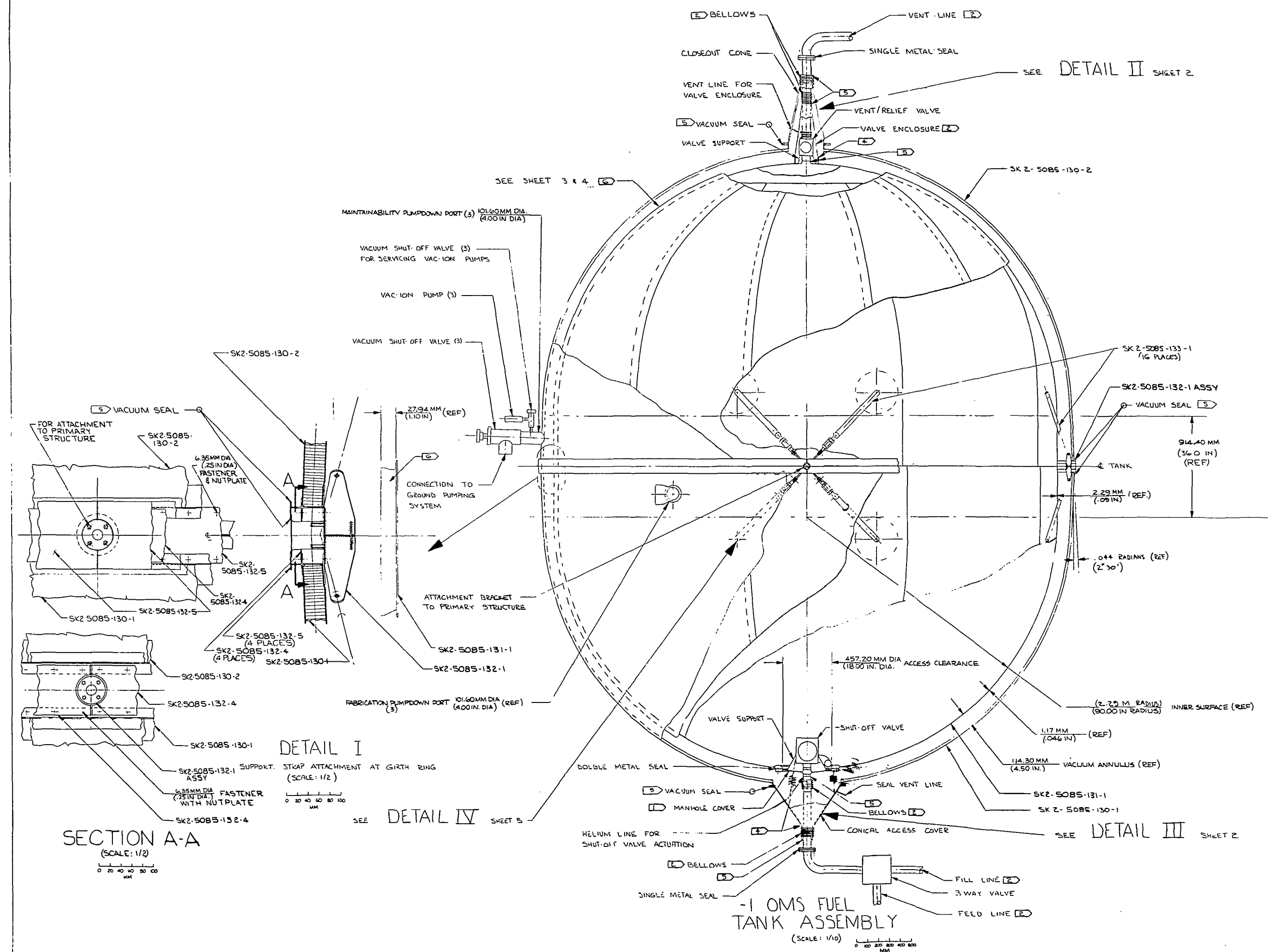
This is a companion document to Volume I, NASA CR 134759, "Final Report."  
The appendices contained herein supplement the technical discussion in that document.



APPENDIX A

OMS FUEL TANK PRELIMINARY DESIGN DRAWINGS

REVISIONS		
9/11/72	10/10/72	
11/10/72	12/10/72	
1/10/73		



# ASSEMBLY AND QUALIFICATION PROCEDURES

- 1.0 DETAILS AND SUBASSEMBLIES
  - a) PRESSURE VESSEL WELD ASSEMBLY. LN<sub>2</sub> COLD SHOCK, PROOF PRESSURE TEST, AND HELIUM LEAK CHECK
  - b) SUPPORT STRAP DETAILS
  - c) GIRTH RING WELD ASSEMBLY AND MACHINING
  - d) VACUUM JACKET SUBASSEMBLIES
  - e) MLI BLANKET SUBASSEMBLIES
  - f) MANHOLE COVER ASSEMBLY (COVER, SHUT-OFF VALVE, AND FEED LINE)
- 2.0 ASSEMBLE PRESSURE VESSEL, SUPPORT STRAPS, GIRTH RING, VENT LINE, AND VENT RELIEF VALVE.
- 3.0 INSTALL MLI BLANKET ON PRESSURE VESSEL. INSULATE SUPPORT STRAPS AND VENT LINE.
- 4.0 INSTALL VACUUM JACKET SUBASSEMBLIES. SEAL JOINTS AT GIRTH RING AND VENT LINE OUTLET FOR VACUUM.
- 5.0 INSTALL MANHOLE COVER, FEED LINE, AND SEAL VENT LINE. INSULATE THE AREA. INSTALL VACUUM JACKET COVER. SEAL JACKET COVER AND FEED LINE OUTLET FOR VACUUM.
- 6.0 HELIUM LEAK CHECK ASSEMBLY. REPAIR AS NECESSARY. PUMPDOWN VACUUM ANNULUS TO  $5 \times 10^{-5}$  TORR. CONDUCT VACUUM DECAY TEST. PRECONDITION WITH HEAT AND VACUUM PUMPING AS NECESSARY TO REACH REQUIRE DECAY RATE. PINCH OFF AND SEAL THE THREE FABRICATION PUMPDOWN PORTS. CLOSE VACUUM VALVES ON THE MAINTAINABILITY PUMPDOWN PORTS.
- 7.0 CONDUCT BOIL-OFF TEST.

# NOTES

1. ALUMINUM ALLOY 2219-T81
2. STAINLESS STEEL
3. DIFFUSION BOND
4. WELD
5. MULTILAYER INSULATION (MLI)
  - a) ALUMINIZED MYLAR (.15 MIL)/DACRON NET (B4A) - 2.75 LAYERS/MM (.75 LAYERS/INCH)
  - b) OUTER 2.54MM (.10 IN) LAYER - ALUMINIZED KAPTON (D.30 MIL)/DACRON NET (B4A) - 2.75 LAYERS/MM (.75 LAYERS/INCH)
6. HOOK & PILE FASTENER - VELCRO OR EQUIVALENT
7. DACRON THREAD
8. ZYTEL 103 HS-L NYLON RESIN

Figure A-1

N43-1-58-16  
 11/10/72  
 OMS FUEL TANK  
 ASSEMBLY  
 01205 SK2-5085-129

REVISIONS	
5/11/72	11/10/72
12/10/72	1/10/73

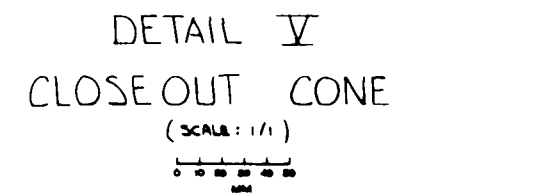
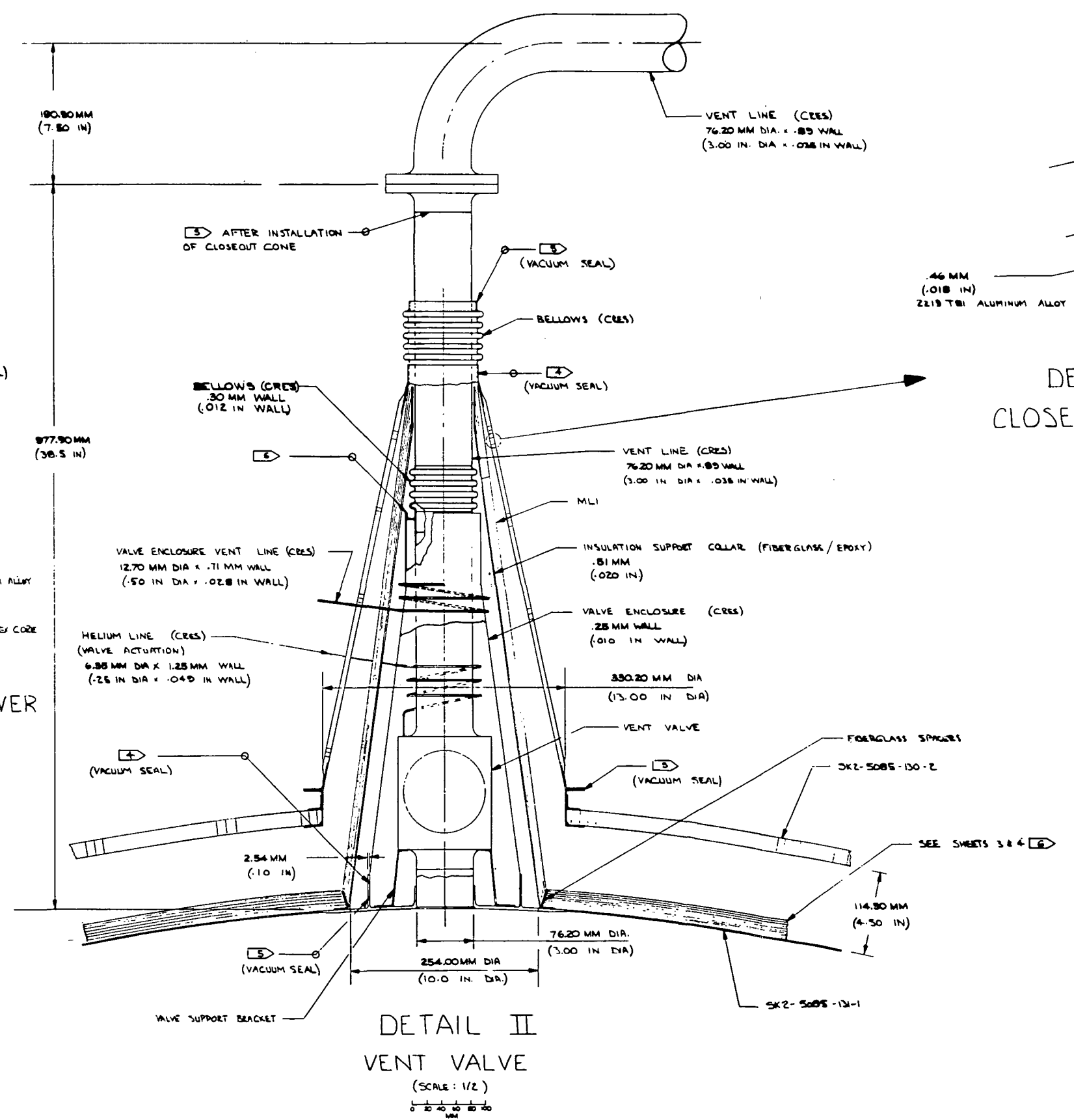
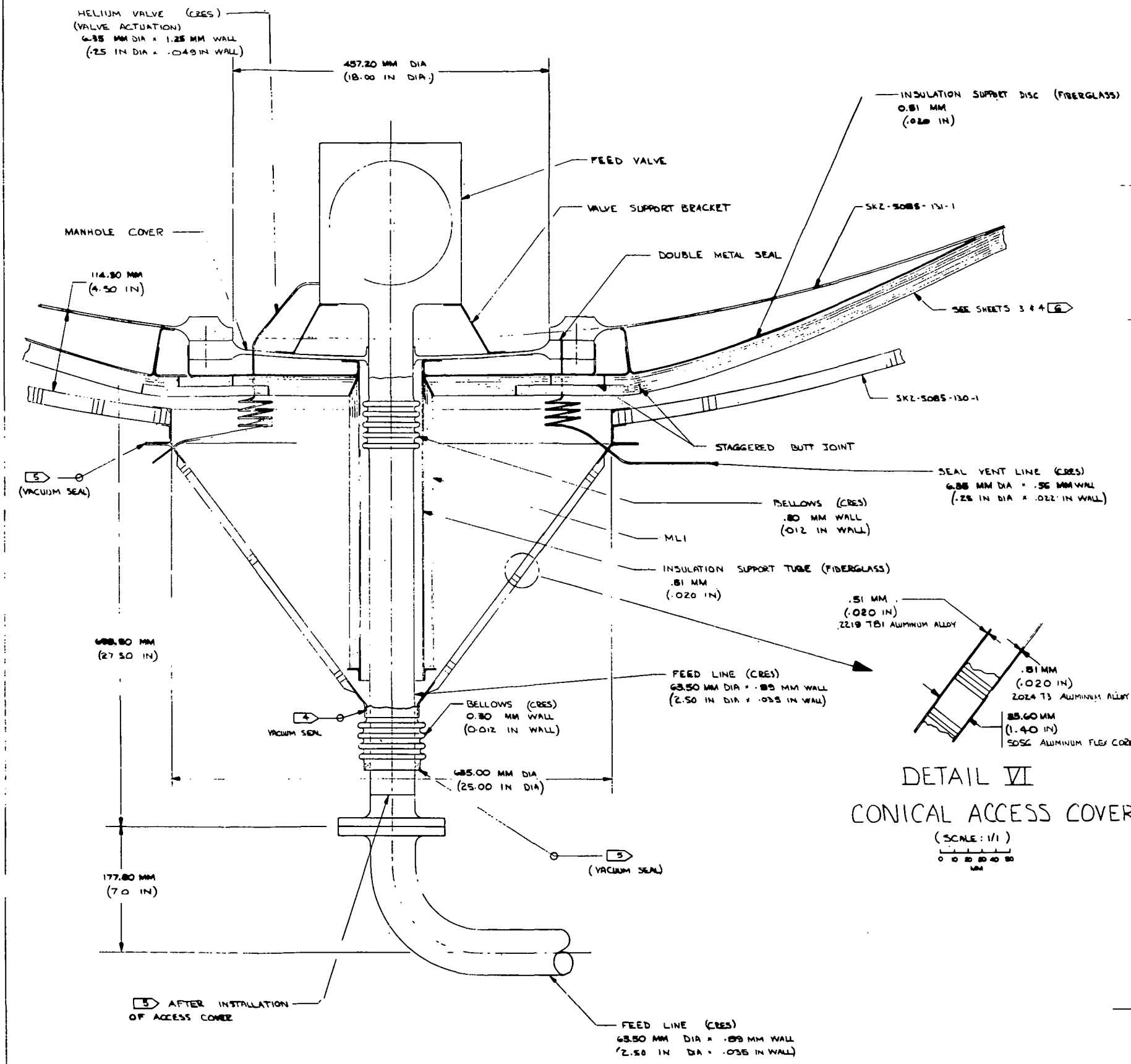


Figure A-2

NAS3-15848

DL DARCLAY BLSR

DL DARCLAY BLSR

OMS FUEL TANK  
MANHOLE & PLUMBING  
PENETRATIONS

8205SK2-5085-129

NOTED

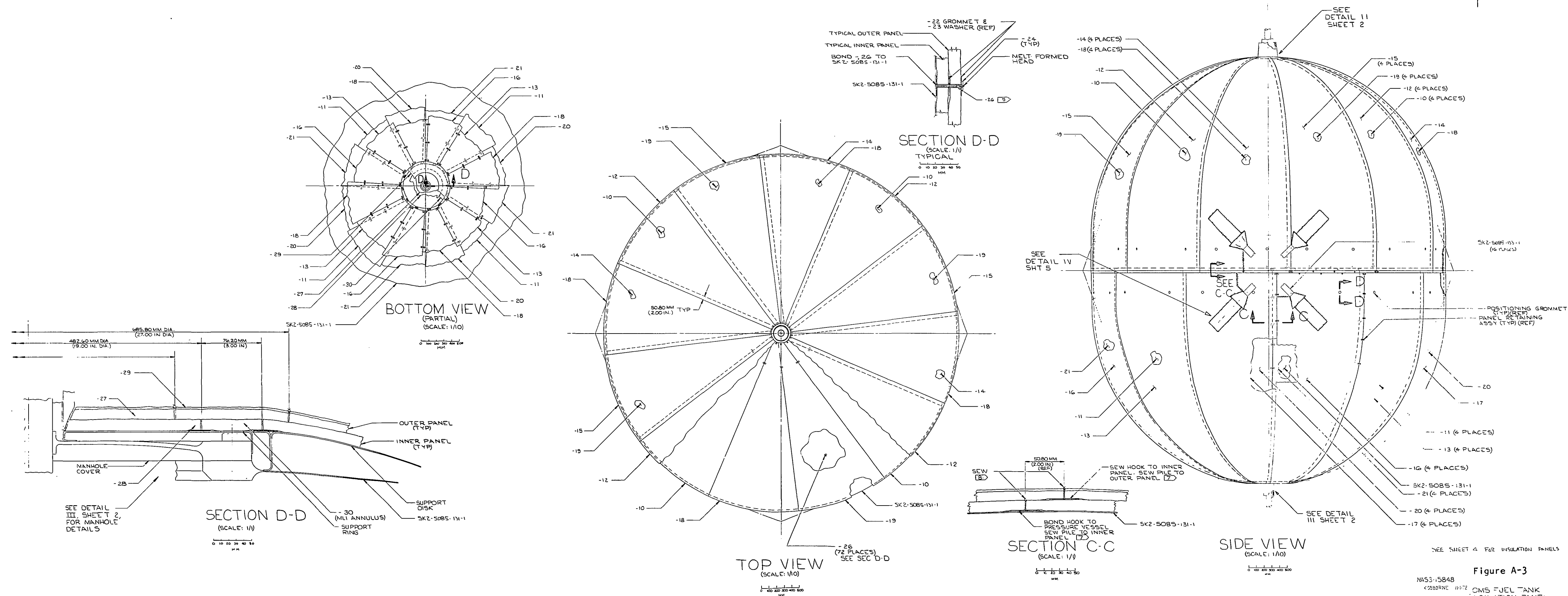


Figure A-3

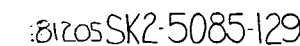
NAS3-15848

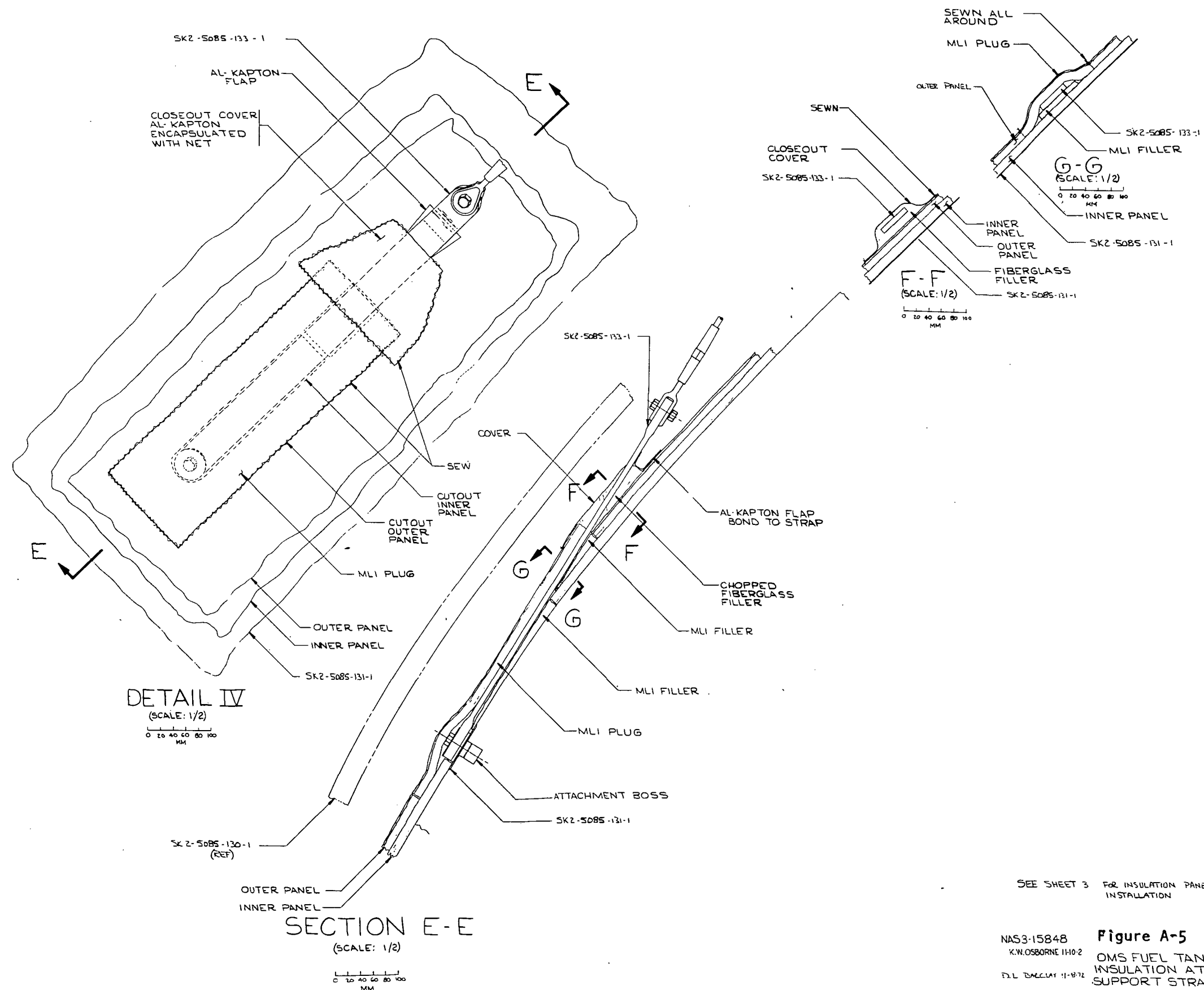
CROSSBORNE 11-172

CMS FUEL TANK  
INSULATION PANEL  
INSTALLATION

51205SK2-5085-129
























21205SK2-5085-129-2H10

FIGURE A-6

1 of 5

PARTS LIST		THE <b>BOEING</b> COMPANY CORPORATE OFFICES SEATTLE, WASHINGTON 98124								CONTRACT NUMBER NAS 3-15848		CODE IDENT NUMBER 81205		PL SK2-5085-130		SHEET 1		REV LTR	
FAWR NUMBER	FIND NUMBER	QTY REQD	QTY REQD	QTY REQD	QTY REQD	QTY REQD	QTY REQD	QTY REQD	CODE IDENT NUMBER	PART OR IDENTIFYING NUMBER	NOMENCLATURE OR DESCRIPTION	ZONE	MATERIAL AND SPECIFICATION	T R A C E	NOTES	REV LTR			
								-2	-1		-1	HEAD ASSEMBLY							
								-			-2	HEAD ASSEMBLY							
								1	1		-3	CORE							
								-	3		-4	GORE-INNER							
								3	-		-5	GORE-INNER							
								-	5		-6	GORE-INNER							
								5	-		-7	GORE-INNER							
								-	3		-8	GORE-OUTER							
								3	-		-9	GORE-OUTER							
								-	5		-10	GORE-OUTER							
								5	-		-11	GORE-OUTER							
								-	1		-12	ANNULUS							
								-	1		-13	ANNULUS							
								1	-		-14	ANNULUS							
								1	-		-15	ANNULUS							
								-	8		-16	FILLER							
								8	-		-17	FILLER							
								-	1		-18	FLANGE							
								1	-		-19	FLANGE							
								-	8		-20	STRIP-INNER							
								8	-		-21	STRIP-INNER							

PARTS LIST		THE <b>BOEING</b> COMPANY CORPORATE OFFICES SEATTLE, WASHINGTON 98124								CONTRACT NUMBER NAS3-15848		CODE IDENT NUMBER 81205		PLSK2-5080-130		SHEET 2		REV LTR	
EWNR NUMBER	FIND NUMBER	QTY REQD	QTY REQD	QTY REQD	QTY REQD	QTY REQD	QTY REQD	QTY REQD	CODE IDENT NUMBER	PART OR IDENTIFYING NUMBER	NOMENCLATURE OR DESCRIPTION	ZONE	MATERIAL AND SPECIFICATION	TRACE	NOTES	REV LTR			
						-34	-27	-2									-1		
								8		- 22	STRIP-OUTER								
								8		- 23	STRIP-OUTER								
								8		- 24	STRIP-EXTERNAL								
								8		- 25	STRIP-EXTERNAL								
								3	3	- 27	TUBE ASSY								
								1	-	- 28	TUBE								
								1	-	- 29	FLANGE								
								3	3	- 30	COLLAR								
								8	8	- 31	DOUBLER								
								-	3	- 32	ANGLE-CLOSEOUT								
								3	-	- 33	ANGLE-CLOSEOUT								
								1	1	- 34	RING ASSY								
						3	-	-	-	- 35	SEGMENT								
								8	8	- 36	DOUBLER								
								8	8	- 38	FILLER								
								-	8	- 39	FILLER								
								8	-	- 40	FILLER								
										464HE1032-16	INSERT, POTTED								
										- 41									



ASSEMBLY AND QUALIFICATION PROCEDURES  
- 1 & - 2, HEAD ASSEMBLY

REV  
STD

1.0 ASSEMBLE INNER GORES - 4, - 5, - 6, - 7; INNER STRIPS - 20, - 21; CLOSE OUT FLANGE - 18, - 19; ANNULI - 12, - 13, - 14, - 15; FILLERS - 16, - 17 & - 38; TUBE ASSY - 27; GIRTH RING ASSY - 34.  
PER [8]

2.0 HELIUM LEAK CHECK INNER SKIN ASSY AT  $1.333 \text{ cN/m}^2$  ( $1 \times 10^{-5}$  TORR)

3.0 FILL GORE SEAM GAPS FLUSH WITH XA3919 (3MCo)

4.0 BOND STRIPS - 22, - 23; - 36 DOUBLERS PER [8]

5.0 HELIUM LEAK CHECK (2.0)

6.0 APPLY METLBOND 329 TO THE OUTER SURFACE OF THE INNER SKIN PER [4]

ASSEMBLE INNER ANGLE SEG - 36; CORE - 3; APPLY METLBOND 329 TO - 36

INNER ANGLE SEG FAYING SURFACES & CONTACT SURFACES WITH - 3.

WIRE CORE (- 3) TOGETHER AT SEAMS. SPLICE CORE WITH [5] THEN BOND PER [4].

7.0 HELIUM LEAK CHECK (2.0)

8.0 REINFORCE GIRTH CORE WITH [5] COMPLETELY AROUND CIRCUMFERENCE AS SHOWN.

9.0 APPLY METLBOND 329 TO THE INNER SURFACE OF THE OUTER SKIN GORES - 8, - 9, - 10, - 11; OUTER STRIPS - 24, - 25; - APPLY TO - 31 DOUBLERS, INNER CONTACT SURFACES. PER [4] ASSEMBLE TO CORE THEN BOND PER [4].

10.0 HELIUM LEAK CHECK (2.0)

FIGURE: A-8

SIZE B	CODE IDENT NO. 81205	SK2-5085-130
SCALE	SHEET 3 OF 5	

1 2024-T81 ALUM PER QQ-A-250/4 3

2 FLEX-CORE 5056/F40-.0014, 33.6kg/m<sup>3</sup>  
(2.1 # FT<sup>3</sup>) ALUMINUM HONEYCOMB- HEXCEL  
PRODUCTS INC - SPLICE CORE GORE'S  
WITH 5

3 CLEAN PER BAC 5514

4 CLEAN & BOND PER DOC DG-23841 TN.  
METLBOND 329 ADHESIVE (NARMCO  
MATERIALS DIV., WHITTACER CORP.) SHALL BE  
APPLIED TO FAYING SURFACES NOTED:

5 STRUCTURAL FOAMING ADHESIVE PER BAC  
5-90, TYPE 2, CLASS 350, GRADE 50.

6 ALL RAW MATERIALS, SUBASSEMBLIES  
& ASSEMBLYS SHALL BE PROTECTED  
FROM OIL & PARTICLE CONTAMINATION.  
ASSEMBLY SHALL BE DONE IN A DUST FREE  
ROOM.

7 ALUMINUM 2219-T6 OR T62, PER BMS7-110B  
OR BMS7-108

8 BOND WITH XA3919 (3M Co) APPLY BY  
BRUSH COATING BOTH SURFACES.  
AIR DRY THE SURFACES FOR 15 MIN.  
FOLLOWED BY A 45 MIN. DRYING CYCLE  
AT 380°K (225°F). CURE BOND AT 450°K  
(350°F) FOR 3600 SEC AT 344.74 kN/m<sup>2</sup>  
(50 PSI).

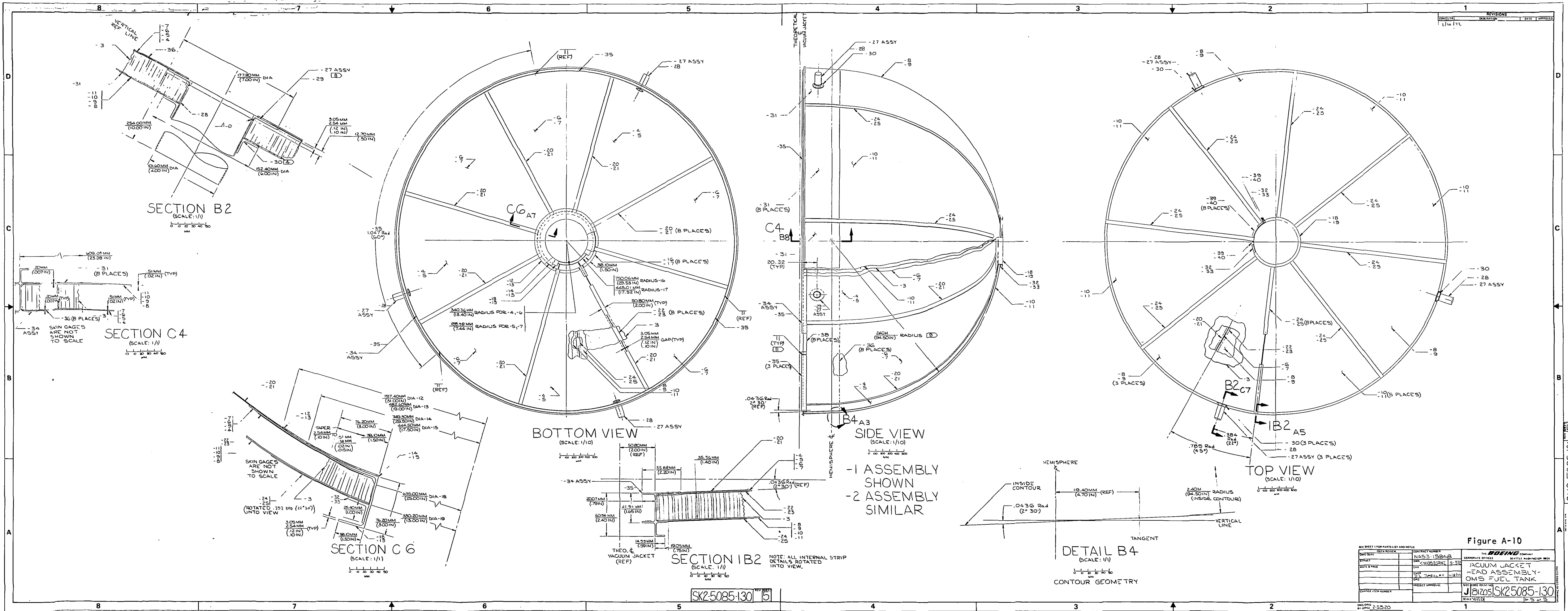
9 DEVIATION FROM RADIUS  $\pm .051\text{cm}$   
(.02 IN) PROVIDED THE DISCONTINUITIES  
DO NOT EXCEED  $\pm .051\text{cm}$  IN 25.4cm  
(10.0 IN) LENGTH IN ANY DIRECTION ALONG  
THE SURFACE

10 BMS 5-25 GRADE 3, APPLY PER BAC  
5514-590.

11 FUSION WELD PER BAC 5935, CLASS A.  
EXCEPT RADIOGRAPHIC INSPECTION IS WAIVED.  
PENETRANT INSPECT PER BAC 5423.  
HELIUM LEAK CHECK  $1.333\text{cN/m}^2$  ( $1 \times 10^{-5}$  TORR)

FIGURE: A-9

SIZE B	CODE IDENT NO. 81205	SK2-5085-130
SCALE	SHEET 4 OF 5	



SK2-5085-130

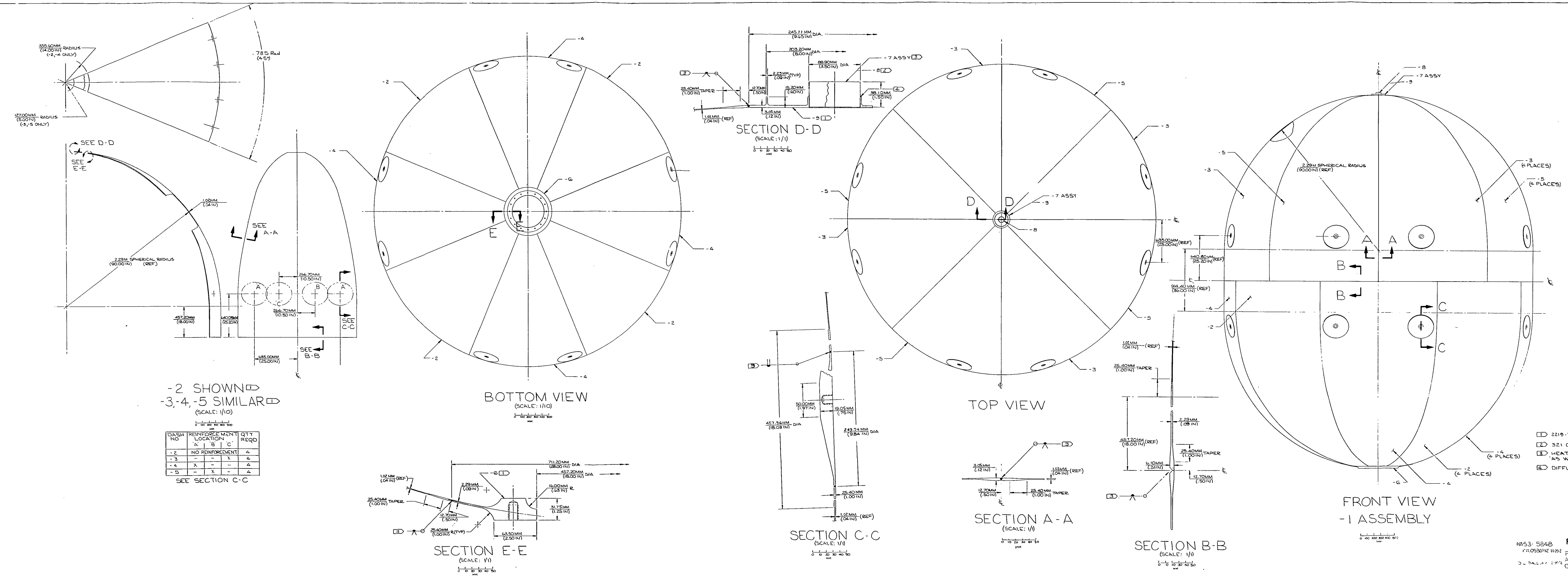
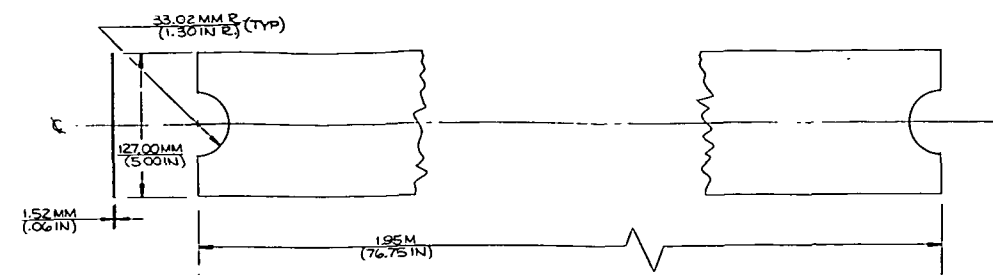
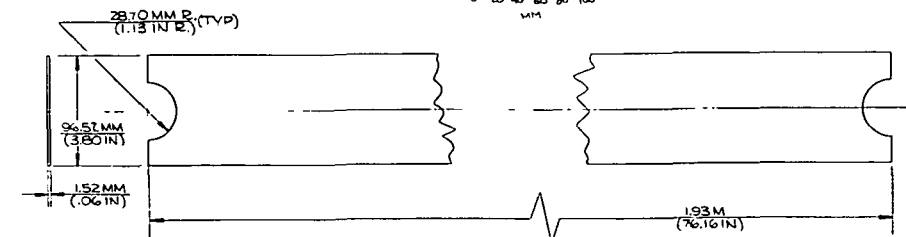


Figure A-11  
NAS-5848  
PRESSURE VESSEL  
ASSEMBLY  
OMS FUEL TANK  
812053K2-5085-131

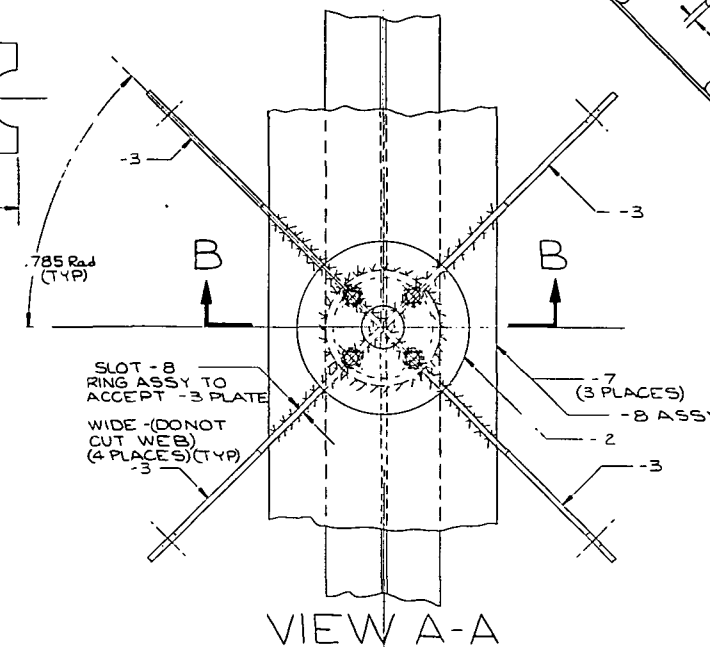




-5  
(SCALE: 1/2)  
(4 REQ'D)  
2219-T0 AL SHEET

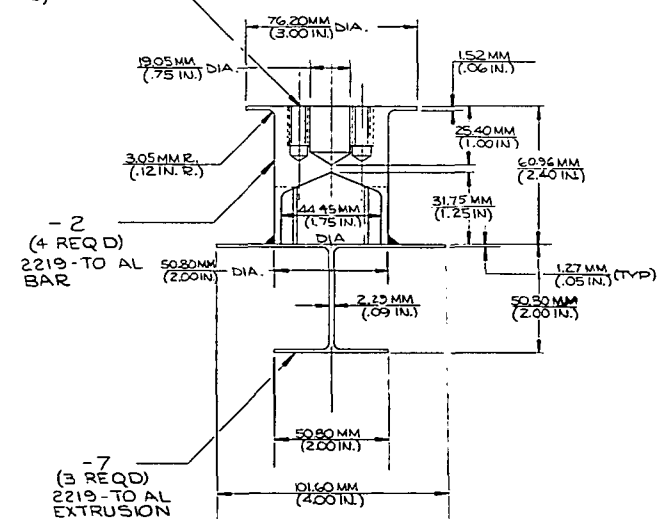


-4  
(SCALE: 1/2)  
(4 REQ'D)  
2219-T02 AL SHEET

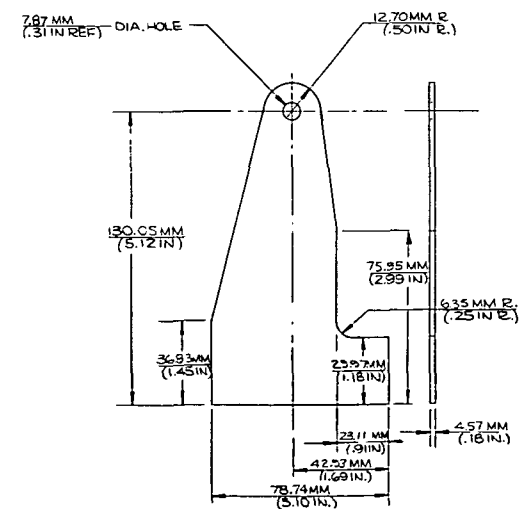


VIEW A-A  
(SCALE: 1/1)

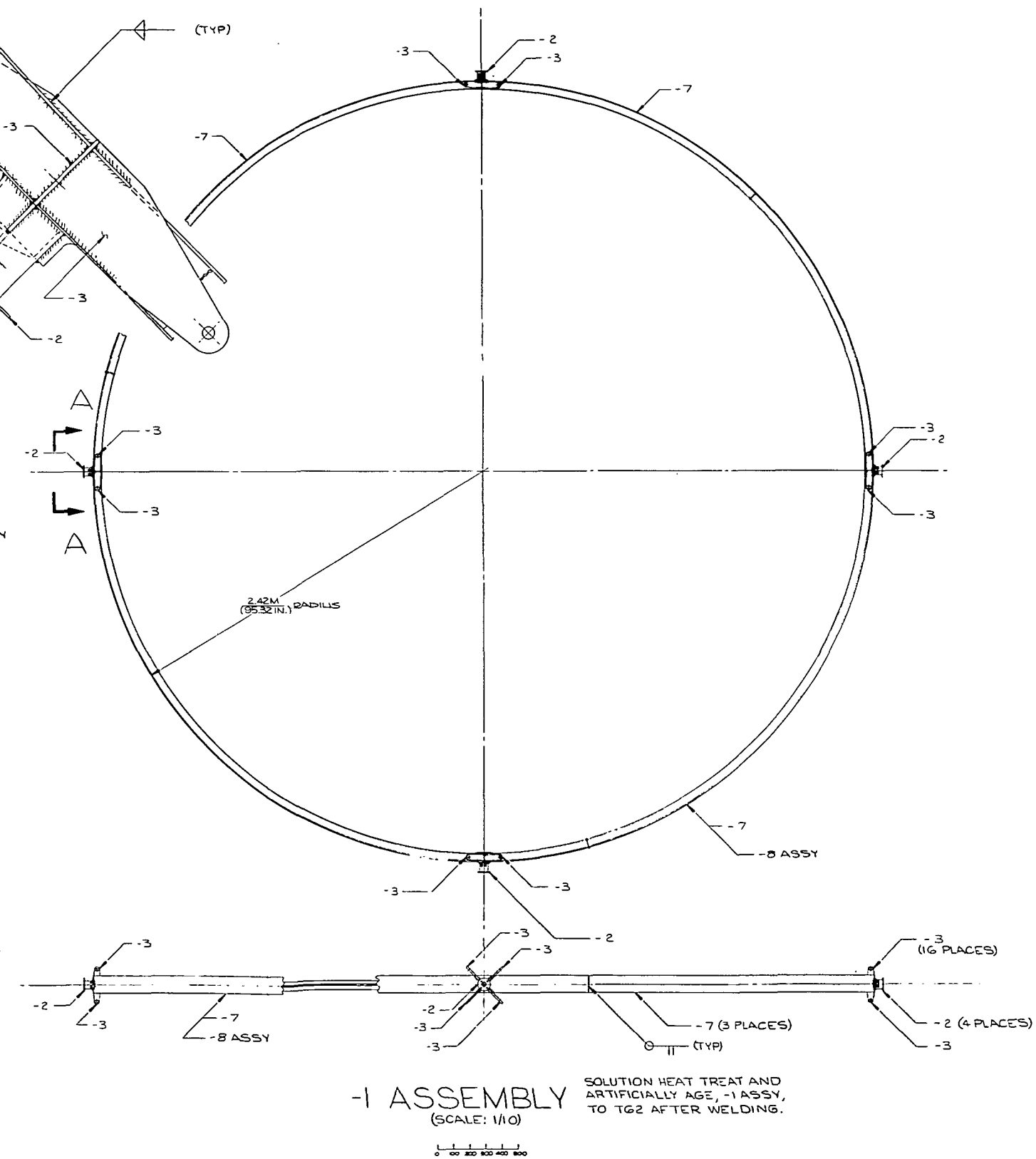
THREADED INSERT  
7.87 MM DIA. BOLT  
(.31 IN)  
(4 PLACES)



SECTION B-B  
(SCALE: 1/1)



-3  
(SCALE: 1/1)  
(16 REQ'D)  
2219-T0 AL PLATE



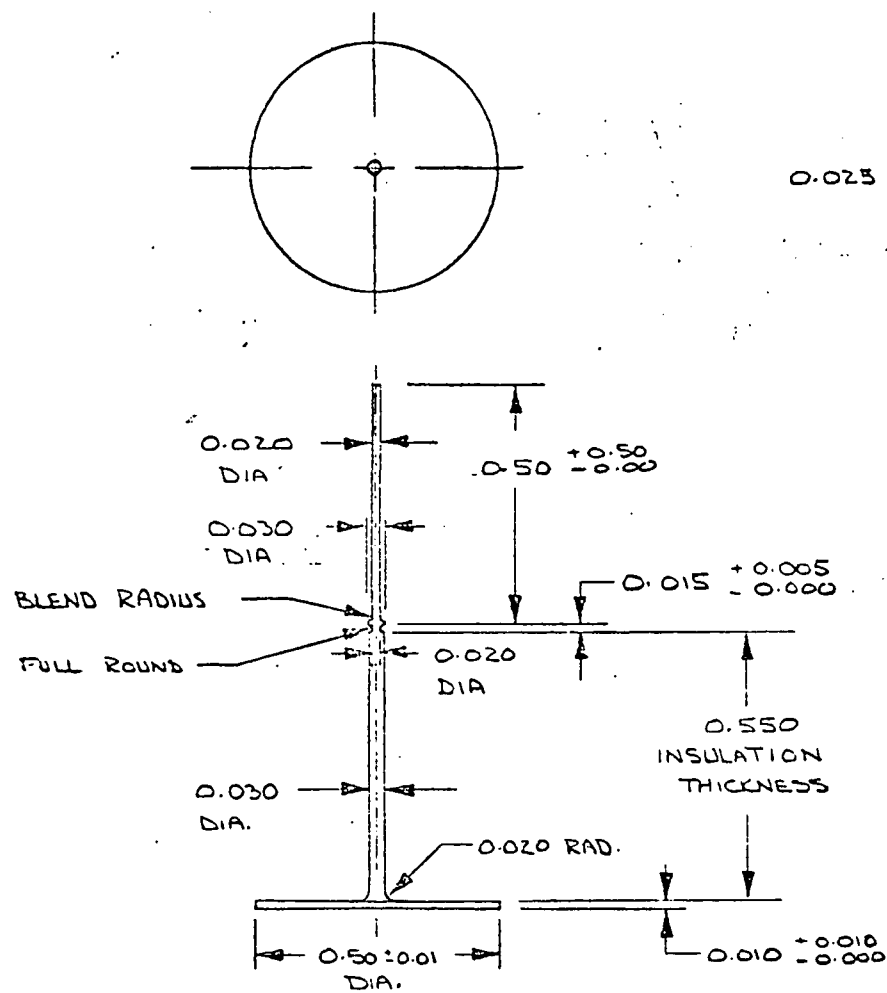
-1 ASSEMBLY  
(SCALE: 1/10)

SOLUTION HEAT TREAT AND  
ARTIFICIALLY AGE, -1 ASSY,  
TO T62 AFTER WELDING.

Figure A-14

NAS3-15848  
WOSBORNE 11-2-72  
D.L. BAZELAY 12-8-72  
GIRTH RING ASSEMBLY  
OMS FUEL TANK

81205 SK2-5085-132



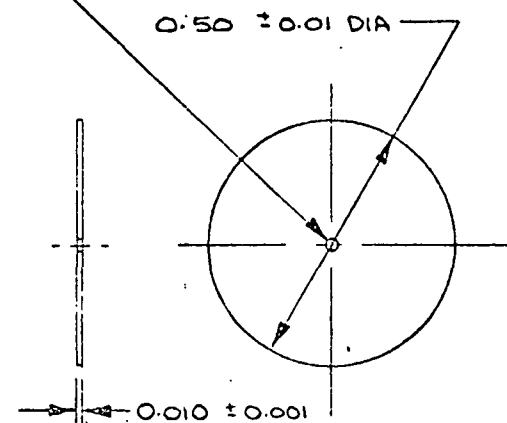
- 1

BUTTON - PIN STUD

MAT'L: ZYTEL 103 HSI-L NYLON RESIN

TOLERANCE ON X.XXX IS  $\pm$  0.002

(SCALE 4/1)



- 2

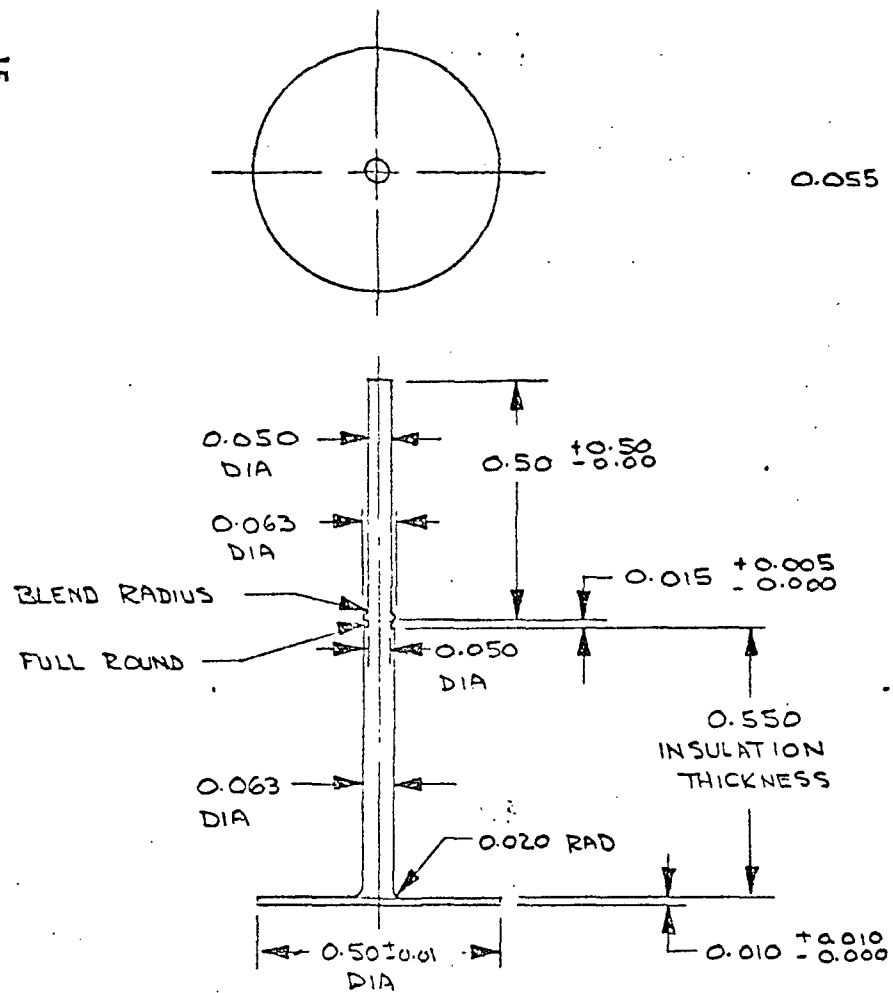
BUTTON

MAT'L: ZYTEL 103 HSI-L NYLON RESIN

(SCALE 4/1)

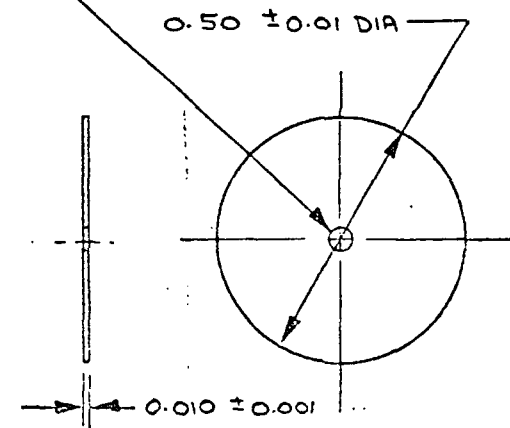
FIGURE A-13

SK 2-5085-147 — MLI FASTENERS



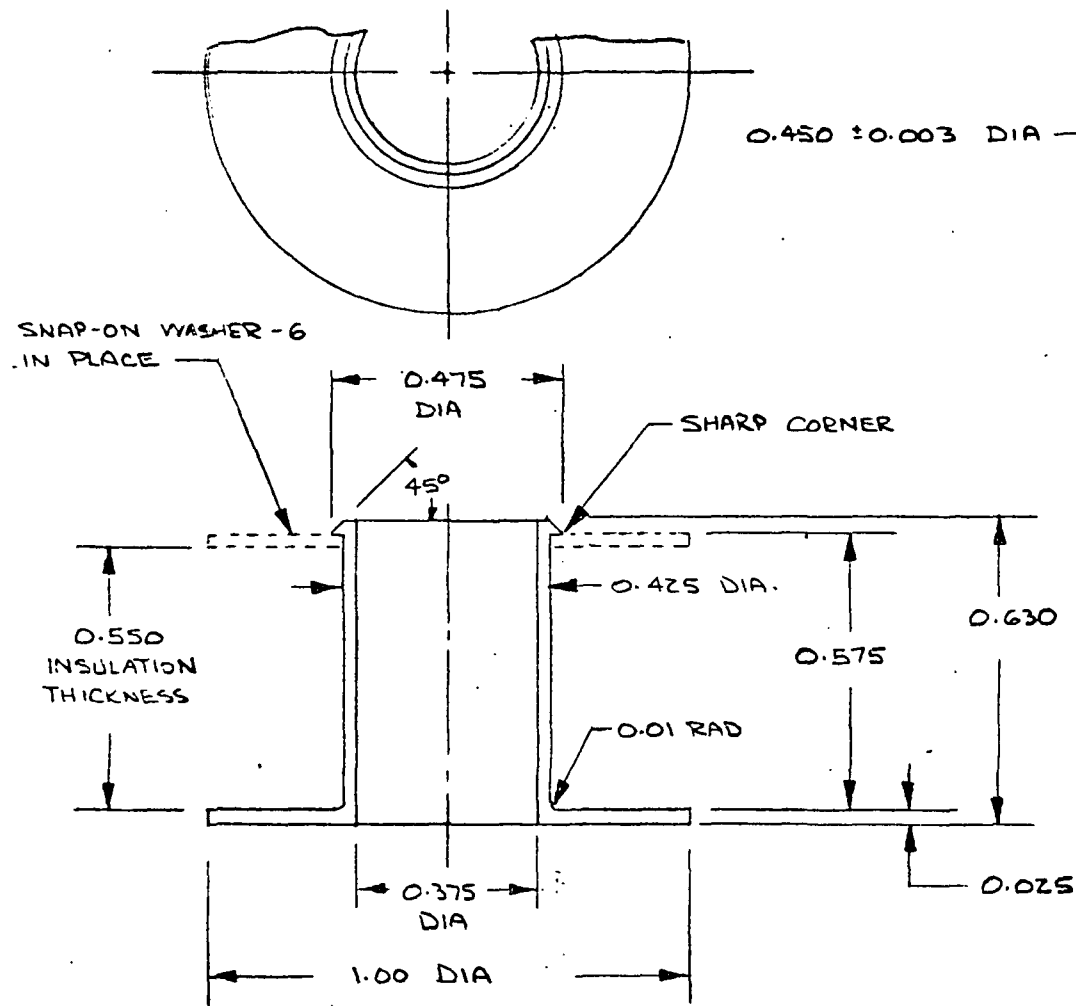
BUTTON - PIN STUD  
 MAT'L: ZYTEL 103 HSI-L NYLON RESIN  
 TOLERANCE ON X.YXX IS  $\pm$  0.002

(SCALE 4/1)



BUTTON  
 MAT'L: ZYTEL 103 HSI-L NYLON RESIN  
 (SCALE 4/1)

FIGURE A-14  
 SKZ-5085-147-MLI FASTENERS



-5

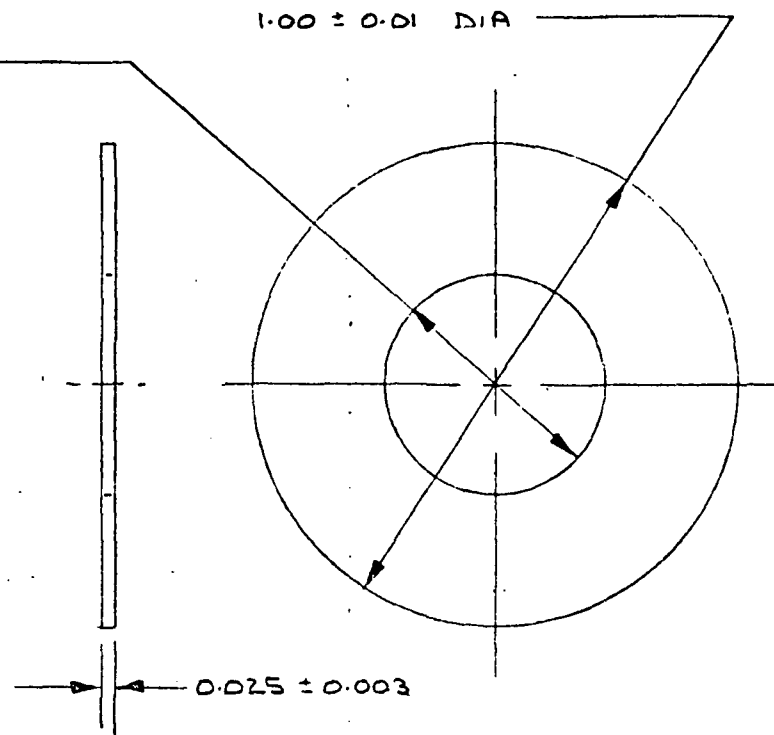
### POSITIONING GROMMET

MAT'L: ZYTEL 103 HSI-L NYLON RESIN

TOLERANCE ON X.XX IS ± 0.01

ON X.XXX IS ± 0.003

(SCALE 4/1)



-6

### SNAP-ON WASHER

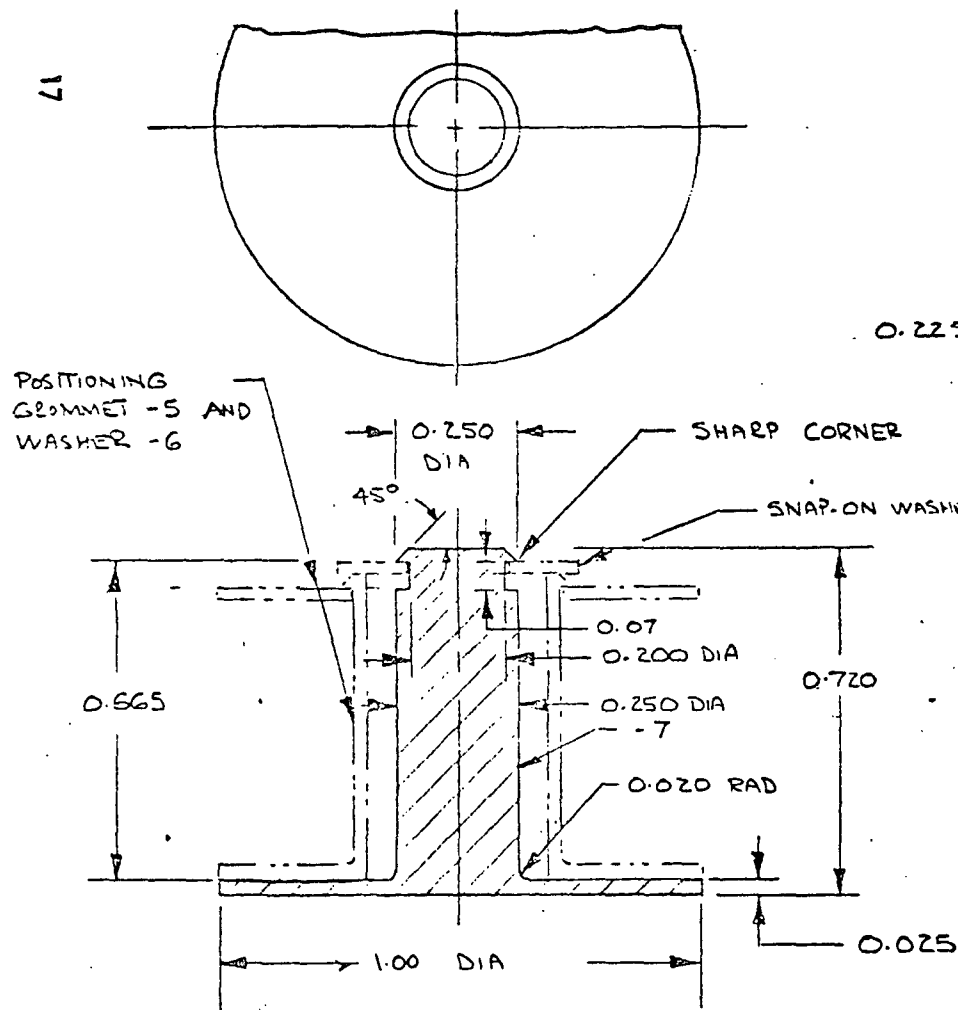
MAT'L: ZYTEL 103 HSI-L NYLON RESIN

(SCALE 4/1)

FIGURE A-15

SK2-5085-147—MLI FASTENERS





### LOCATING PIN

MAT'L: ZYTEL 103 HSI-L NYLON RESIN

TOLERANCE ON X.XX IS ± 0.01

ON X.XXX IS ± 0.003

(SCALE 4/1)

FIGURE A-16  
SK2-5085-147—MLI FASTENERS



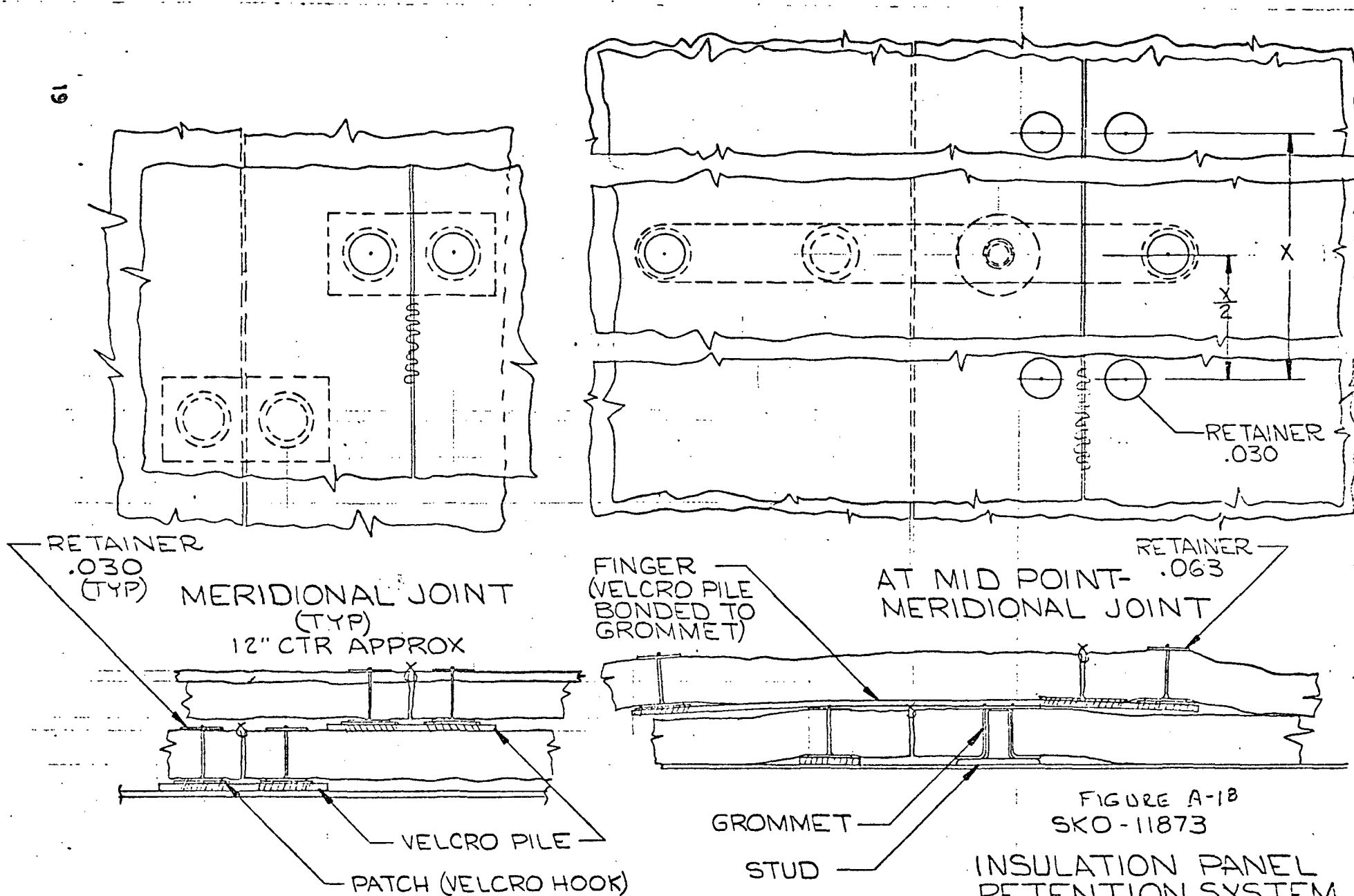


FIGURE A-18  
SKO-11873  
INSULATION PANEL  
RETENTION SYSTEM

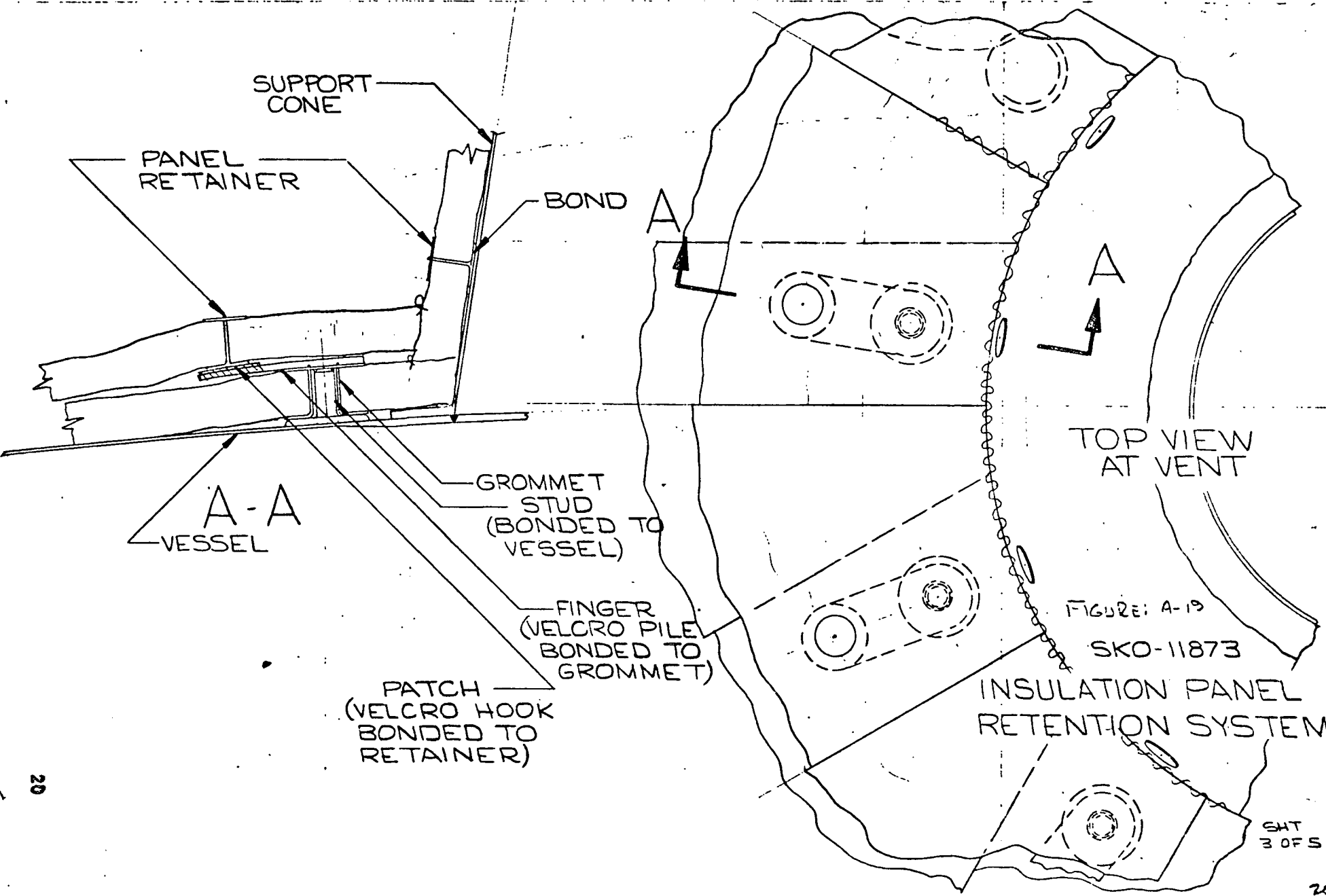


FIGURE: A-19

SKO-11873

INSULATION PANEL  
RETENTION SYSTEM

SHT  
3 OF 5

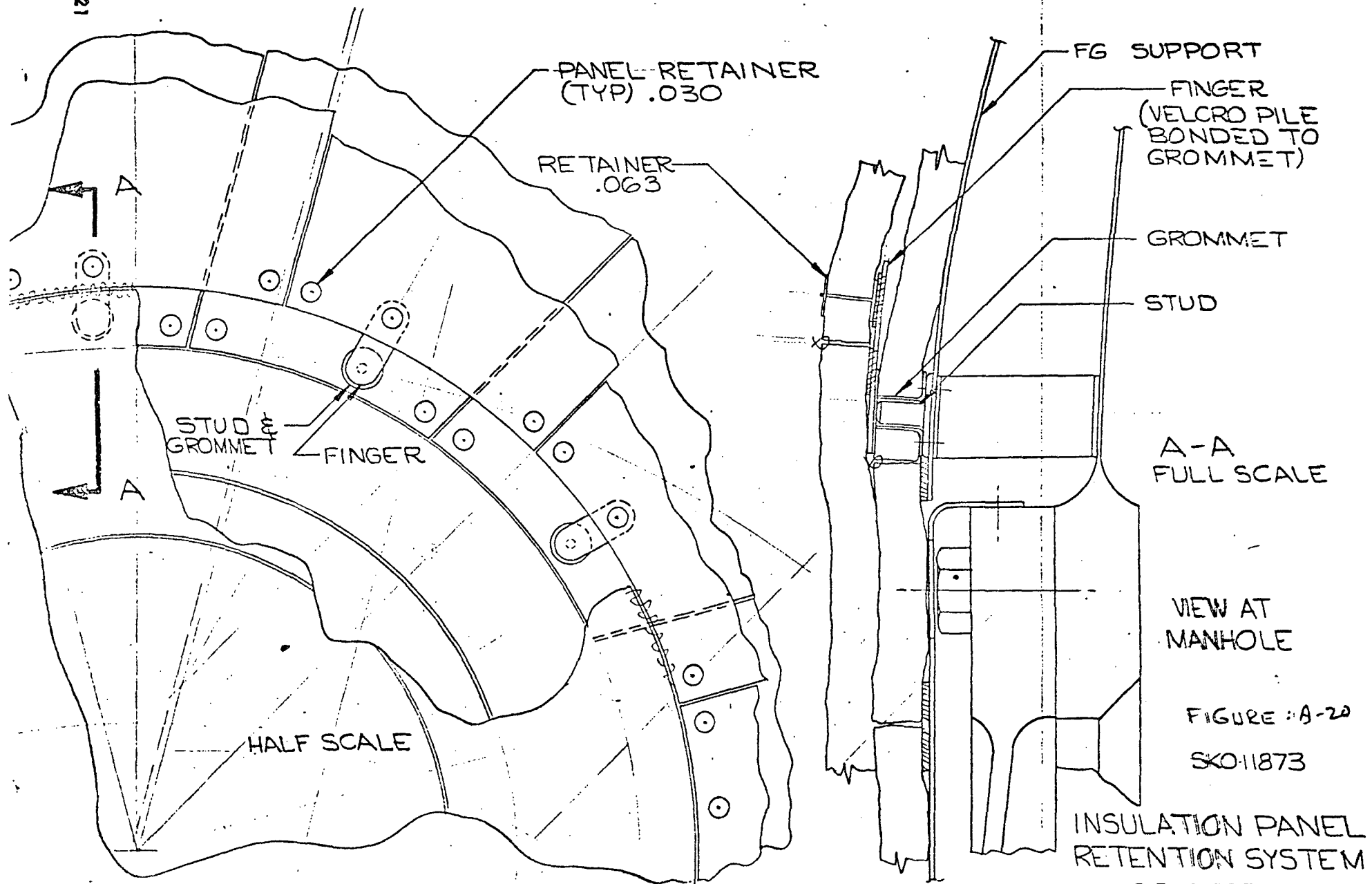
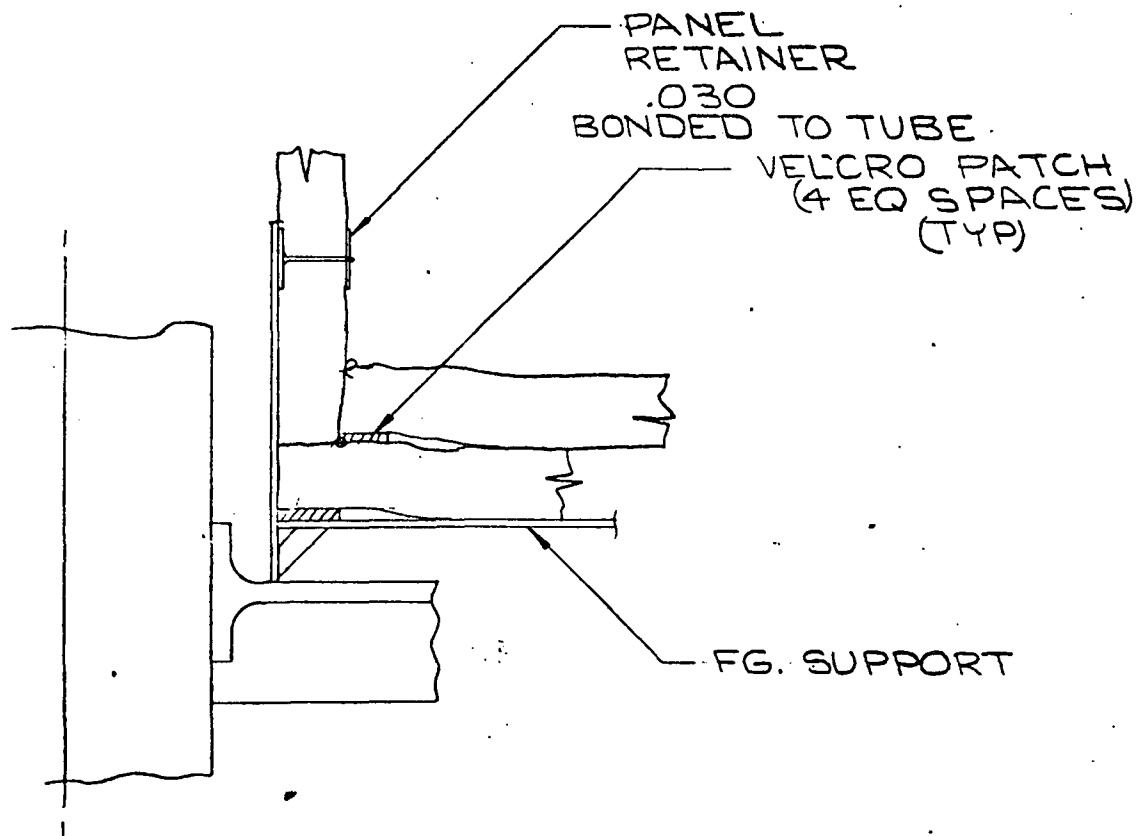


FIGURE A-20

SKO11873

INSULATION PANEL  
RETENTION SYSTEM



VIEW AT MANHOLE  
FEED LINE PENATRATION

FIGURE: A-21

SKO-11873

INSULATION PANEL  
RETENTION SYSTEM

SHT 5 OF 5

## APPENDIX B

### MODIFIED THEORETICAL EFFECTIVE GAS CONDUCTIVITY

APPENDIX B  
MODIFIED THEORETICAL EFFECTIVE GAS CONDUCTIVITY

In Reference 1 an expression for elemental heat transfer due to free molecule gas conduction was derived for open net MLI systems employing layer densities from 2.0 to 4.0 layers per mm (51 to 102 layers per inch). That expression is

$$q_x = \alpha k_g \left( \frac{h}{h + \ell} \right) \frac{dT^*}{dx} \quad (8)^*$$

where

$q_x$  = heat flux in the  $x$  direction

$\alpha$  = accommodation coefficient

$k_g$  = true gas conductivity

$h$  = thickness of one shield plus one space (layer interval)

$\ell$  = mean free path

$T$  = temperature

$x$  = MLI thickness dimension

After substitution of curve-fit formulas expressing the gas property parameters of Equation (8) in terms pressure, temperature and molecular weight, the equation becomes

$$q_x = \frac{a_0 T^n a_1 T^m h}{\left[ \frac{8.6 a_2 T^{0.5}}{pM^{.5}} \right]} = \frac{dT}{dx} \quad (10)$$

\*Equation numbers, units and nomenclature (except where conflicting with symbols used elsewhere in this report) are those of Reference 1.



for which the gas-dependent constants are:

Gas	$a_0$	$n$	$a_1$	$m$	$a_2$	$o$	$M$
H <sub>2</sub>	2.66	-.42	$.971 \times 10^{-5}$	.92	$1.5 \times 10^{-6}$	.7	2.016
He	1.17	-.333	$2.35 \times 10^{-5}$	.74	$5.0 \times 10^{-6}$	.65	4.003
N <sub>2</sub>	1.0	0	$1.38 \times 10^{-6}$	.92	$1.2 \times 10^{-6}$	.9	28.013

The values for  $a_2$  and  $o$  (gas viscosity constants) were not listed directly in Reference 2 but were derived here from the curves of the reference.

With  $h \ll \lambda$  (free molecule regime), Equation (10) may be simplified to

$$q_x = \left[ C(T)^S \text{ ph} \right] \frac{dT}{dx} \quad (12)$$

or

$$q_x = \frac{C p T^S}{\bar{N}} \frac{dT}{dx}$$

where

$$C = f(a_0, a_1, a_2, M)$$

$$S = f(n, m, o)$$

$$\bar{N} = \text{number of layers per unit of thickness}$$

$$\bar{N} = 1/h$$

In Reference 1, Equation (12) was integrated under the assumption that  $T$  varied with  $x$  but  $p$  was independent of  $x$ . The result was

$$q_g = \frac{C_p}{N_o(S+1)} \left[ T_H^{S+1} - T_C^{S+1} \right] \quad (13)$$

or

$$q_g = \frac{C_p}{x\bar{N}(S+1)} \left[ T_H^{S+1} - T_C^{S+1} \right]$$

For the free molecule regime, however, Knudsen's principle (Reference 3) states that

$$\frac{\partial}{\partial x} \left( \frac{\rho}{T^{.5}} \right) = 0$$

If it can be assumed that

$$\frac{\partial}{\partial x} (pT^S) \approx \frac{\partial}{\partial x} (pT^{-.5}) = 0$$

then from Equation (12)

$$q_g = \frac{C}{N} pT^S (T_H - T_C)$$

Because of the independence of  $(pT^S)$  of  $x$ , the non-subscripted  $p$  and  $T$  may be evaluated at any consistent location within the MLI, usually most conveniently at the hot surface. The effective gas conductivity, which follows directly from the preceding expression, has exactly the same form as the previously used semi-empirical formula,

$$k_g = \frac{C}{N} pT^S \quad (A)$$

The gas-dependent constants of Equation (10) were used to evaluate  $C$  and  $S$  for use in the above formula. The values, along with the corresponding values for the previous, semi-empirical effective conductivity expression are shown below.

Gas	$\frac{C, \text{ watt-m}}{\text{mm-N-K}^{S+1}}$	$\left( \frac{C, \text{ Btu}}{\text{ft}^2\text{-hr-Torr-R}^{S+1}} \right)$	$S$
H <sub>2</sub>	.244	(7610)	-.70
He	.111	(3460)	-.74
N <sub>2</sub>	.0575	(1790)	-.48
N <sub>2</sub> , He	.0266	(828)	-.5
(Previous Semi-Emp. Formula)			

The newly derived theoretical values of  $S$  fail to satisfy exactly the assumption that

$$\frac{\partial}{\partial x} (pT^S) \approx \frac{\partial}{\partial x} (pT^{.5})$$

but are close enough to the ideal value of 0.5 to offer hope as to the validity of the theoretical formula for  $k_g$ . In order to test this validity predictions were made for comparison with effective gas conductivity test data of reference 4. Results of this comparison, shown in Figure A-1, support the validity of the theoretical expression, Equation (A), at least for  $N_2$ . On the basis of this correlation, the validity for  $H_2$  was assumed and Equation (A) was adopted for subsequent thermal analyses in the current program.

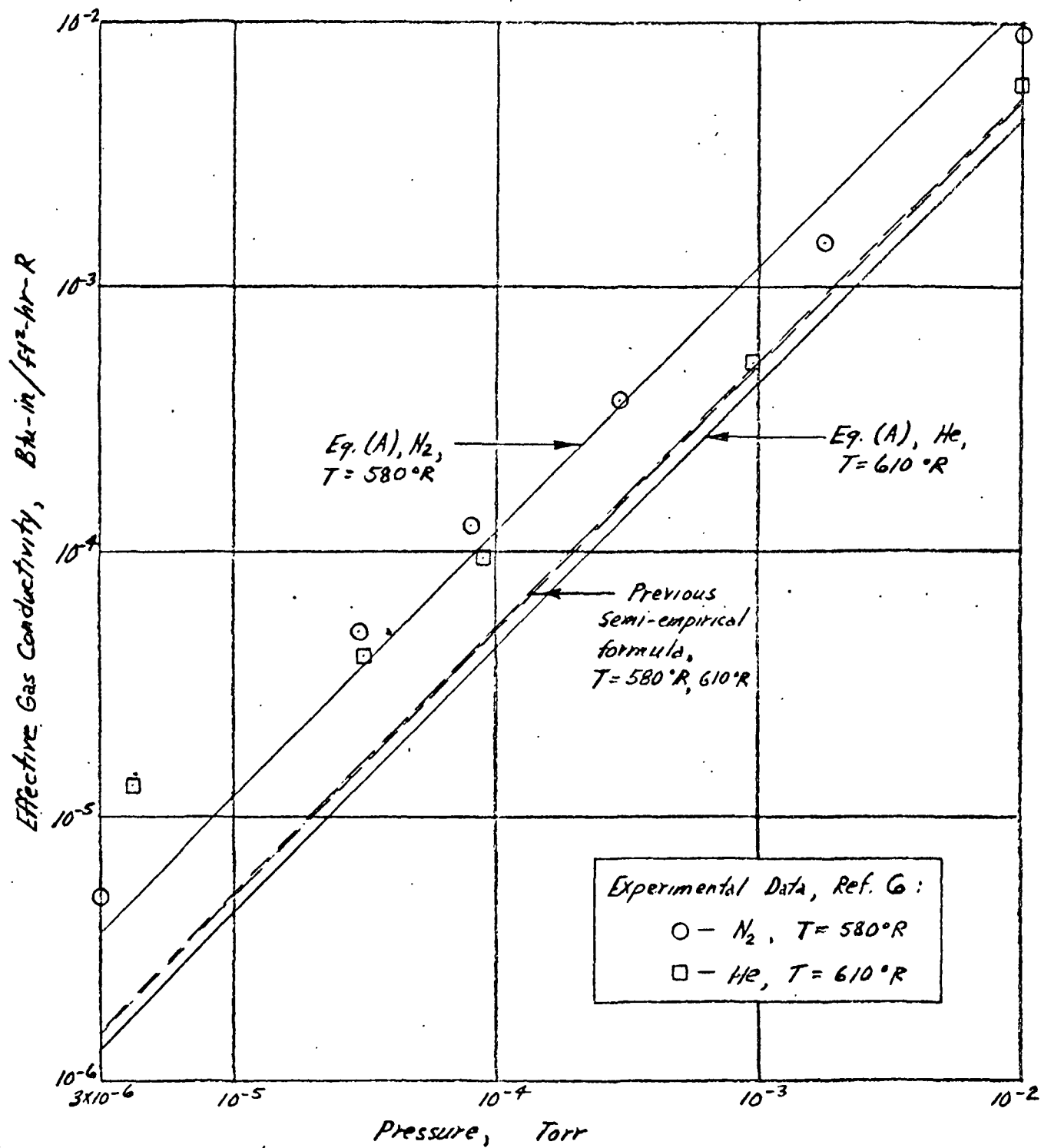


FIGURE B-1

EFFECTIVE GAS CONDUCTIVITY THEORY-EXPERIMENT COMPARISONS

## REFERENCES

1. Keller, C. W., "Thermal Performance of Multilayer Insulation," Sixth Quarterly Activity Report, 1 October - 31 December 1972, Contract NAS3-14377, LMSC/D337102, Lockheed Missiles and Space Company.
2. Vance, R. W., and Duke, W. M., Applied Cryogenic Engineering, John Wiley and Sons, N.Y., 1962.
3. Kennard, E. H., "Kinetic Theory of Gases," McGraw Hill, New York, 1938.
4. Ottestad, J. T., "Cryogenic Tank Insulation Properties Measurement," D2-113369-1, The Boeing Company, September 1971.

## APPENDIX C

### ANALYTICAL THERMAL MODELS

## APPENDIX C

### ANALYTICAL THERMAL MODELS

The analytical models used in the detailed thermal analysis are illustrated in Figures C-1 through C-7. Symbols employed in the illustrations are explained in Table C-1. The figures show the network of thermal conductors for each case treated, in order to provide an indication of the degree of detail to which the conduction heat paths were considered. Radiation heat paths, which in the analyses connected each pair of terminal nodes between which radiation interchange was significant, have been omitted from the figures for clarity. Although the analytical models, as currently defined, represent details of the full scale design, most of them can be easily adapted to analysis of the subscale design. The following paragraphs describe features of the individual models.

#### 1. Idealized MLI Blankets

The basic heat flow across the idealized MLI, i.e., blankets having no discontinuities or penetrations, was computed with the aid of the one-dimensional model illustrated in Figure C-1. The total heat path included conduction across the vacuum jacket skins, coupled radiation and conduction across the vacuum jacket core, radiation across the vacuum annulus, effective conduction across the MLI, and conduction across the pressure vessel wall. For the steady state analysis, representing the 30-day  $\text{LH}_2$  storage condition, constant boundary temperatures at T1 and T10 of 311.0K (560R) and 20.5K (37R), respectively, were assumed. For the transient analysis, representing response to an assumed orbiter reentry with an empty tank, the temperature at T1 was defined to follow a prescribed history while all other nodes were free to respond. The interface between the two MLI blankets was effectively ignored with the assumption of continuous properties through the entire MLI thickness. The MLI thickness was varied parametrically in the basic heat flow analysis in order to develop data required for thermal optimization.

## 2. Typical Lap Joint

Figure C-2 shows the model used to compute the incremental heat leak per unit length of single-step lap joint in the MLI. The two-dimensional model represented a 65.0 cm (26.6 in) wide section of blanket perpendicular to the joint. It was assumed that this section was wide enough to effectively isolate the influence of the joint on the basic heat flow. The gap at the butt interfaces in the joint, which had an important effect on the magnitude of the joint incremental heat leak, was assumed as .0127 cm (.050 in). As in the basic heat flow analysis, the MLI thickness was varied parametrically.

Unlike the basic heat flow model of Figure C-1, the joint analytical model did not include the vacuum jacket or the vacuum annulus. Instead, the temperatures at the outer surface of the MLI (T1, T8, T15, T22, ..., T85 in Figure C-2) were defined as fixed at the value derived from the corresponding point (T5, Figure C-1) from the basic heat flow analysis. The temperatures along the pressure vessel wall (T7, T14, T21, T28, ..., T91) were fixed at the cold boundary condition of 20.5K (37R).

## 3. Pin-and-Grommet Fastener

The model used for analyzing the pin-and-grommet (MLI panel installation) type fastener is shown in Figure C-3. This is the design for which alternatives having lower heat leaks are being considered. As in the case of the lap joint, the model includes an area of MLI assumed sufficient to isolate the thermal influence of the fastener. The analytical model is two-dimensional but, by virtue of assumed axial symmetry, represents the fastener and a surrounding circular area of MLI 52.07 cm (20.5 in) in diameter. In a manner similar to the previously described models, the MLI thickness and the affected lengths of pin shaft and grommet wall were varied parametrically. For the purpose of computing radiation interchange in the annular space between the pin and the grommet, the pin was assumed to be positioned concentric with the grommet.

The high temperature boundary condition, at T61, T60, T63, T21, T27, T33, T39, and T45, was set at the computed value for T5 from Figure C-1. The



cold (20.5°K) boundary condition was assumed to exist at T65, T64, T67, T26, T32, T38, T44, and T50.

#### 4. Pin Fasteners

Figure C-4 shows the model used for analyzing the pin-type (MLI panel assembly) fasteners. Because of the alternating positions of the fasteners in the two blankets and the possibility of thermal interactions between pairs of fasteners, it was necessary to employ a three-dimension analytical model. For the fastener pattern assumed, the minimum size recurring cell is the triangular area shown in the figure. The validity of such a recurring cell for an analytical model depends upon infinite repetition of the pattern. Its use here assumes no interactions between the pin-type fasteners and the pin-and-grommet fasteners, MLI lap joints, or other penetrations. For analysis of the pin-type fasteners, both the MLI thickness (Dimension "B" in the figure) and the pin spacing (dimension "A") were varied parametrically. Boundary temperatures were prescribed on the MLI surfaces and the surfaces of exposed pin heads in a manner similar to that used for the pin-and-grommet and lap joint analyses.

#### 5. Structural Support Strap

Figure C-5 illustrates the model used to analyze the incremental heat flow due to a typical support strap penetration. This penetration was actually treated in two parts: the strap itself, as a direct conductor of heat to the pressure vessel; and the MLI in the vicinity of the strap attachment on the pressure vessel wall. Both components of the model are shown in the figure.

The actual MLI blanket design at the strap penetration includes a number of cuts, seams, and joints. The modelling of these details, especially in view of the difficulties in estimating the dimensions of gaps and spaces at joints and the effects of compression of the MLI, appeared to be a formidable task and one of questionable value. It was observed that the design provided total MLI thicknesses at least equal to the basic thickness everywhere around the strap except in the vicinity of the attachment boss.

It was therefore assumed that the total heat leak due to the strap penetration could be estimated as the sum of the heat flow through the strap and the increased heat flow through the MLI due to the reduced thickness over the strap attachment bolt.

The strap was considered a one-dimensional conductor between the point where it emerged from the insulation cover to the thickened lug on its pressure vessel end. Those two points, designated at T51 and T20 on Figure C-5, were fixed at 311.0°K (560°R) and 20.5°K (37°R), respectively.

The MLI analysis employed a two-dimensional model of an axially symmetric area 52.07 cm (20.5 in) in diameter, centered on the strap attachment bolt. No interaction between the strap and the MLI was assumed. As in previous cases, the MLI thickness was varied parametrically.

Warm side boundary temperatures, at T1, T8, T15, T21, ..., T45 were again set at the value for MLI outer surface from the basic heat flow analysis. The cold side boundary temperature, at T20, T26, T32, T38, T44, and T50, were set at 20.5°K (37°R).

#### 6. Manhole Access and Feed Valve Assembly

The model for analysis of the manhole and feed valve penetration is shown in Figure C-6. The heat flow through this region was expected to be a complex combination of conduction and radiation with the thermal influence extending well beyond the periphery of the assembly.

With the exception of the two small vent and valve actuator tubes, the network shown in the figure was treated as axisymmetric. Heat flow through the two small tubes was computed on an independent, one-dimensional basis and added to the heat flow resulting from the axisymmetric model analysis. Radiation interchange between the small tubes and other surfaces was thus ignored, but all other possible radiant exchanges permitted by the nodal network, including radiation through the main feed line interior, were

considered. The conductor values for the two small coiled tubes were based on their true length, although the figure shows shorter lengths for clarity.

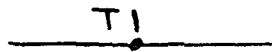
The model for manhole and feed valve penetration analysis included pressure vessel, MLI blanket, vacuum annulus, and vacuum jacket area out to a radius of 107 cm (42 in) from the feed line. The thickness of the main MLI blanket was taken as 2.79 cm (1.10 in) and the feed line MLI as 1.27 cm (0.50 in) rather than being varied as in some of the other component analyses. It was assumed that within the range of MLI thicknesses of interest for optimization purposes, the incremental heat leak associated with the manhole and feed valve penetration would be little affected by variations in thickness. The joints in the main MLI providing access to the manhole cover bolts were not included as part of this model but were included as part of the total length of joint in the computation of total joint heat leak.

Boundary temperatures for the model of Figure C-6 were 311.0°K (560°R) at T1, the outer, warm surface, and 20.5°K (37°R) at T2, at the cold boundary.

#### 7. Vent Valve Assembly

Figure C-7 shows the analytical model for the vent valve assembly penetration. Although the model was developed as illustrated, the analysis of this penetration, as described in the discussion of predicted heat flow results, did not incorporate all details of the model. Specifically, radiation within the valve enclosure and its vacuum jacket housing, and lateral heat flow in the pressure vessel MLI and the entire vacuum jacket were ignored. The analysis did include conduction through the main vent line, valve enclosure, insulation support collar, valve enclosure MLI (laterally only), and the valve enclosure vent and valve actuator tubes, as well as radiation through the main vent line interior. Boundary temperatures for the analysis were 20.5°K (37°R) at T70 and 311.0°K (560°R) at T1 and T69.

TABLE C-1  
SYMBOLS FOR ANALYTICAL THERMAL  
MODEL DIAGRAMS



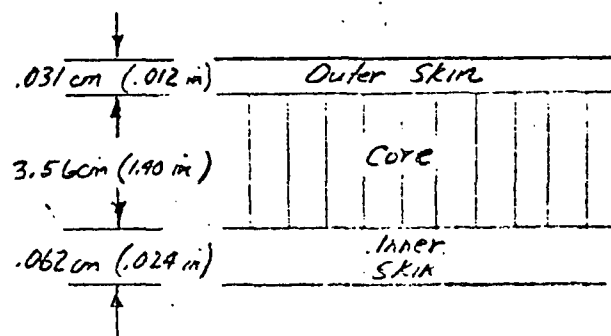
— Terminal Node: Terminus for thermal conductors, point at which temperatures are evaluated, point at which surface area (for radiation interchange) and thermal capacitance (for transient thermal diffusion) are assumed concentrated.



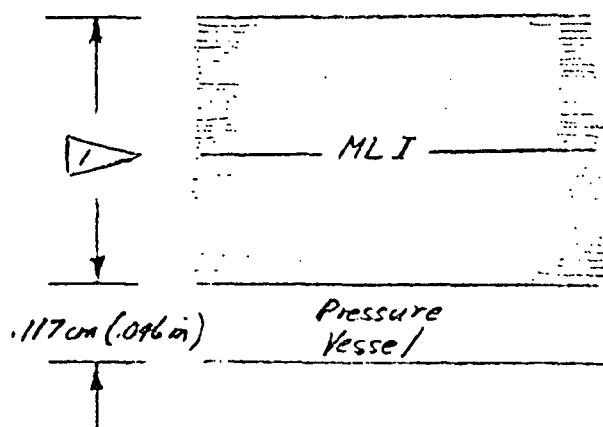
— Thermal Conductor: Connects two terminal nodes, incorporating properties of material conductivity, conduction path length and conduction path effective cross section area.



— Zero Resistance Conductor: Connects terminal nodes whose temperatures are assumed identical.



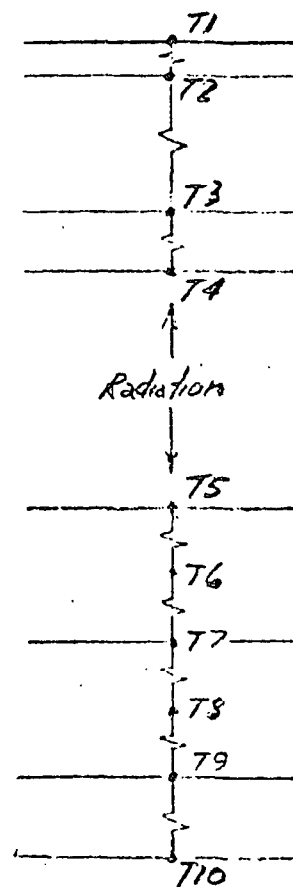
Vacuum  
Annulus



Section

△ MLI thickness varied parametrically

FIGURE C-1



Model

(NO TO SCALE)

ANALYTICAL THERMAL MODEL - AREAS REMOTE FROM PENETRATIONS OR DISCONTINUITIES

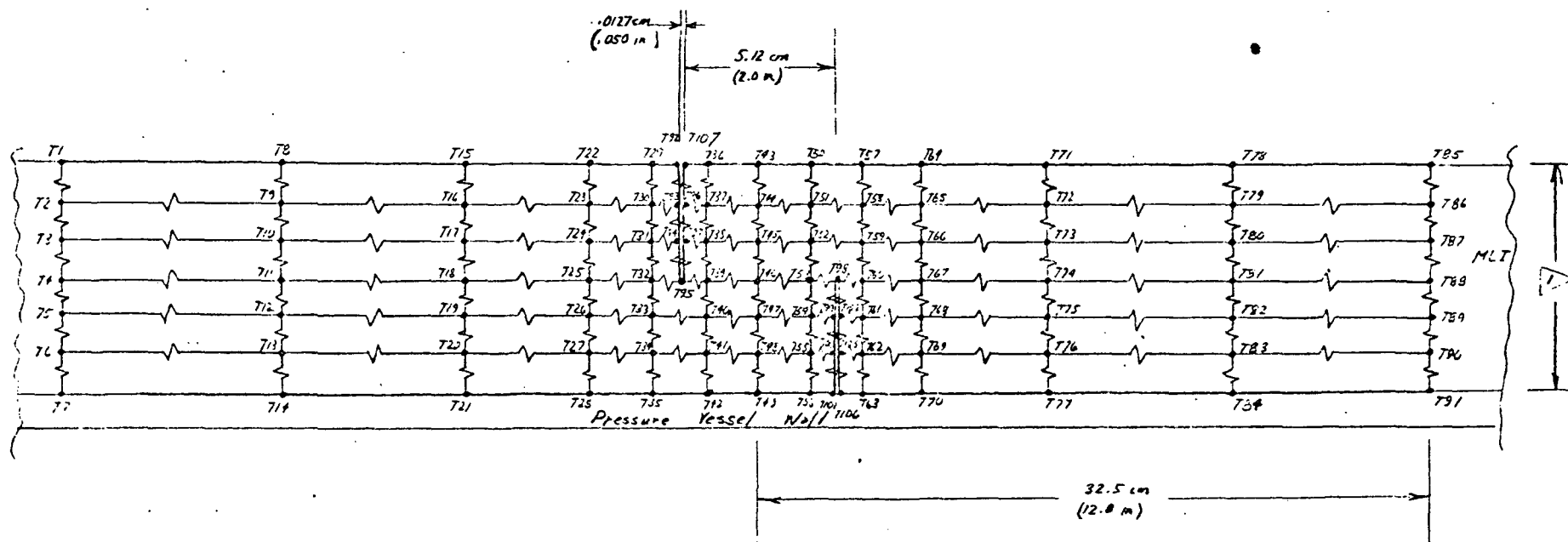
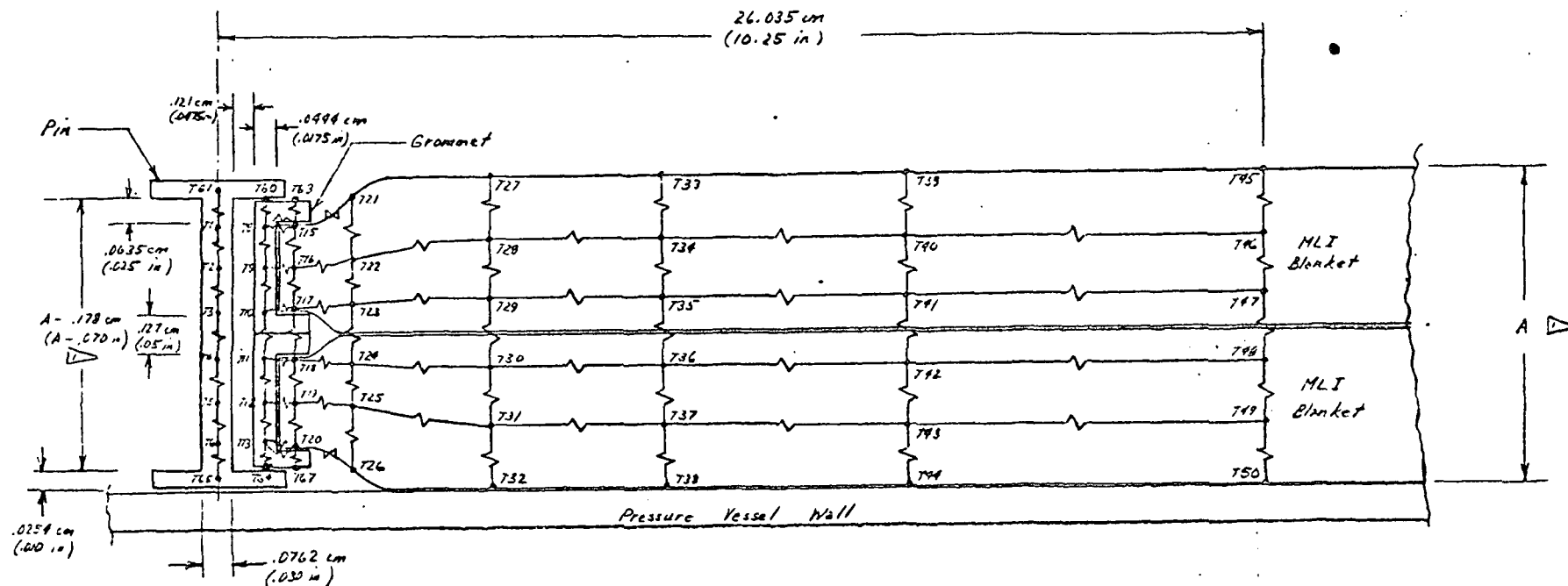


FIGURE C-2

ANALYTICAL THERMAL MODEL - TYPICAL SECTION THROUGH SINGLE-STEP LAP JOINT  
(NOT TO SCALE)

SHEET



△ MLI Thickness varied parametrically

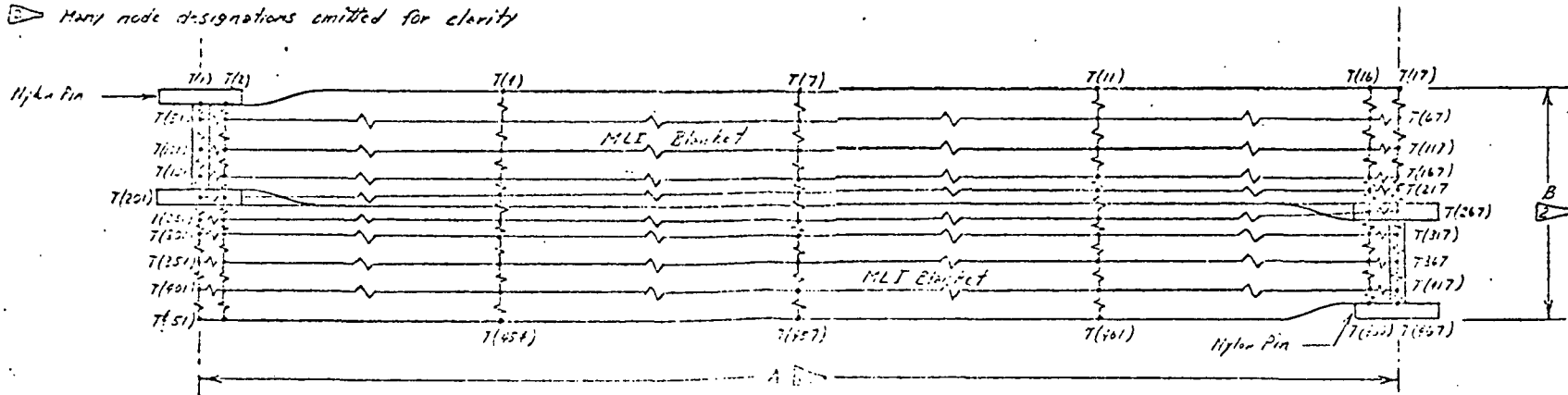
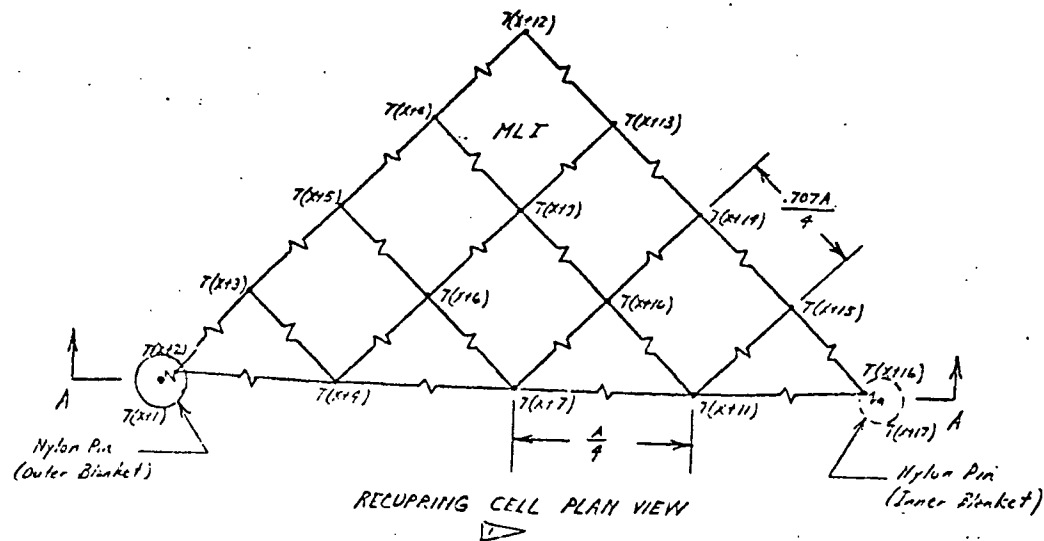
FIGURE C-3  
ANALYTICAL THERMAL MODEL - TYPICAL NYLON PIN-AND-GROMMET FASTENER & SURROUNDING MLI  
(NOT TO SCALE)

TYPICAL FASTENER PATTERN

▷  $X = 0, 50, 100, \dots, 450$

⇒ MLT thickness and fastener pitch varied parametrically

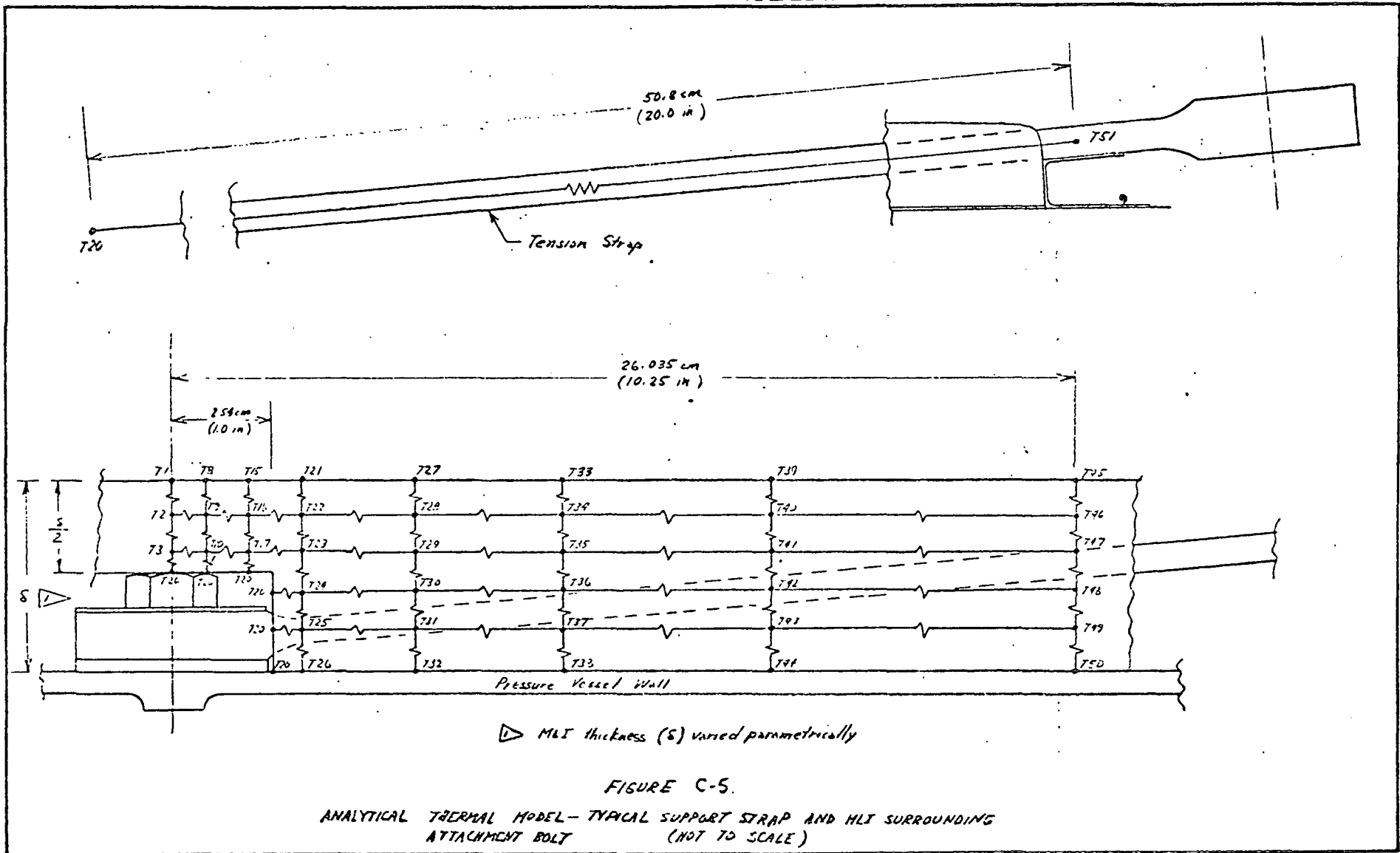
Many node designations omitted for clarity



A-A نى ئۆزگەرتىش

ANALYTICAL THERMAL MODEL - PIN-TYPE FASTENERS &amp; ASSOCIATED MLI (Not to Scale)





SHEET

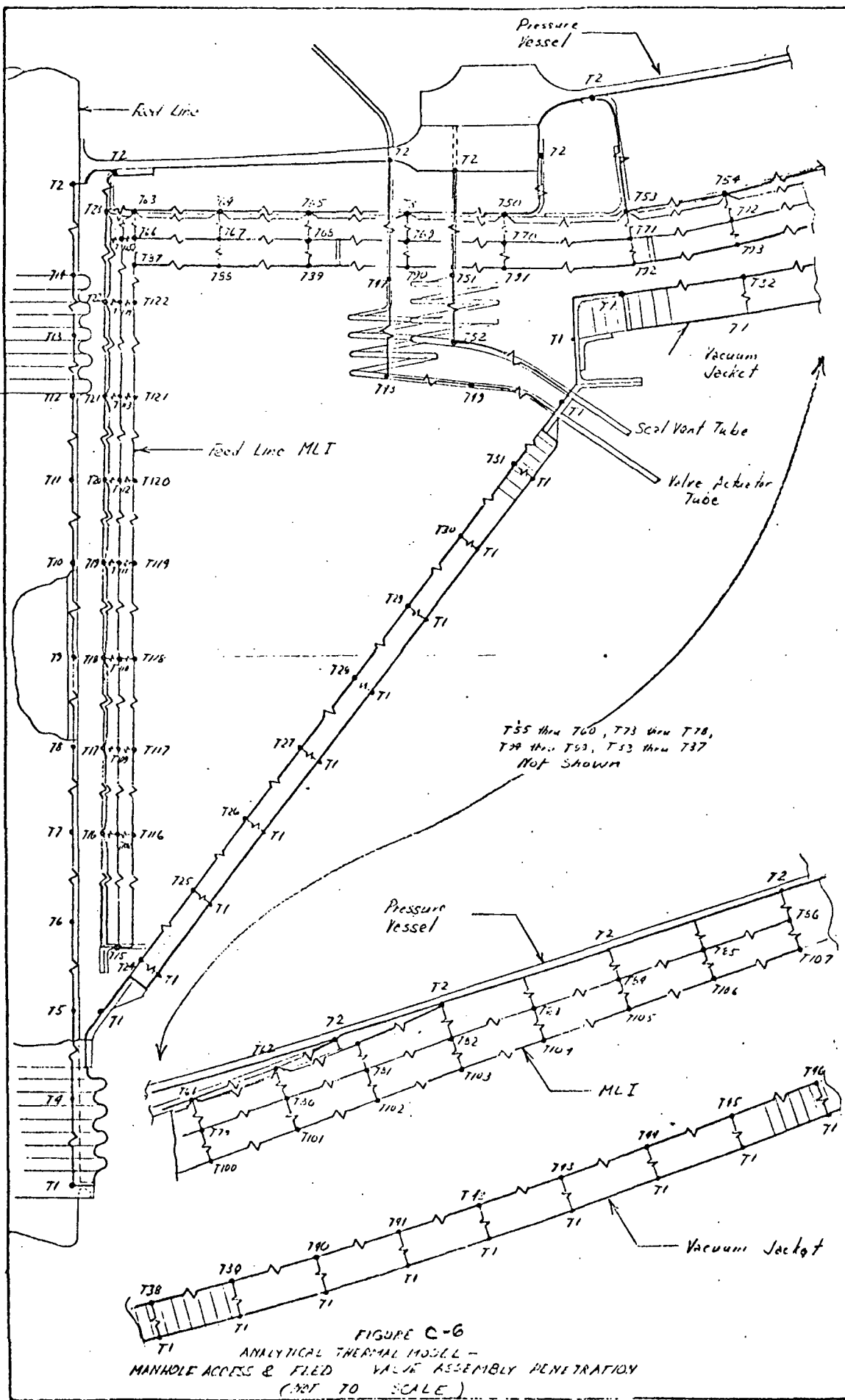


FIGURE C-6  
ANALYTICAL THERMAL MODEL -  
MANHOLE ACCESS & FIELD VALVE ASSEMBLY PENETRATION  
(NOT TO SCALE)

BOEING COMPANY

NUMBER  
REV LTR

FIGURE C-7  
ANALYTICAL THERMAL MODEL --  
VENT VALVE ASSEMBLY PENETRATION  
(Not to Scale)

Valve Enclosure Vent Tube

Vacuum Jacket

Valve Actuator Tube

MLI

Insulation Support Collar

Valve Enclosure

Vacuum Jacket

MLI

T40 thru T46  
T49 thru T55

Pressure Vessel

Vent Line

Vent Valve

## APPENDIX D

### HALF SCALE LH<sub>2</sub> TEST MODEL DRAWINGS

ASSEMBLY AND QUALIFICATION PROCEDURES

1. APPLY KEL F 50 (M) GREASE BETWEEN THE MATING SURFACES OF THE -G ATTACHMENT FITTING ASSEMBLIES AND THE EMITE 5103 SUPPORT FIXTURE PULOW BLOCKS. CLAMP -G ASSEMBLIES IN PLACE.

2. CENTER THE LH<sub>2</sub> PRESSURE VESSEL ASSEMBLY SK2-5085-139-1 IN THE SUPPORT FIXTURE AND SECURE IT IN POSITION WITH TEMPORARY SUPPORTS. ATTACH THE SUPPORT STRAP ASSEMBLIES SK2-5085-151-1, TURNBUCKLE ASSEMBLIES SK2-5085-163-1 SPRING PINS MS 16562-249, BOLTS BAC30LT G411, AND NUTS BACN10JC GCM. ALIGN PRESSURE VESSEL PER 20 AND TORQUE TURNBUCKLES PER 21.

3. REMOVE TEMPORARY SUPPORTS. SLOWLY ROTATE THE PRESSURE VESSEL IN THE SUPPORT FIXTURE. DURING ROTATION DETERMINE EXTENT OF PRESSURE VESSEL AND SUPPORT STRAP MOVEMENT.

4. SAW OFF SK2-5085-139-5 TUBE PER SK2-5085-137 SHEET G. WELD INLET TUBE ASSEMBLY SK2-5085-163-4 IN PLACE PER 7.

5. HELIUM LEAK CHECK PRESSURE VESSEL ASSEMBLY.

6. CLEAN ALL -1 ASSEMBLY SURFACES EXPOSED TO THE VACUUM ANNULUS PER 5 AND ENGINEERING INSTRUCTIONS. WHITE NYLON GLOVES AND COATS (OR EQUIVALENT) SHALL BE WORN BY ALL ASSEMBLY PERSONNEL TO INSURE CLEANLINESS OF ALL SURFACES EXPOSED TO THE VACUUM ANNULUS.

7. INSTALL MLI PANELS PER SK2-5085-141 MLI INSTALLATION.

8. INSTALL VACUUM JACKET HEADS SK2-5085-140-1, INNER JOINT PLATE -3, OUTER JOINT PLATE -4, AND SPICE PLATE -5.

9. WELD SEALING SKIN -Z VACUUM JACKET CLOSEOUT ASSEMBLIES SK2-5085-154-1 AND SK2-5085-156-1 IN PLACE. HELIUM LEAK CHECK WELDS.

10. CYCLOPS DIVISION OF CYCLOPS ENGINEERING CORP. PART NUMBER.

11. ULTEX DIVISION OF PERKIN-ELMER PART NUMBER.

12. TORQUE 19 TURNBUCKLES ATTACHING SUPPORT STRAPS AS FOLLOWS:  
A) AFTER PRESSURE VESSEL ALIGNMENT 20, SET TURNBUCKLES ON LOWER STRAP TO GO: 8 IN-LB TORQUE.  
B) BEFORE SETTING TURNBUCKLES ON UPPER STRAP, FIRST ROTATE PRESSURE VESSEL 180° IN SUPPORT FIXTURE THEN SET TURNBUCKLES TO GO: 8 IN-LB TORQUE.

13. ALIGN PRESSURE VESSEL SK2-5085-139-1 IN SUPPORT FIXTURE EMITE 5103 AS FOLLOWS:  
A) TORQUE 19 TURNBUCKLES ATTACHING SUPPORT STRAPS - FINGER TIGHT.  
B) ALIGN PRESSURE VESSEL BY INCREASING AND DECREASING TORQUE 19 OF TURNBUCKLES.  
DO NOT EXCEED 100 IN-LB TORQUE.

14. DO NOT TWIST SUPPORT STRAPS SK2-5085-151-1.

15. 30Z OR 30A CWS WIRE PER QQ-W-423 OR EQUIVALENT.

16. TAPE TO OUTER SURFACE OF OUTER MLI PANELS PER 13.

17. TAPE TO OUTER SURFACE OF INNER MLI PANELS PER 13.

18. TAPE TO SK2-5085-139-1 PER 13.

19. EE-6600 (PERMACEL) REFLECTIVE, POLYESTER, INTERLUMED, PRESSURE SENSITIVE TAPE.

20. TAPE THERMOCOUPLES IN PLACE WITH 14, 1.00 ± .15 INCHES.

21. WELD PER BAC 5930 CLASS C. INSPECT PER 11.

22. LEAK RATE SHALL NOT EXCEED 1X10<sup>-9</sup> STD CC/HELIUM PER SECOND.

23. RADIOGRAPHIC INSPECT PER BAC 5915. PENETRANT INSPECTION SHALL NOT BE USED.

24. COLD SHOCK WELD WITH LN<sub>2</sub> PRIOR TO HELIUM LEAK CHECK.

25. STANDARD PRESSED STEEL PART NUMBER.

26. WELD PER BAC 5935 CLASS A. FILLER WIRE EX 4043 PER QQ-R-566 TYPE 4043 OR MIL-E-16053 TYPE 4043. COLD SHOCK PER 25. INSPECT PER 10 & 11.

27. BAKE AT 480°K (350°F) IN A VACUUM FOR 2 HOURS. STORE IN SEALED PACKAGE AFTER BAKING TO PRESERVE CLEANLINESS.

28. ASSEMBLY OF THE TEST MODEL SHALL BE DONE IN A DUST FREE ROOM. ALL ASSEMBLY TOOLS SHALL BE FREE FROM OIL AND PARTICLE CONTAMINATION. METAL SURFACES EXPOSED TO THE VACUUM ANNULUS SHALL BE CLEANED PER BAC 5765. AND THEN PROTECTED FROM OIL AND PARTICLE CONTAMINATION.

29. HEAT TREAT TO T6 PER BAC 5602.

30. WELD PER BAC 5935 CLASS C. FILLER WIRE EX 4043 PER QQ-R-566 TYPE 4043 OR MIL-E-16053 TYPE 4043. INSPECT PER 10 & 11.

31. 6061 T4 ALUM BAR PER QQ-A-225/8.

32. 6061 T6 ALUM. SHEET PER QQ-A-250/11.

33. 32 BACN10JC GCM NUT

34. 32 BAC30LT G411 BOLT 3/8 DIA

35. 24 AN360 D44L WASHER

36. 208 AN360 D616L WASHER

37. 6 BAC30LT G420 BOLT 1/4 DIA

38. 8 BAC30LT G415 BOLT 1/4 DIA

39. 16 BAC30LT G420 BOLT 3/8 DIA

40. 32 BAC30LT G417 BOLT 3/8 DIA

41. 160 BAC30LT G415 BOLT 3/8 DIA

42. 16 MS16562-249 DIN - SPRING 7/32 DIA

43. 24 13705 E-048 ANCHOR NUT - FLOATING 2500-28 UNIF. 30 THRD

44. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

45. 32 BAC30LT G415 BOLT 3/8 DIA

46. 16 MS16562-249 DIN - SPRING 7/32 DIA

47. 24 13705 E-048 ANCHOR NUT - FLOATING 2500-28 UNIF. 30 THRD

48. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

49. 32 BAC30LT G415 BOLT 3/8 DIA

50. 16 MS16562-249 DIN - SPRING 7/32 DIA

51. 24 13705 E-048 ANCHOR NUT - FLOATING 2500-28 UNIF. 30 THRD

52. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

53. 32 BAC30LT G415 BOLT 3/8 DIA

54. 16 MS16562-249 DIN - SPRING 7/32 DIA

55. 24 13705 E-048 ANCHOR NUT - FLOATING 2500-28 UNIF. 30 THRD

56. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

57. 32 BAC30LT G415 BOLT 3/8 DIA

58. 16 MS16562-249 DIN - SPRING 7/32 DIA

59. 24 13705 E-048 ANCHOR NUT - FLOATING 2500-28 UNIF. 30 THRD

60. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

61. 32 BAC30LT G415 BOLT 3/8 DIA

62. 16 MS16562-249 DIN - SPRING 7/32 DIA

63. 24 13705 E-048 ANCHOR NUT - FLOATING 2500-28 UNIF. 30 THRD

64. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

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66. 16 MS16562-249 DIN - SPRING 7/32 DIA

67. 24 13705 E-048 ANCHOR NUT - FLOATING 2500-28 UNIF. 30 THRD

68. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

69. 32 BAC30LT G415 BOLT 3/8 DIA

70. 16 MS16562-249 DIN - SPRING 7/32 DIA

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72. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

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74. 16 MS16562-249 DIN - SPRING 7/32 DIA

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312. 208 13705 E-064 ANCHOR NUT - FLOATING 3150-24 UNIF. 38 THRD

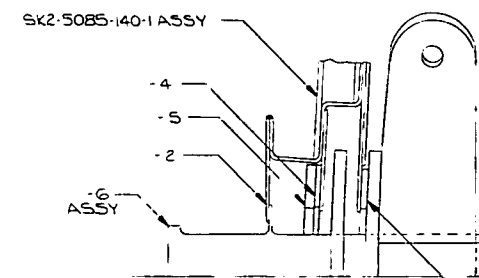
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314. 16 MS16562-249 DIN - SPRING 7/32 DIA



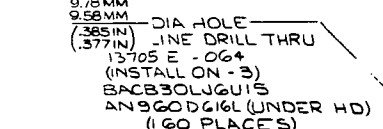
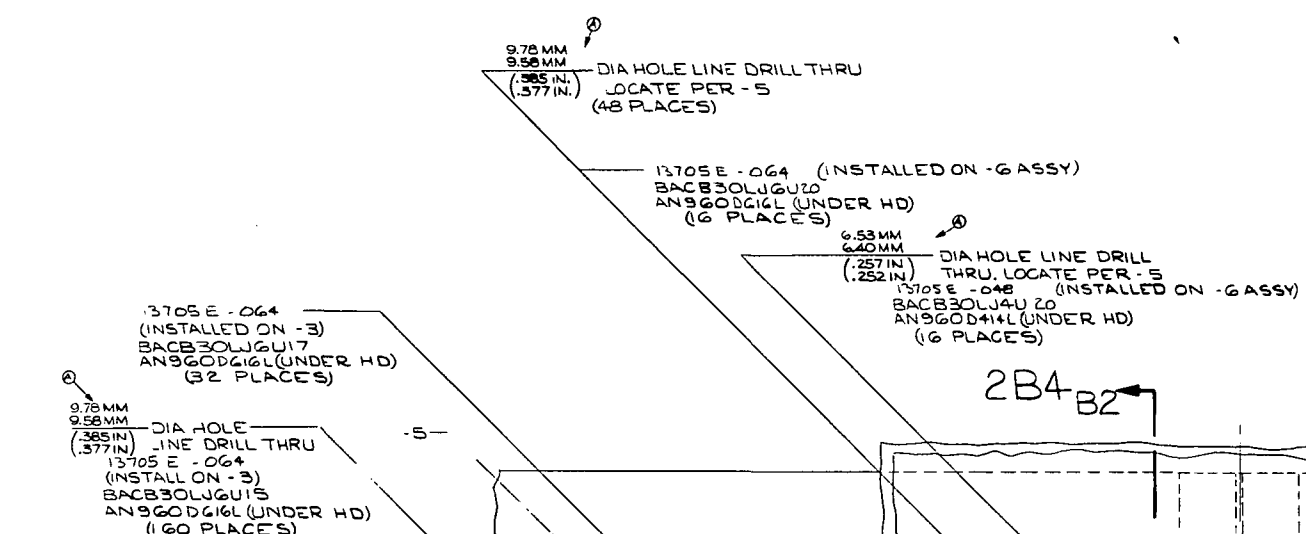
252082-1313

REVISIONS			
REV	DESCRIPTION	DATE	APPROVED
A	252 DIA HOLES WERE	250	
B	385 DIA HOLES WERE	379	1/18/74
C	377	375	
D	ADDED .25 DIA VENT HOLES	2/11/74	



SECTION B4

0 10 20 30 40 50  
MM



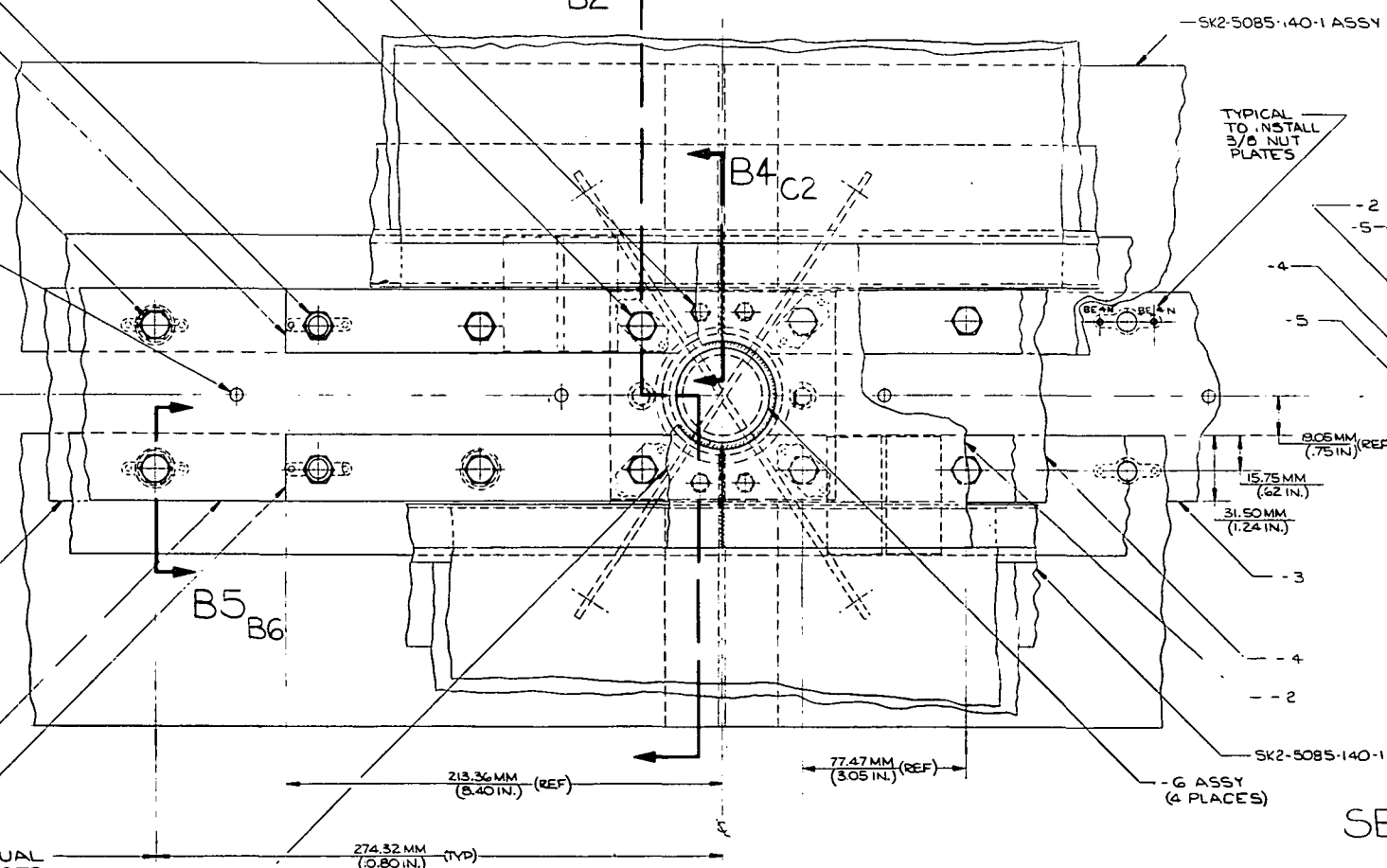
SECTION B5

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SK2-5085-140-1 ASSY (4 PLACES)

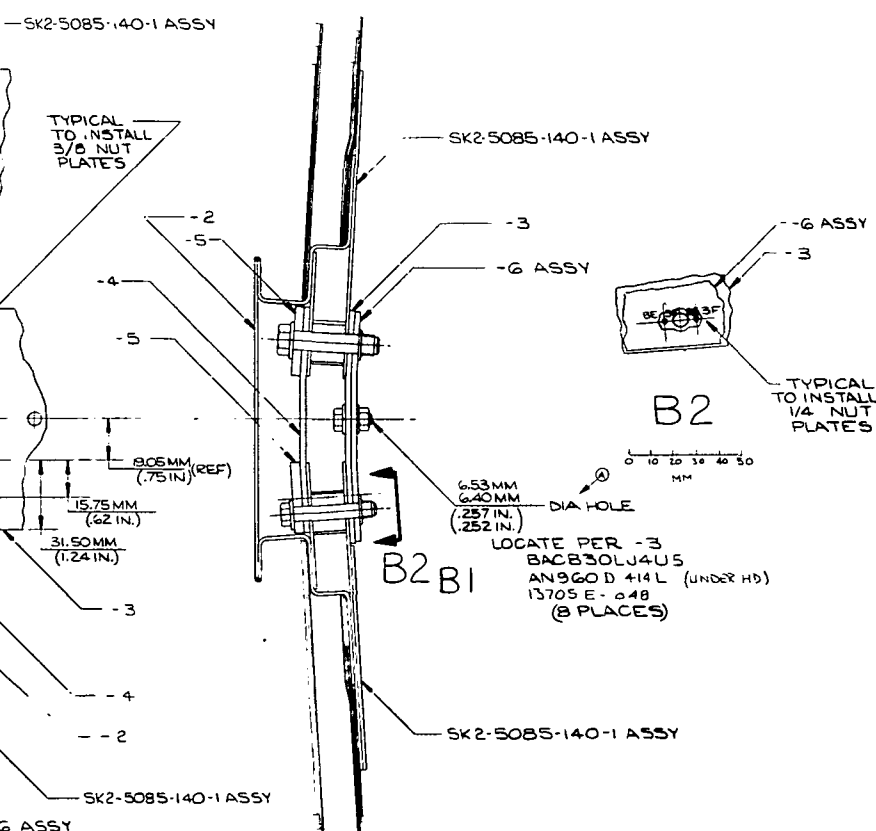
(4 PLACES)  
(8 PLACES)

19 EQUAL SPACES (TYP)



DETAIL I  
(ROTATED 90°)

0 10 20 30 40 50  
MM



SECTION 2B4

0 10 20 30 40 50  
MM

DRAWING DATA		CONTRACT NUMBER	
DRAWING NUMBER		NAS3-15848	
DATE		DATE	
REV		REV	
MATERIAL		MATERIAL	
QUANTITY		QUANTITY	
UNIT		UNIT	
PROJECT		PROJECT	
DRAWN BY		DRAWN BY	
CHECKED BY		CHECKED BY	
APPROVED BY		APPROVED BY	
DATE		DATE	
SCALE		SCALE	
SHEET NUMBER		SHEET NUMBER	
TOTAL SHEETS		TOTAL SHEETS	

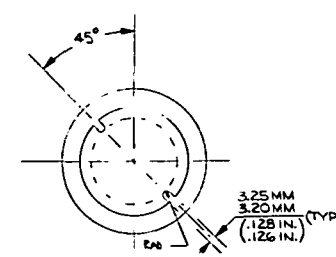
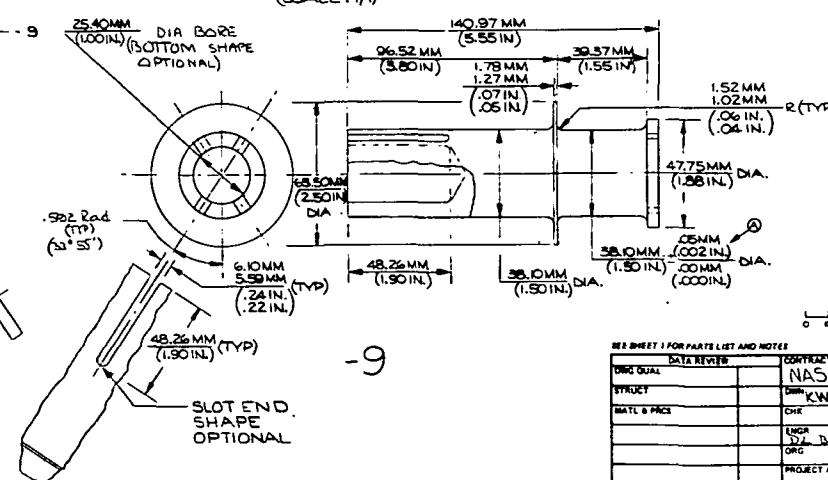
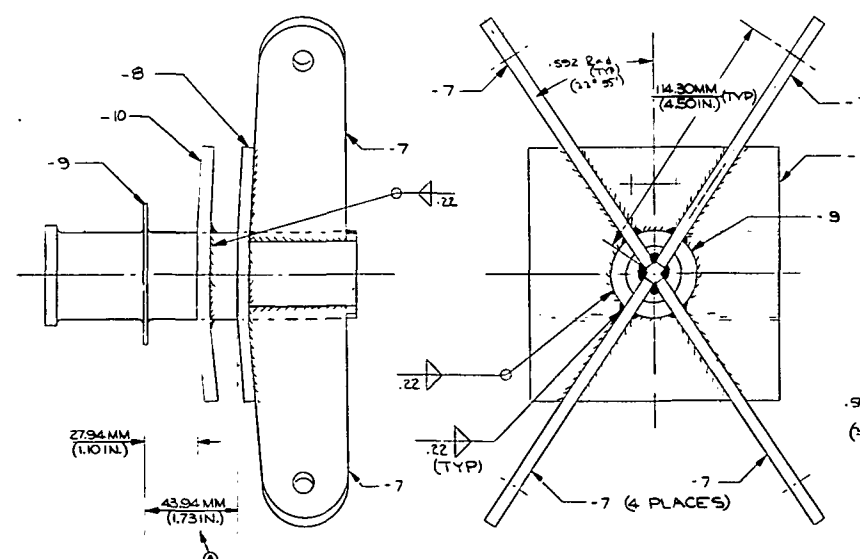
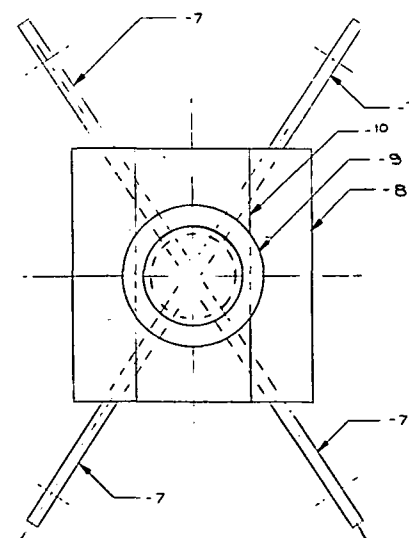
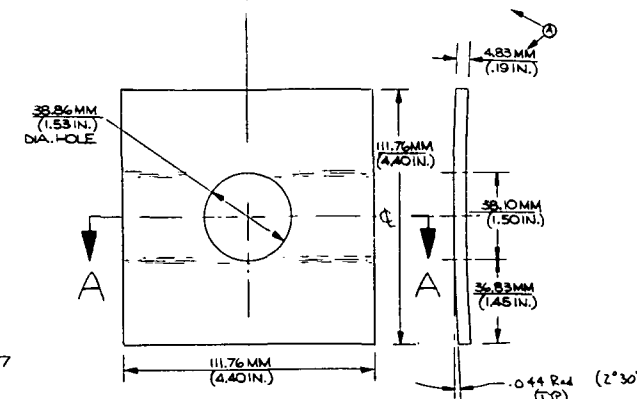
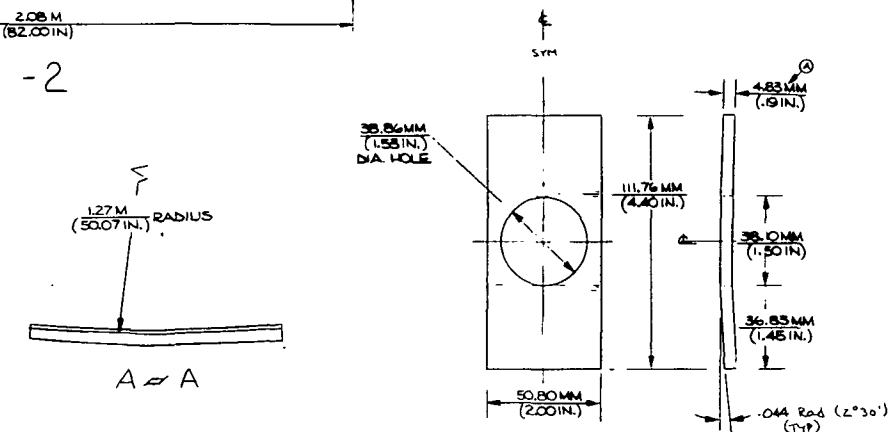
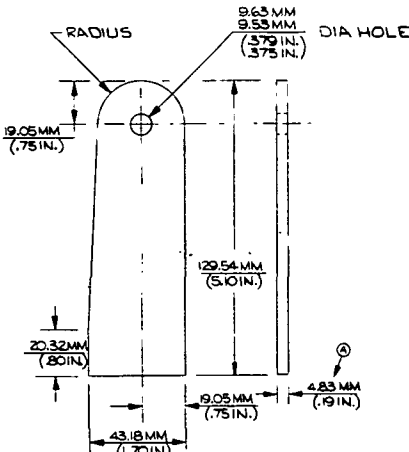
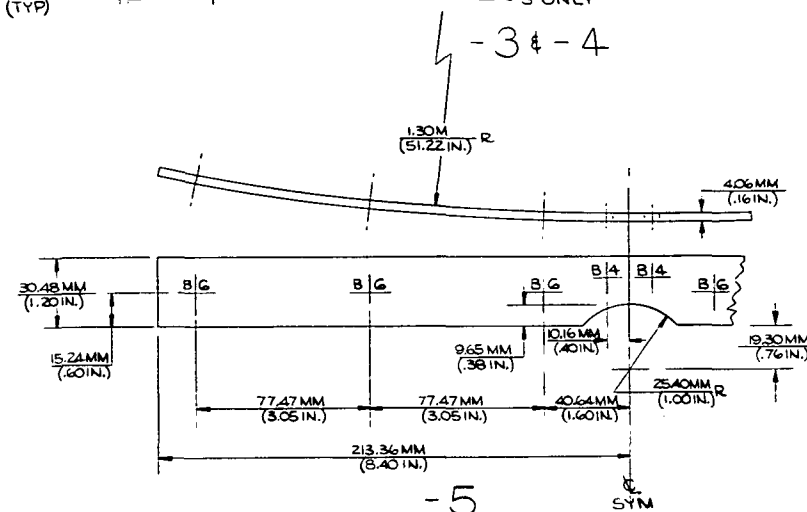
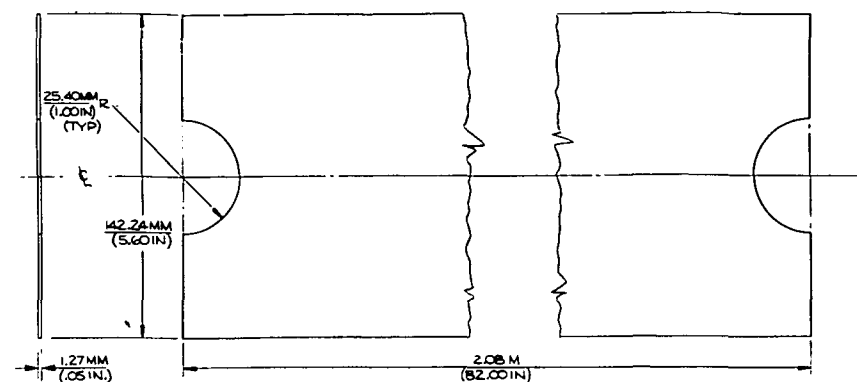
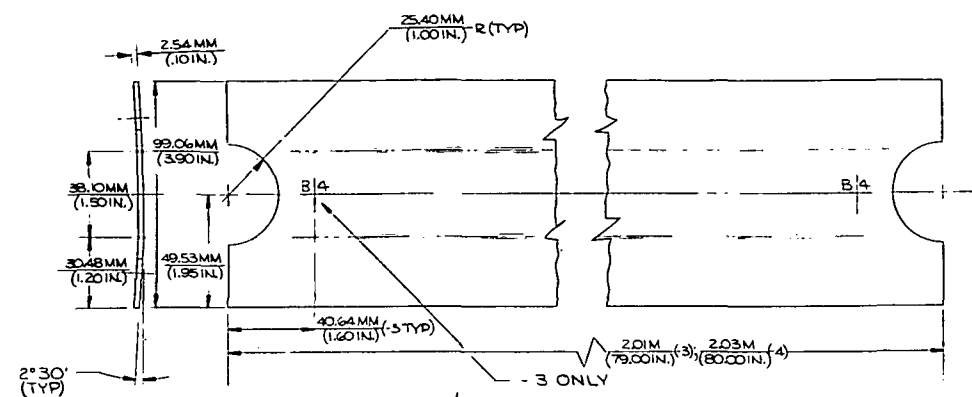
Figure D-3

THE BOEING COMPANY	
CORPORATE OFFICES	
SEATTLE, WASHINGTON 98106	
HALF SCALE	
LH2 TEST MODEL	
ASSEMBLY	
J812.05	
SK2-5085-137	
SCALE 1/1	
SHEET 3 OF 4	

SK2-5085-131B3

252082-1313

REVIEWS			
ZONE	LTA	DESCRIPTION	DATE APPROV
	A	ADDED A-A ALTRED - 20 THICKNESS FOR THRS - 7 - 8 - 4-10 1725 DPA FOR - 6 1-500' <sup>ONE</sup> DIA FOR - 9	11/11/11 <i>LEAF</i>

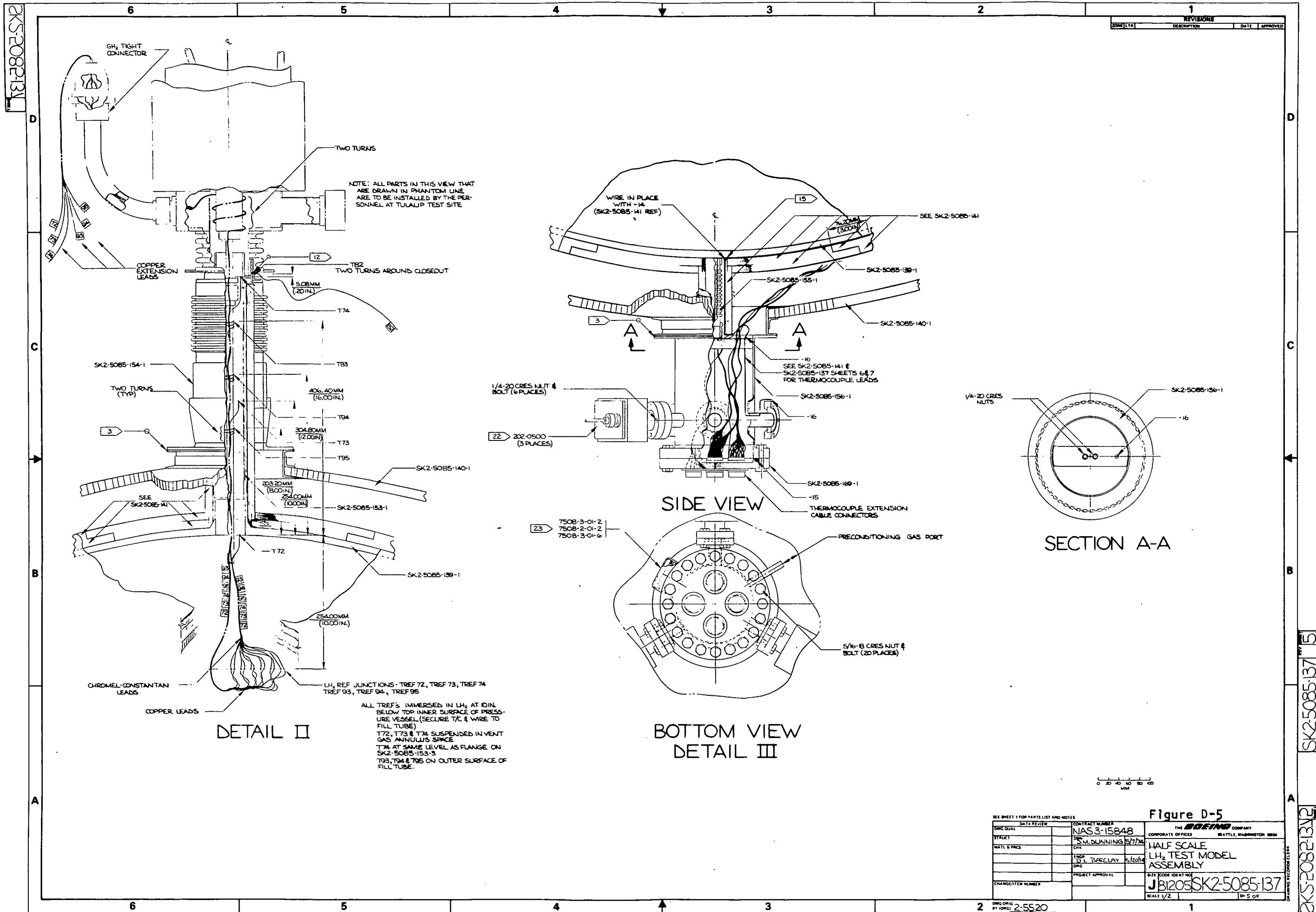


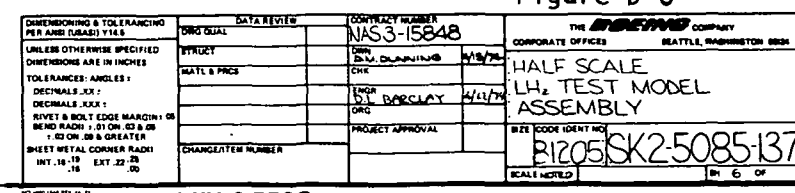
-6 ASSEMBLY

Figure D-4

NLS SHEET 1 FOR PARTS LIST AND NOTES		Figure D-4	
DATA REVIEW		CONTRACT NUMBER	
DRG QUAL		NAS 3-15848	
STRUCT		DRG	KW050809-4-73
SCALE & PRICE		CHG	
		ENGR	D. DARLEY 9-0-73
		DRG	
CHANGED NUMBER		PROJECT APPROVAL	
		SITE CODE IDENT NO	J81205
		SCALE (V)	SK2-5085-137
		IN 4 OF 6	







DWG ORIG 2-5520  
BY (ORG)



A

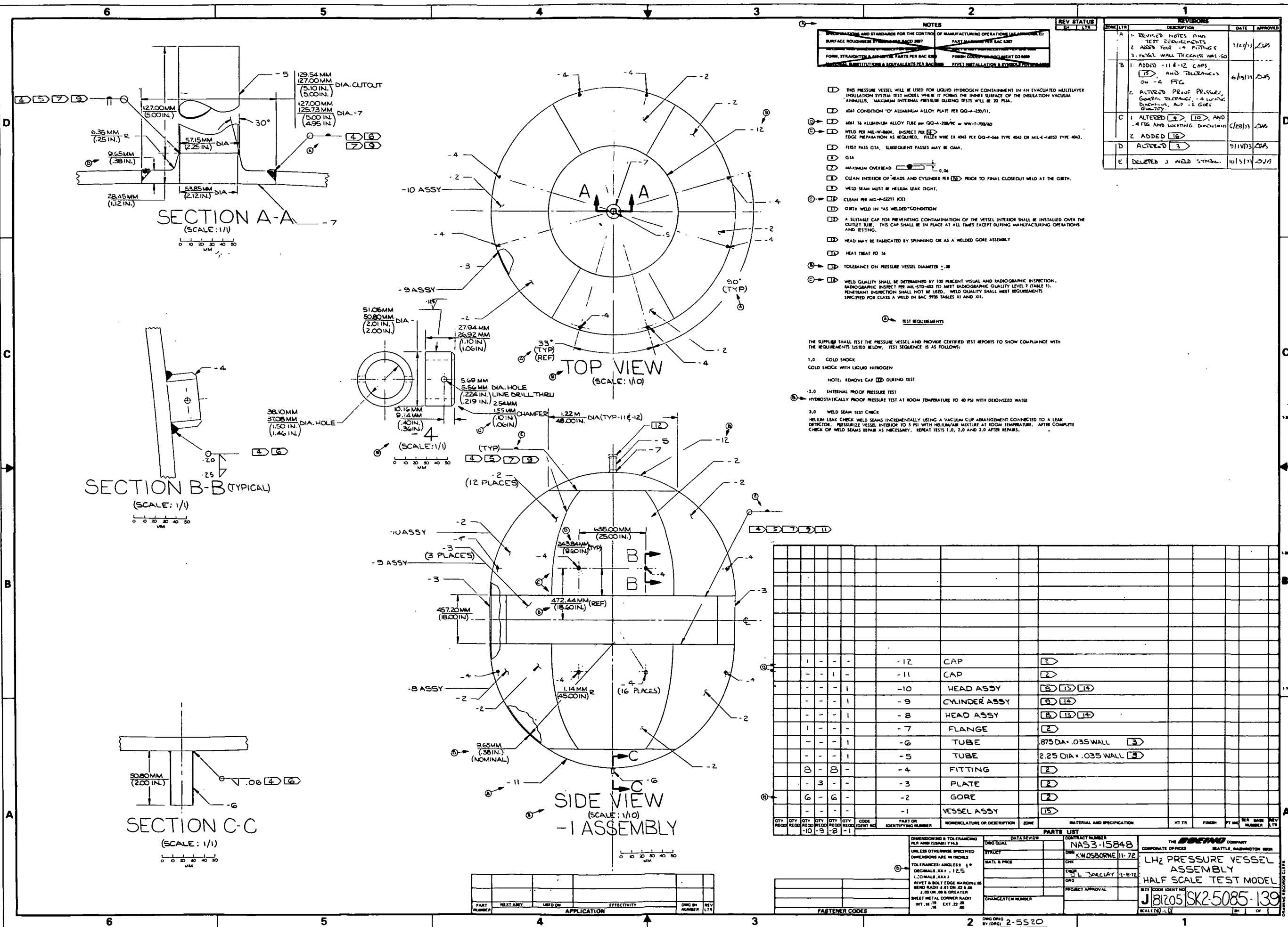
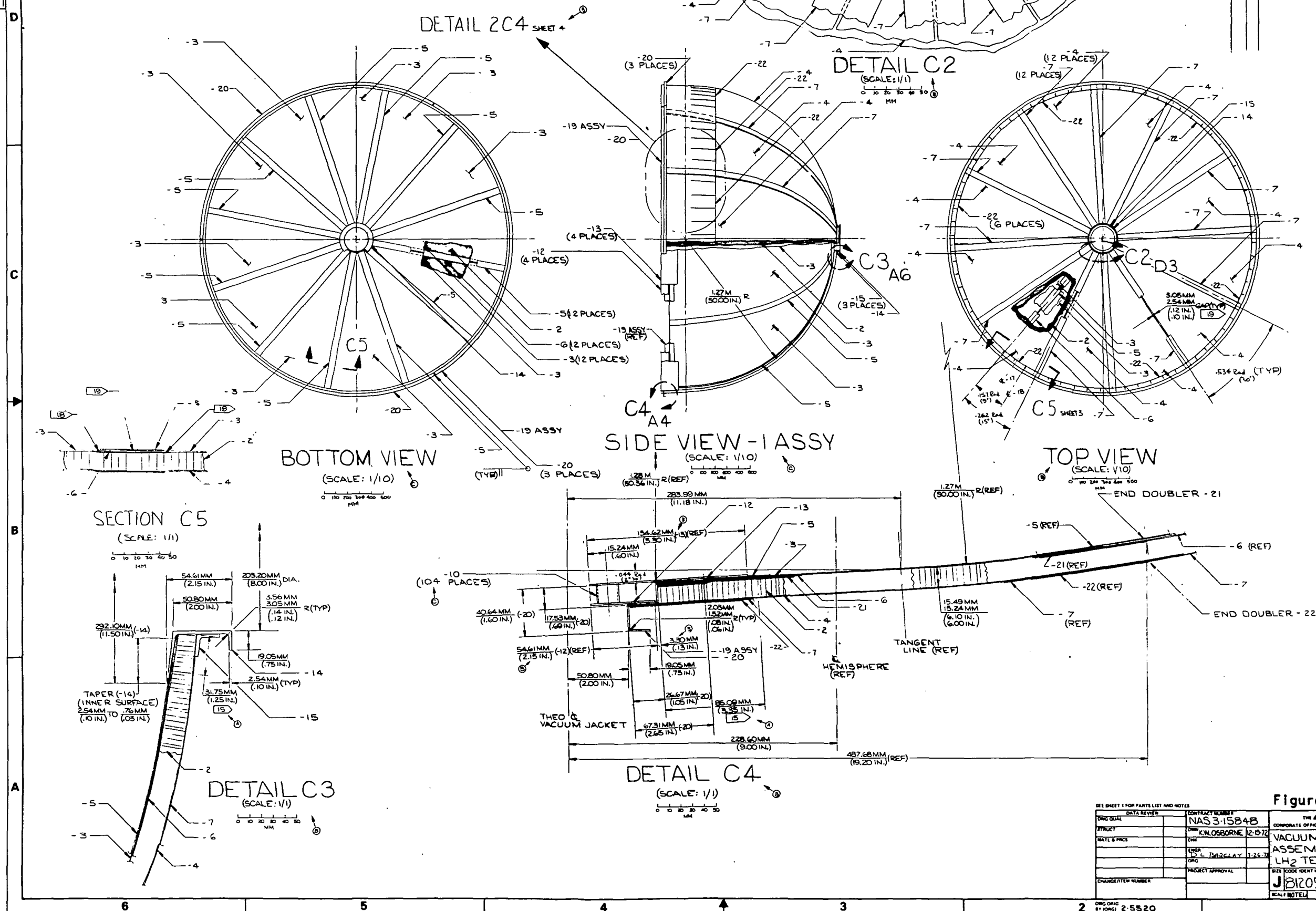


Figure D-8

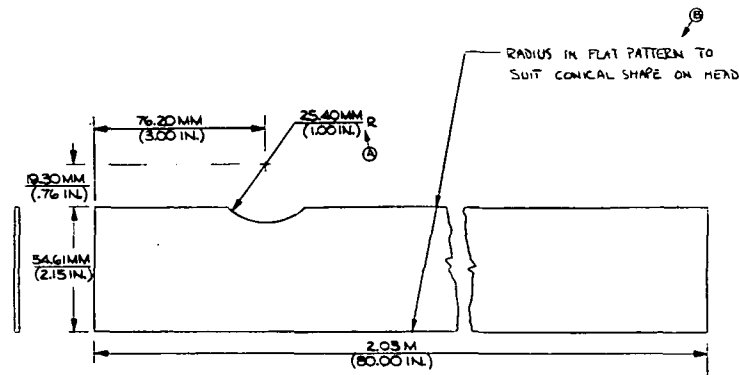


INVESTIGATIONS				
ZONE	LT#	DESCRIPTION	DATE	APPROV
	A	DELETED - 9 & 15 ASSD 15	4/16/73	SWG
	B	ADDED 4 LOCATING - 17 & 18 SECTION 2 C 5, DETAIL C 5 (MAY) DELETED 204, 204 & 12 DELETED STEP EFF - 16	7/6/73	SWG
	C	DELETED - 8 & 11 FILLERS - 10 SPACER UNIT 20 PLACES	7/24/73	SWG

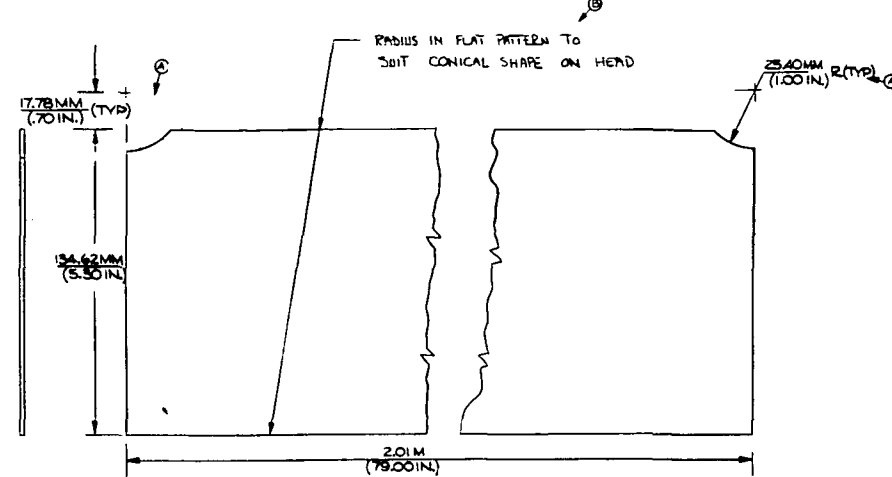


**Figure D-10**

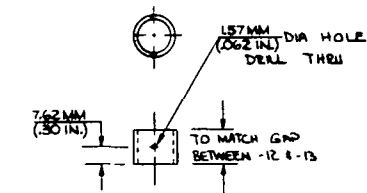
USE SHEET 1 FOR PARTS LIST AND NOTES		Figure D-10	
DATE REVISION		CONTRACT NUMBER	
CNO: DALL		NAS 3:15848	
ATTACH:		CNO: K.W.OSBORNE 12-5-72	
MATERIAL PRICE		CNO:	
		CNO: D.L.DORCLAY 1-25-73	
		PROJECT APPROVAL	
CHANGE/ITEM NUMBER		SIZE CODE IDENT NO	
		J B1205 SK25085-14	
		SCALE: (NOTES)	
		BY 2 of	



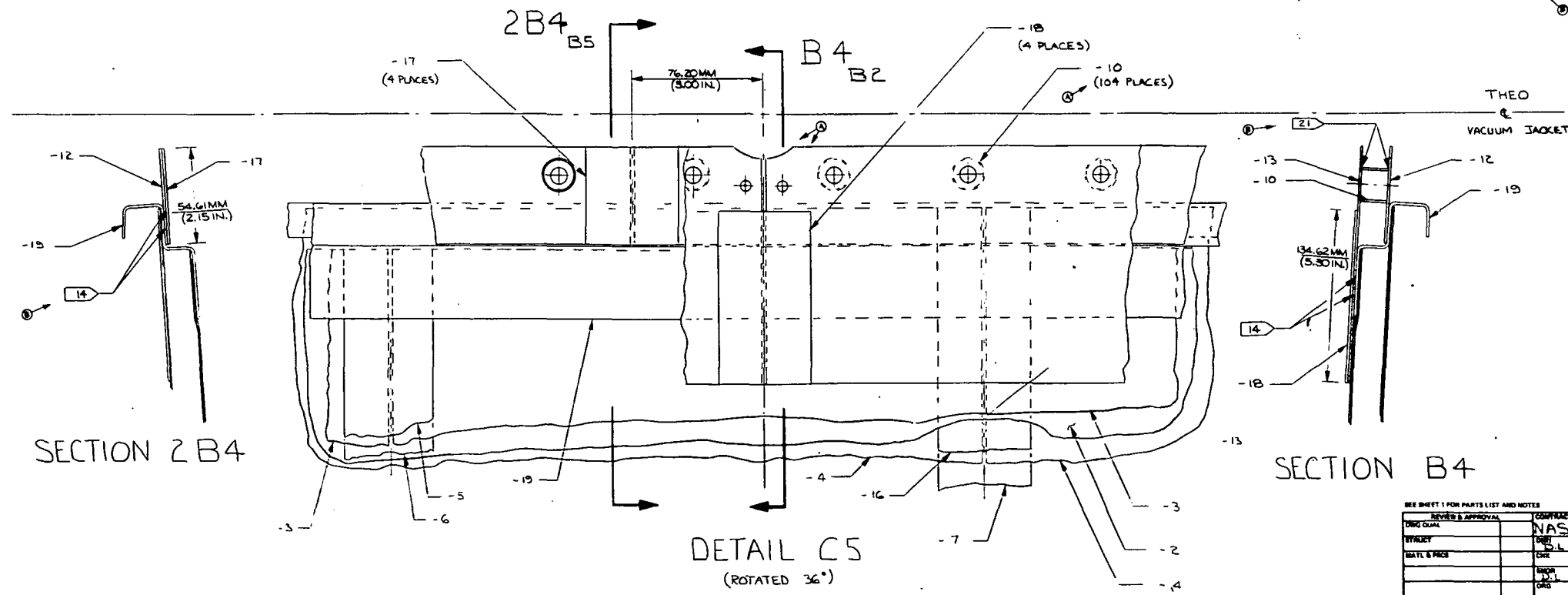
-12



-13



-10



SECTION 2 B4

SECTION B4

DETAIL C5  
(ROTATED 36°)

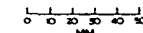
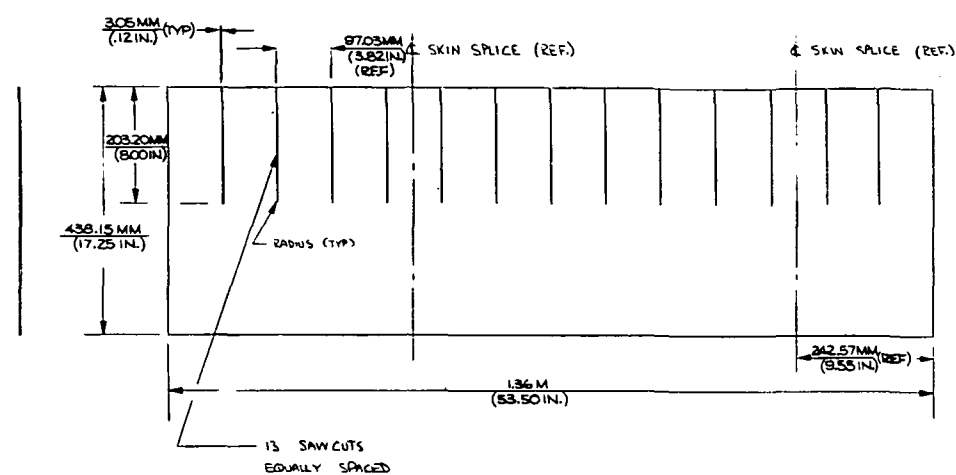


Figure D-11

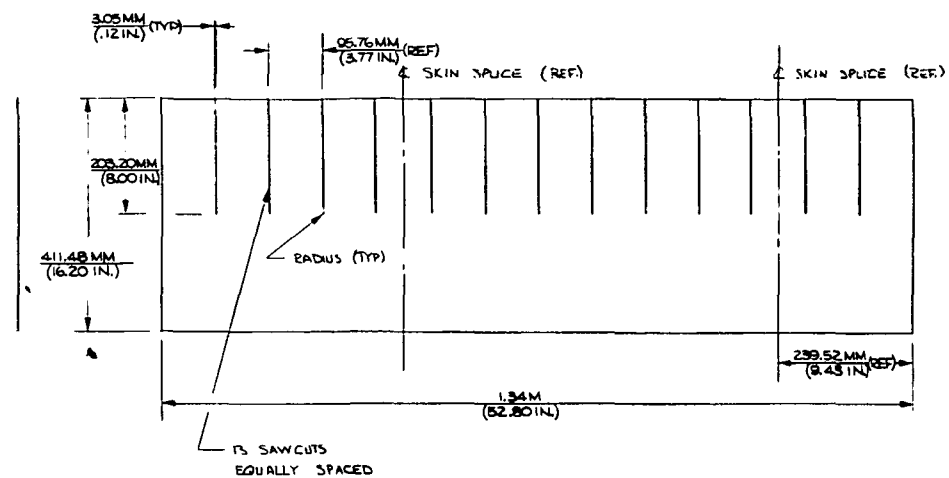
SEE SHEET 1 FOR PARTS LIST AND NOTES		CONTRACT NUMBER <b>NAS 3-15848</b>		THE <b>BOEING</b> COMPANY SEATTLE, WASHINGTON 98108	
DESIGNED BY <b>W. L. BAGLEY</b>	DATE <b>5/1/75</b>	DESIGNED BY <b>W. L. BAGLEY</b>	DATE <b>5/1/75</b>	VACUUM JACKET HEAD ASSEMBLY - HALF SCALE LH <sub>2</sub> TEST MODEL	
PROJECT APPROVAL <b>J. B. 205</b>		PROJECT APPROVAL <b>J. B. 205</b>		SCALE 1/1	
CHANGE/ITER NUMBER		CHANGE/ITER NUMBER		SCALE 1/1	

SK2-5085-140 B3

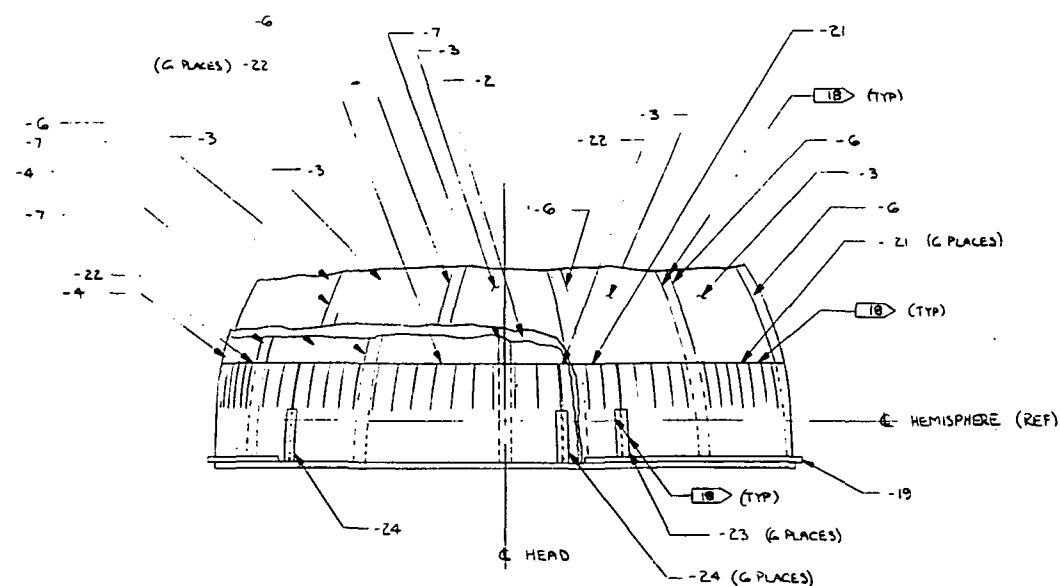
2KS-2082-1403



-22  
(SCALE: 1/4)  
0 40 80 120 160 200  
MM



-21  
(SCALE: 1/4)  
0 40 80 120 160 200  
MM



DETAIL 2C4 (ROTATED 90°)  
(SCALE: 1/10)  
0 100 200 300 400 500  
MM

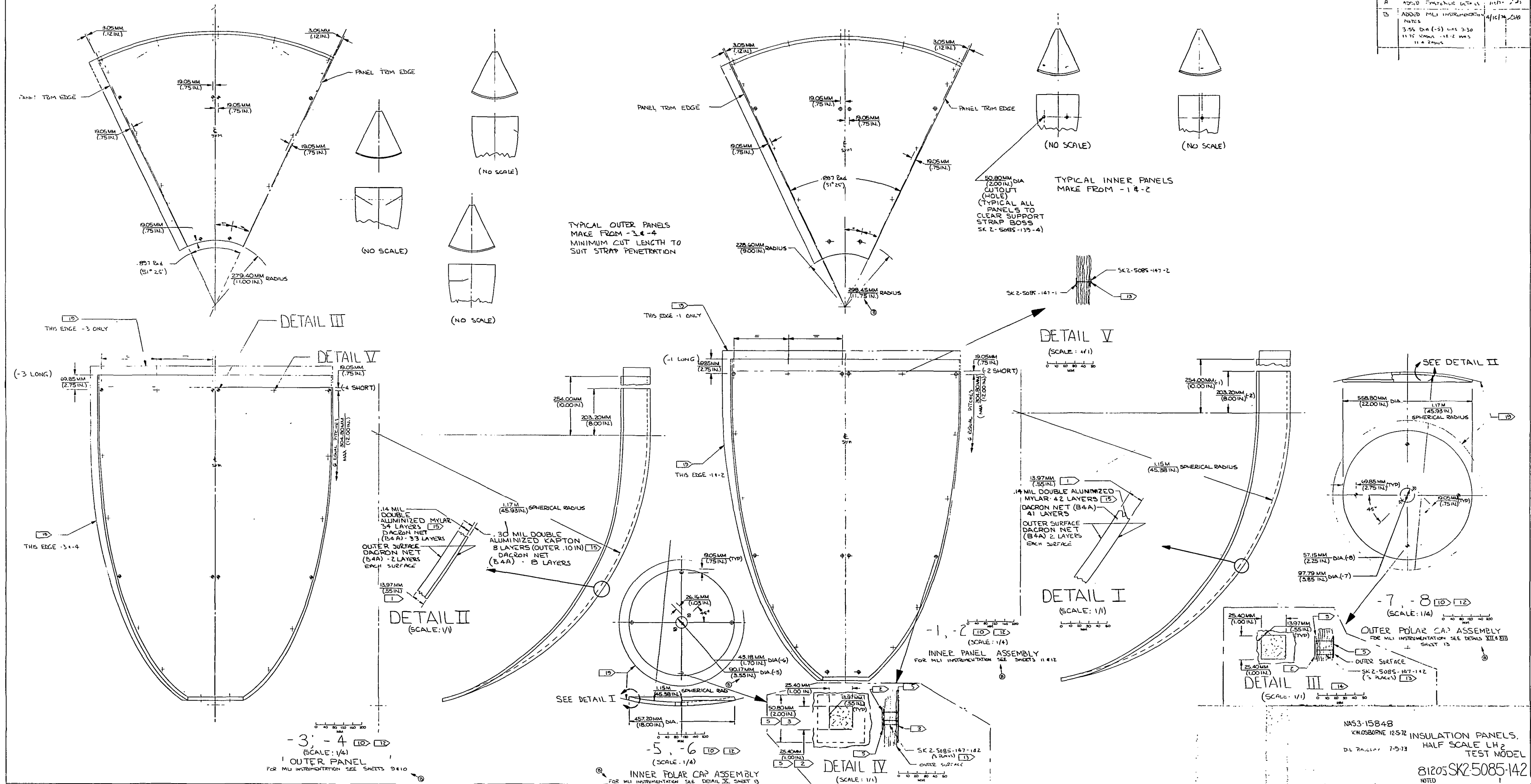
Figure D-12

REVISIONS		DATE	APPROVED
1	REVISED		
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4	REVISED		
5	REVISED		
6	REVISED		
7	REVISED		
8	REVISED		
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99	REVISED		
100	REVISED		





REVISIONS		DATE	APPRO
1	DESIGNATION		
A	ADDED FINISHING DETAILS	11/27/74	JDS
B	ADDED MLI INSTRUMENTATION NOTES	4/16/74	SLB
	3.55 DIA (-S) WAS 3.30		
	11.75 RADIUS - 18-2 WAS		
	11.4 RADIUS		



## LIST OF MATERIAL

				QTY REQD	PART OR IDENTIFYING NUMBER	NOMENCLATURE OR DESCRIPTION	MATERIAL AND SPECIFICATION
				1	-21	OUTER PANEL ASSEMBLY (SHORT)	9
				1	-20	↑	↑
				1	-19	↑	↑
				1	-18	↑	↑
				1	-17	↓	↓
				1	-16	OUTER PANEL ASSEMBLY (SHORT)	9
				1	-15	OUTER PANEL ASSEMBLY (LONG)	8
				1	-14	↑	↑
				1	-13	↑	↑
				1	-12	↑	↑
				1	-11	↑	↑
				1	-10	↓	↓
				1	-9	OUTER PANEL ASSEMBLY (LONG)	8
				1	-8	OUTER MLI POLAR CAP ASSEMBLY	1
				1	-7	OUTER MLI POLAR CAP ASSEMBLY	1
				1	-6	INNER MLI POLAR CAP ASSEMBLY	1
				1	-5	INNER MLI POLAR CAP ASSEMBLY	1
					-4	OUTER MLI PANEL ASSEMBLY (SHORT)	1
					-3	OUTER MLI PANEL ASSEMBLY (LONG)	1
					-2	INNER MLI PANEL ASSEMBLY (SHORT)	1
					-1	INNER MLI PANEL ASSEMBLY (LONG)	1
				QTY REQD	PART OR IDENTIFYING NUMBER	NOMENCLATURE OR DESCRIPTION	MATERIAL AND SPECIFICATION

## LIST OF MATERIAL

FIGURE D-15

SIZE

A

CODE IDENT NO.

81205

SK 2-5085-142

SCALE

SHEET

2

				LIST OF MATERIAL				REV LTR	A
				QTY REQD	PART OR IDENTIFYING NUMBER	NOMENCLATURE OR DESCRIPTION	MATERIAL AND SPECIFICATION		
A				1	-42	SIMULATED PLMB WRAP SUPPORT			
A				1	-41	SIMULATED PLMB WRAP ASSY. (LONG)			
A				1	-40	SIMULATED PLMB WRAP ASSY. (SHORT)			
A				1	-39	INLET TUBE WRAP SUPPORT			
A				1	-38	INLET TUBE WRAP ASSEMBLY (LONG)			
A				1	-37	INLET TUBE WRAP ASSEMBLY (SHORT)			
				1	-36	INNER PANEL ASSEMBLY (LONG)		6	
				1	-35				
				1	-34				
				1	-33				
				1	-32				
				1	-31				
				1	-30	INNER PANEL ASSEMBLY (LONG)		6	
				1	-29	INNER PANEL ASSEMBLY (SHORT)		7	
				1	-28				
				1	-27				
				1	-26				
				1	-25				
				1	-24				
				1	-23	INNER PANEL ASSEMBLY (SHORT)		7	
				1	-22	OUTER PANEL ASSEMBLY (SHORT)		9	
				QTY REQD	PART OR IDENTIFYING NUMBER	NOMENCLATURE OR DESCRIPTION	MATERIAL AND SPECIFICATION		
				LIST OF MATERIAL					

FIGURE D-16		SIZE A	CODE IDENT NO. 81205	SK2-5085-142
60	SCALE			SHEET 3



1 BLANKET DENSITY = 75 LAYERS/INCH.  
ONE LAYER EQUALS ONE .14 MIL  
DOUBLE ALUMINIZED MYLAR SHIELD  
(OR ONE .30 MIL DOUBLE ALUMINIZED  
KAPTON, WHICHEVER IS SPECIFIED) PLUS  
ONE DACRON NET (B4A) SPACER.

NUMBER OF SHIELDS = 42

NUMBER OF SPACERS = 41

OUTER PANEL SURFACES CONSIST OF  
2 DACRON NET (B4A) LAYERS.

2 VELCRO HOOK #100  
(THE HARTWELL CORP) 1.00 INCH  
WIDE POLYESTER FASTENING TAPE 11

3 VELCRO PILE (THE HARTWELL CORP.)  
2.00 INCH WIDE POLYESTER  
FASTENING TAPE 11

4 EE-6600 (PERMACEL) REFLECTIVE,  
POLYESTER, INTERLINED, PRESSURE  
SENSITIVE TAPE.

Figure D-18

SIZE A	CODE IDENT NO. 81205	SK2-5085-142
SCALE		SHEET 5

5 7402 (BORDON, MYSTIK) ALUMINUM FOIL,  
SILICONE ADHESIVE, PRESSURE SENSITIVE TAPE.  
POSITION TWO FOIL LAYERS TO  
SANDWICH THE TWO OUTER DACRON NET  
LAYERS AND THE NYLON PIN BUTTONS  
BETWEEN THE ADHESIVE BACKED FOIL SURFACES.

6 MAKE FROM -1

7 MAKE FROM -2

8 MAKE FROM -3

9 MAKE FROM -4

10 ALL HOLES, EDGES, AND SLITS  
CUT NORMAL TO PANEL CURVATURE

11 BOND WITH KNIFE COAT OF  
EC 3532 (3M) STRUCTURAL ADHESIVE

12 FASTENER AND THERMOCOUPLE DIMENSIONS  
MEASURED ALONG PANEL CURVATURE

FIGURE D-19

SIZE A	CODE IDENT NO. 81205	SK2-5085-142
SCALE		SHEET 6

- 13 HEAT FORM RETENTION BALL
- 14 LOCATIONS TO MATCH DETAIL IV  
LOCATIONS ON INNER PANELS
- 15 FORM SHIELD TO CONTOUR BY DRAPING  
OVER LAYUP TOOL THEN FOLDING  
THE WRINKLES ALONG THE EDGES FLAT.  
TAPE FOLDS IN PLACE WITH 4,  
1.00 x .15 IN. ARRANGE FOLDS TO AVOID  
A LOCAL THICKNESS BUILDUP IN THE  
BLANKET.
- 16 TAPE THERMOCOUPLES IN PLACE  
WITH 4, 1.00 x .15 IN.
- 17 E-36-2-507-001 #36 CHROMEL  
-CONSTANTAN THERMOCOUPLE WIRE,  
TEFLON INSULATED.
- 18 RADIATION SHIELDS ARE NUMBERED  
STARTING FROM THE INNER LAYER  
OF EACH PANEL.
- 19 EXTEND OUTER SURFACE DACRON NET  
(B4A), 2 LAYERS, 1.50 IN. BEYOND TRIM  
EDGE OF PANEL AT EDGE SHOWN.

FIGURE D-20

SIZE A	CODE IDENT NO. 81205	5K2-5085-142
SCALE		SHEET 7



# ASSEMBLY PROCEDURES

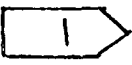
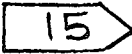

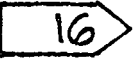
1. WHITE NYLON GLOVES OR EQUIVALENT SHALL BE USED IN HANDLING ALL MATERIAL TO INSURE CLEANLINESS. ALL MATERIALS SHALL BE FREE OF DIRT, GREASE, OR OTHER CONTAMINATION MATERIAL.
2. LAY UP ALTERNATE LAYERS OF 1 RADIATION SHIELDS AND DACRON NET PER  . CONTOUR SHIELDS PER .
3. WHEN SPECIFIED, INSTALL THERMOCOUPLES  DURING INSULATION LAY-UP. TAPE THERMOCOUPLES IN PLACE PER .
4. WHITE NYLON COATS OR EQUIVALENT SHALL BE WORN DURING FABRICATION AND INSTALLATION OF MLI PANEL ASSEMBLIES TO INSURE INSULATION CLEANLINESS.

FIGURE D-21

SIZE A	CODE IDENT NO. 81205	SK2-5085-142
SCALE		SHEET 9



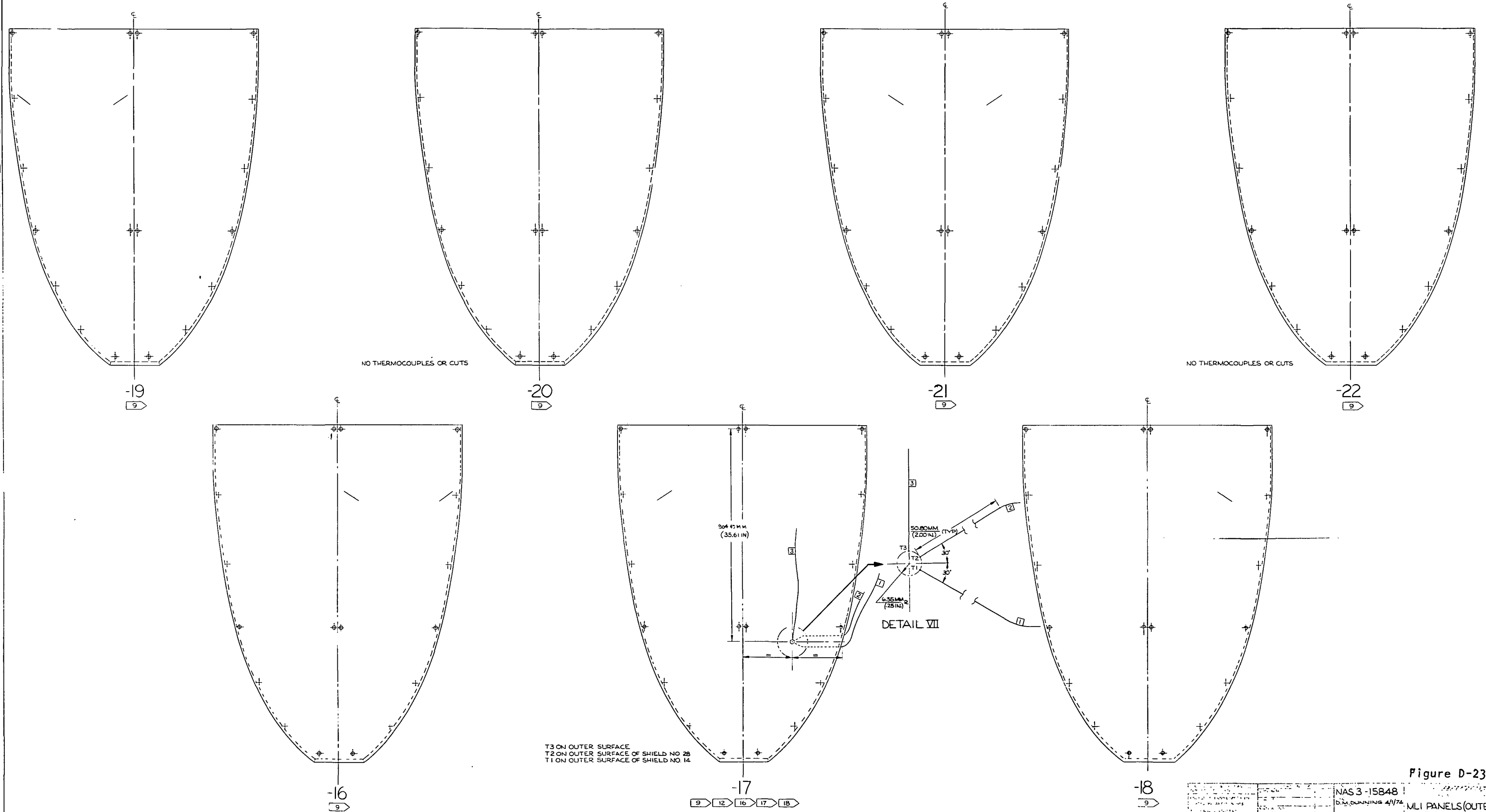


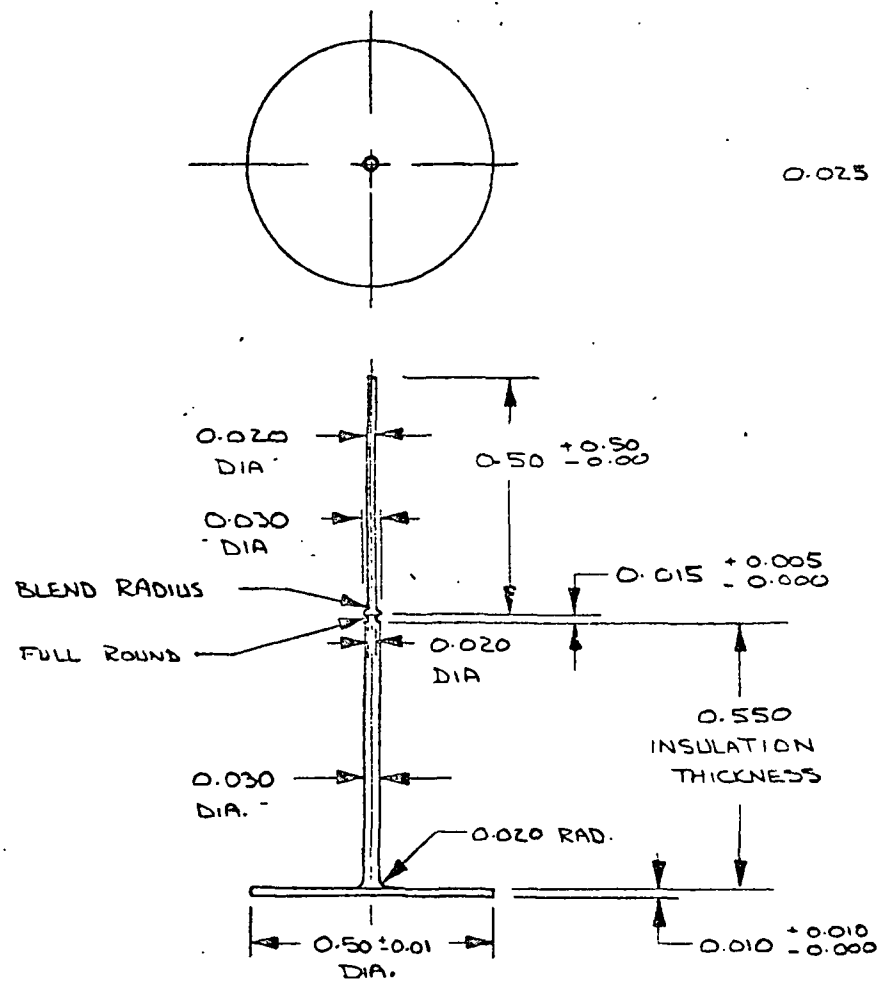
Figure D-23

NAS 3-15848	MLI PANELS (OUTER SHORT)
D.M. DUNNING 4/1/74	HALF SCALE LH <sub>2</sub>
D.L. BARCLAY 4/1/77	TEST MODEL
81205 SK2-5085-142	



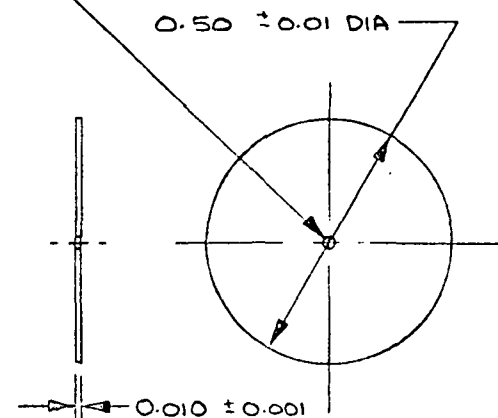


[illegible]



- 1  
 BUTTON - PIN STUD  
 MAT'L: ZYTEL 103HSI-L NYLON RESIN  
 TOLERANCE ON X.XXX IS ± 0.002  
 (SCALE 4/1)

0.025 DIA ± 0.002



- 2  
 BUTTON  
 MAT'L: ZYTEL 103HSI-L NYLON RESIN  
 (SCALE 4/1)

FIGURE D-27  
 SK 2-5085-147 — MLI FASTENERS

## REVISIONS

DESCRIPTION	DATE	APPROVE
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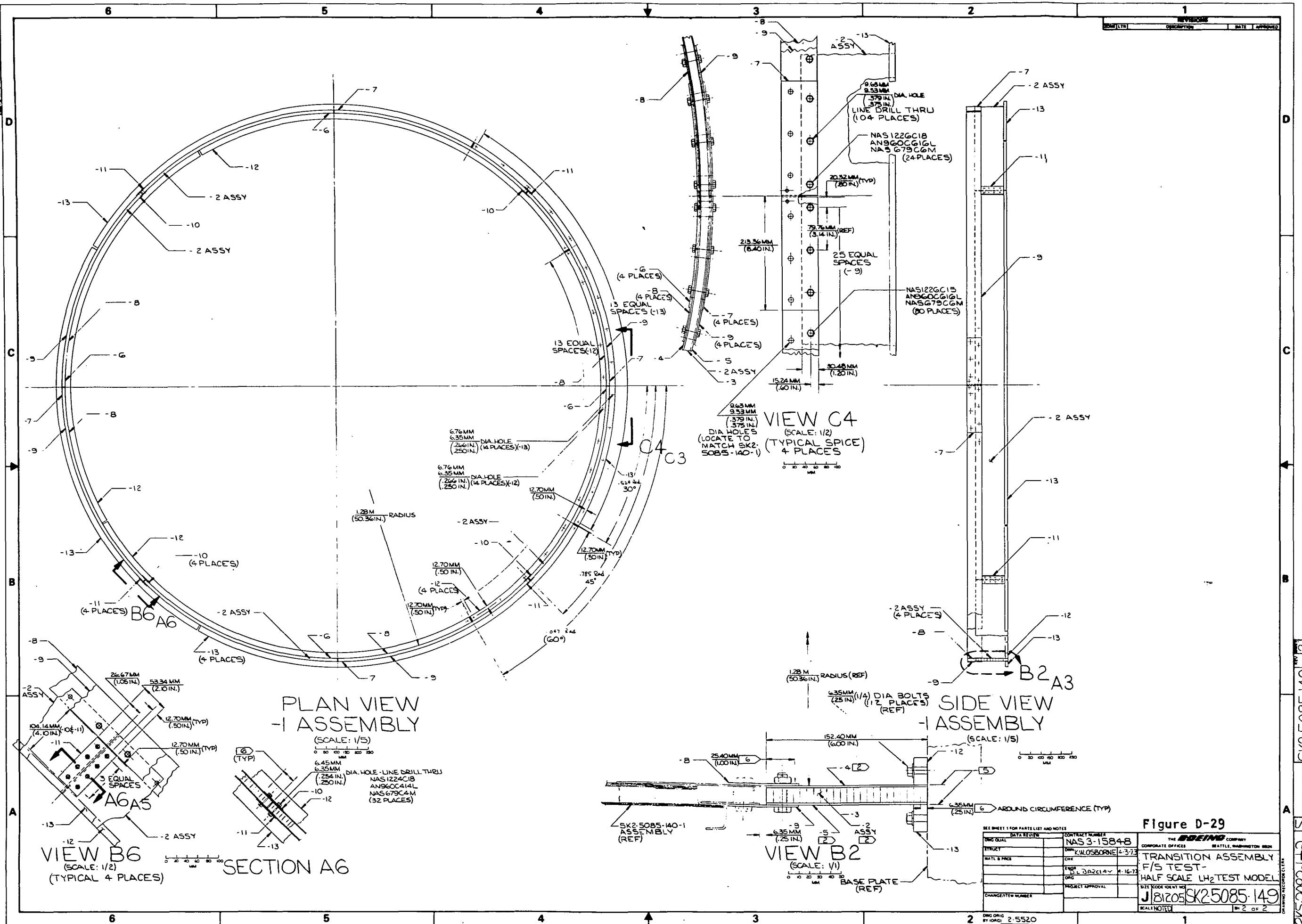
DITHER/DOWNDRAW & TOLERANCING PER ANSI (UNAS) Y14.5		CONTRACT NUMBER <b>NASS-3158AB</b>		THE <b>SPRINT</b> COMPANY CORPORATE OFFICES SEATTLE, WASHINGTON 98101	
UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES TOLERANCES: ANGLES ± .1° DECIMALS .XXX ± .005 DECIMALS .XXXX ± .0005 RIVET & BOLT EDGE MARGINS AS SHOWN RADIO ± .01 IN. .05 & .06 ± .03 IN. .06 & GREATER SHEET METAL CORNER RADIUS INT. ± .16 EXT. ± .20		DRAWING DATE & PRICE CHANGE/ITER NUMBER		DATE <b>04/05/2005</b> CHK <b>1</b> JAN <b>1</b> DATE <b>04/05/2005</b> PROJECT APPROVAL	
FASTENER CODES		2		1	
		2-5520		SK2-5085-149	

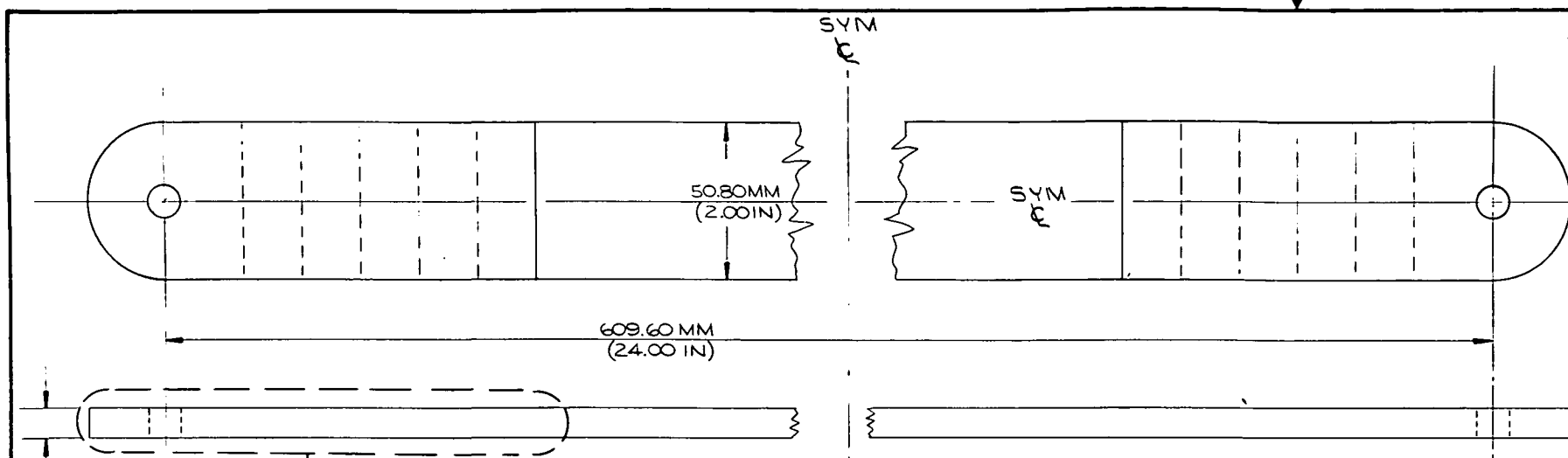
SK2-5085-49

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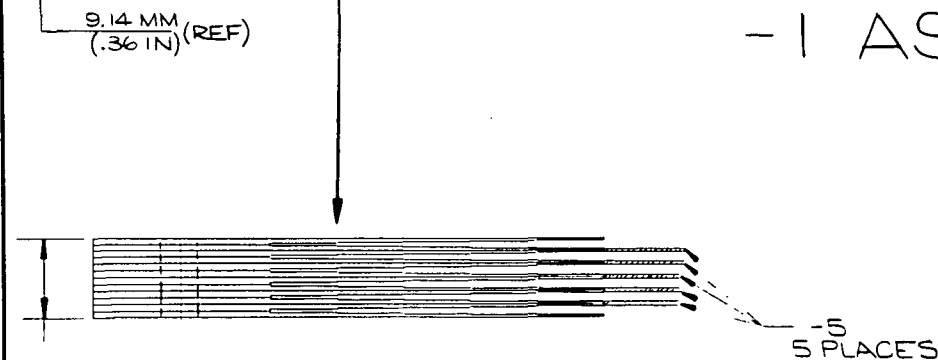
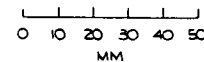
Figure D-28



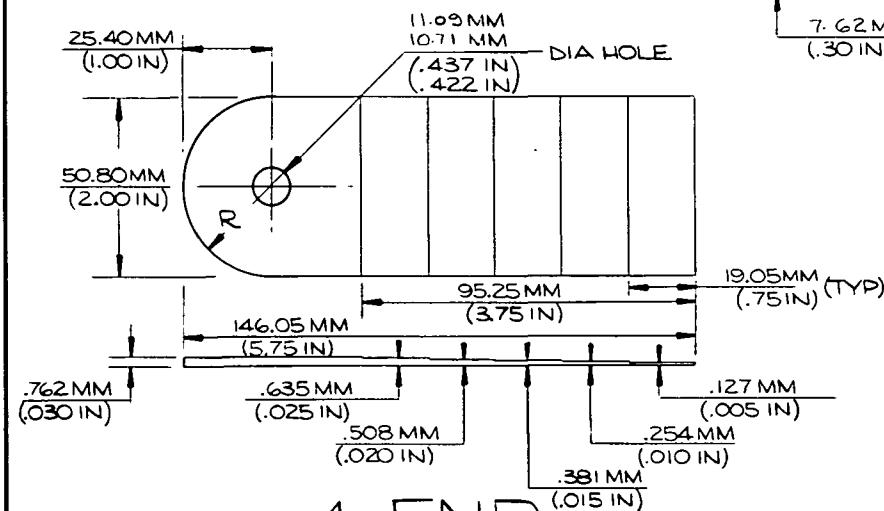




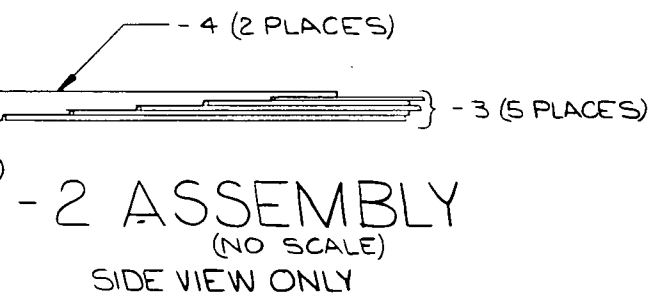
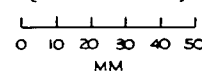
# - 1 ASSEMBLY (SCALE: 1/1)



## DETAIL I (-3's OMITTED FOR CLARITY) (NO SCALE)



## - 4 END FITTING (SCALE: 1/1)



## - 2 ASSEMBLY (NO SCALE) SIDE VIEW ONLY

NOTES  
1. SPECIFICATIONS AND STANDARDS FOR THE CONTROL OF MANUFACTURING OPERATIONS (AS APPLICABLE):  
SURFACE ROUGHNESS SYMBOLS PER BAC 5007  
WELDING AND BRAZING SYMBOLS PER BAC 5002  
FORM, STRAIGHTEN & FIT METAL PARTS PER BAC 5006  
MATERIAL SUBSTITUTIONS & EQUIVALENTS PER BAC 5008  
PART MARKING PER BAC 5307  
BOLT & NUT INSTALLATION PER BAC 5009  
FINISH CODES PER DOCUMENT D7-5080  
RIVET INSTALLATION & SYMBOLS PER BAC 5004

REVISIONS			
LTR	DESCRIPTION	DATE	APPROVED

- CUT FROM 50.80 MM (2.0 IN) WIDE PREPREG [2] P20 49-3 [3] TAPE
- RESIN SYSTEM ERLA 4617 (UNION CARBIDE CORP.) WITH PHENYLENEDIAMINE (CU) CATALYST (ALLIED CHEMICALS)
- PRD-3 SINGLE END YARN - 350 DENIER - 400 FILAMENTS PER YARN - CONTINUOUS - PRODUCERS TWIST - NO FINISH (E.I. DUPONT)
- GAL-4V TITANIUM SHEET PER MIL-T-9046E, TYPE III, COMP C.
- FIBERGLASS EPOXY PRE PREG PER SMS 8-139, TYPE 120, 2 PLY.
- CURE WITH VACUUM BAG PRESSURE TWO HOURS AT 335°K (50°F) & FOUR HOURS AT 422°K (300°F)
- CLEAN FAYING SURFACES OF -4 END FITTINGS PER HYSOL BULLETIN A9-123 & BOND WITH EA 934 (HYSOL DIVISION THE DEXTER CORP) WITH VACUUM BAG PRESSURE PER HYSOL BULLETIN A5-134.
- CHEMICAL MILL PER BAC 5842.

QTY REQD	QTY REQD	CODE IDENT NUMBER	PART OR IDENTIFYING NUMBER	NOMENCLATURE OR DESCRIPTION	MATERIAL AND SPECIFICATION	HT TR	FINISH	PT MLC	REALIZATION	REV LTR
- 5	-	- 5		FILLER	50.80 MM WIDE [5]					
2	-	- 4		END FITTING	.76x 50.80 x 146.08 mm [4] [8]					
5	-	- 3		TAPE	50.80 MM WIDE [1]					
- 12	-	- 2		LAMINATE ASSEMBLY	[6]					
-	-	- 1		SUPPORT STRAP ASSEMBLY	[7]					

DIMENSIONING & TOLERANCING PER ANSI (UNLESS OTHERWISE SPECIFIED) DIMENSIONS ARE IN INCHES TOLERANCES: ANGLES: 1° DECIMALS: .XX: .02 DECIMALS: .XXX: .001 RIVET & BOLT EDGE MARG: .05 BEND RADI: 1.01 ON .03 & .05 1.03 ON .06 & GREATER SHEET METAL CORNER RADI: INT: .16-18 EXT: .22-28		DATA REVIEW STRUCT: [ ] MATERIAL: [ ] CHANGE/ITEM NUMBER: [ ]		CONTRACT NUMBER NAS3-15848 CORPORATE OFFICES K.W. OSBORNE 4-4-3 DL BARCLAY 4-11-3 PROJECT APPROVAL: [ ] SCALE NOTED: [ ]	
--	--	--	--	--	--

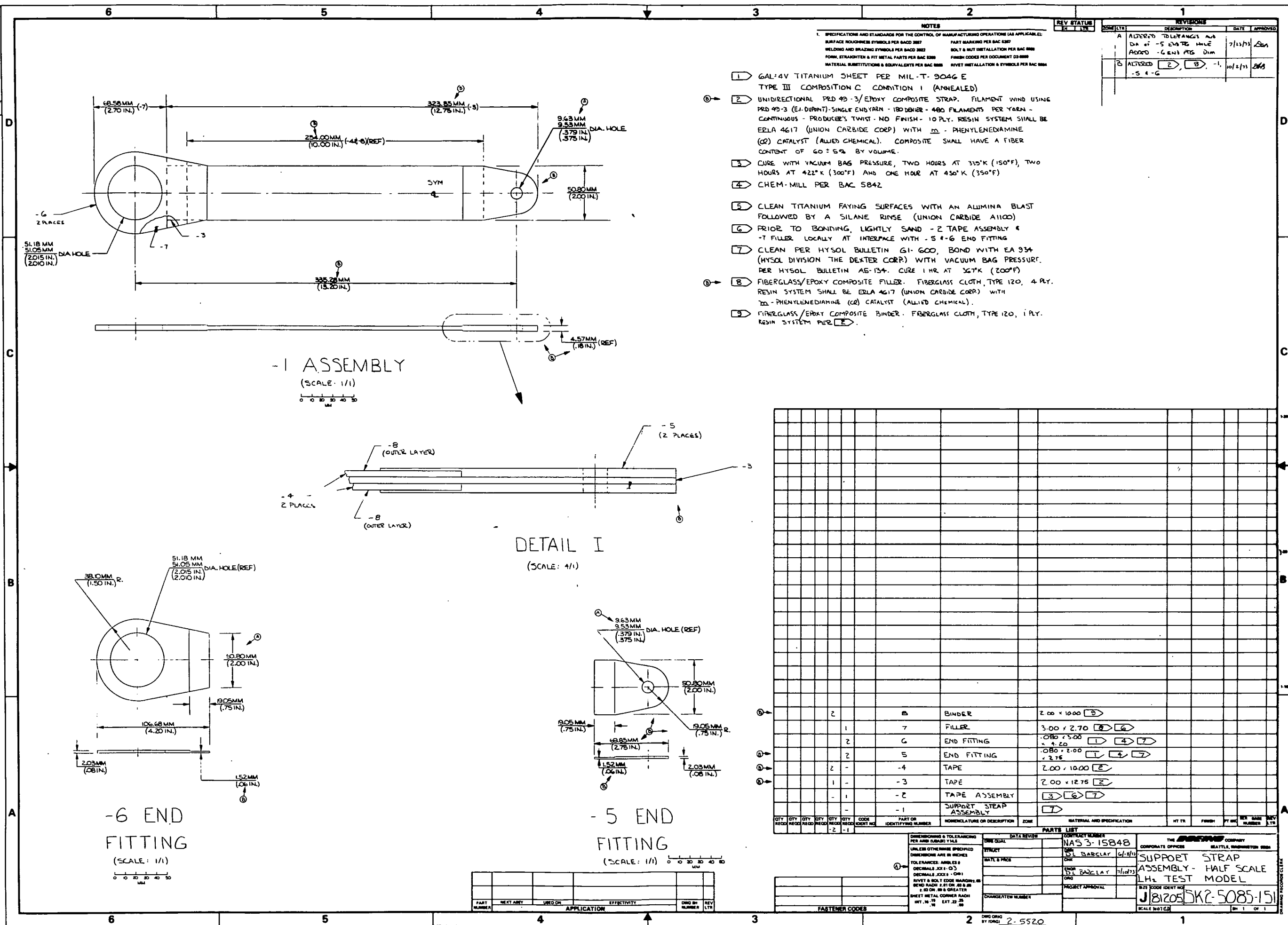
PART NUMBER	REV LTR	REVISION	DATE	DESCRIPTION

FASTENER CODES

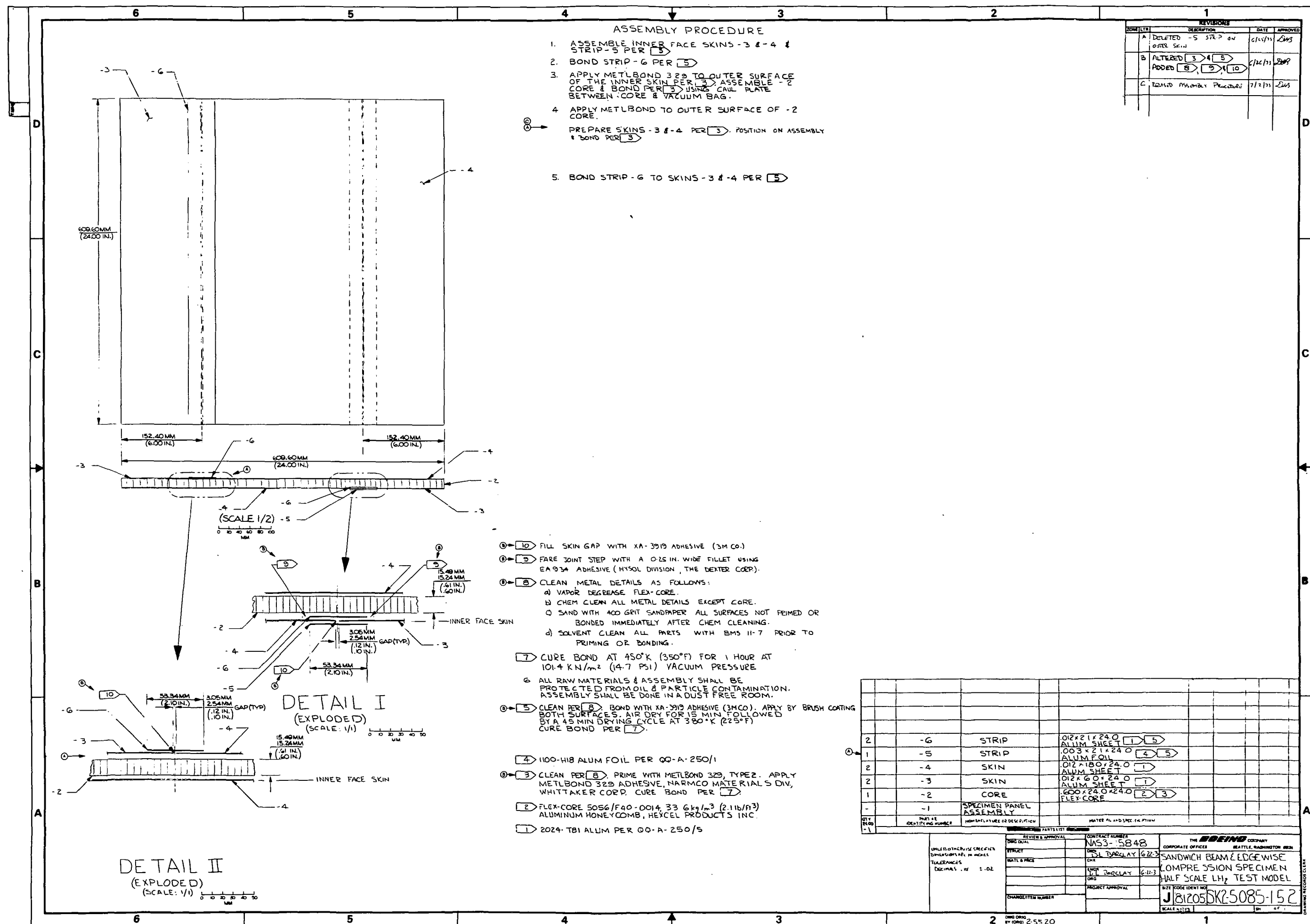
Figure D-30

SK2-5085-133

DO 6000 1225



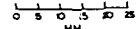
**Figure D-31**







REV STATUS		REVISIONS			
HC	LTR	ZONE	LTR	DESCRIPTION	DATE



- |             |           |         |               |               |          |
|-------------|-----------|---------|---------------|---------------|----------|
|             |           |         |               |               |          |
|             |           |         |               |               |          |
|             |           |         |               |               |          |
| PART NUMBER | NEXT Assy | USED ON | EFFECTIVITY   | DRG IN NUMBER | EXT. LT. |
|             |           |         | APPLICABILITY |               |          |

PLUMBING  
HALF SCALE LH<sub>2</sub>  
L  
2-5085-155

2KS-2082-1221

Figure D-35

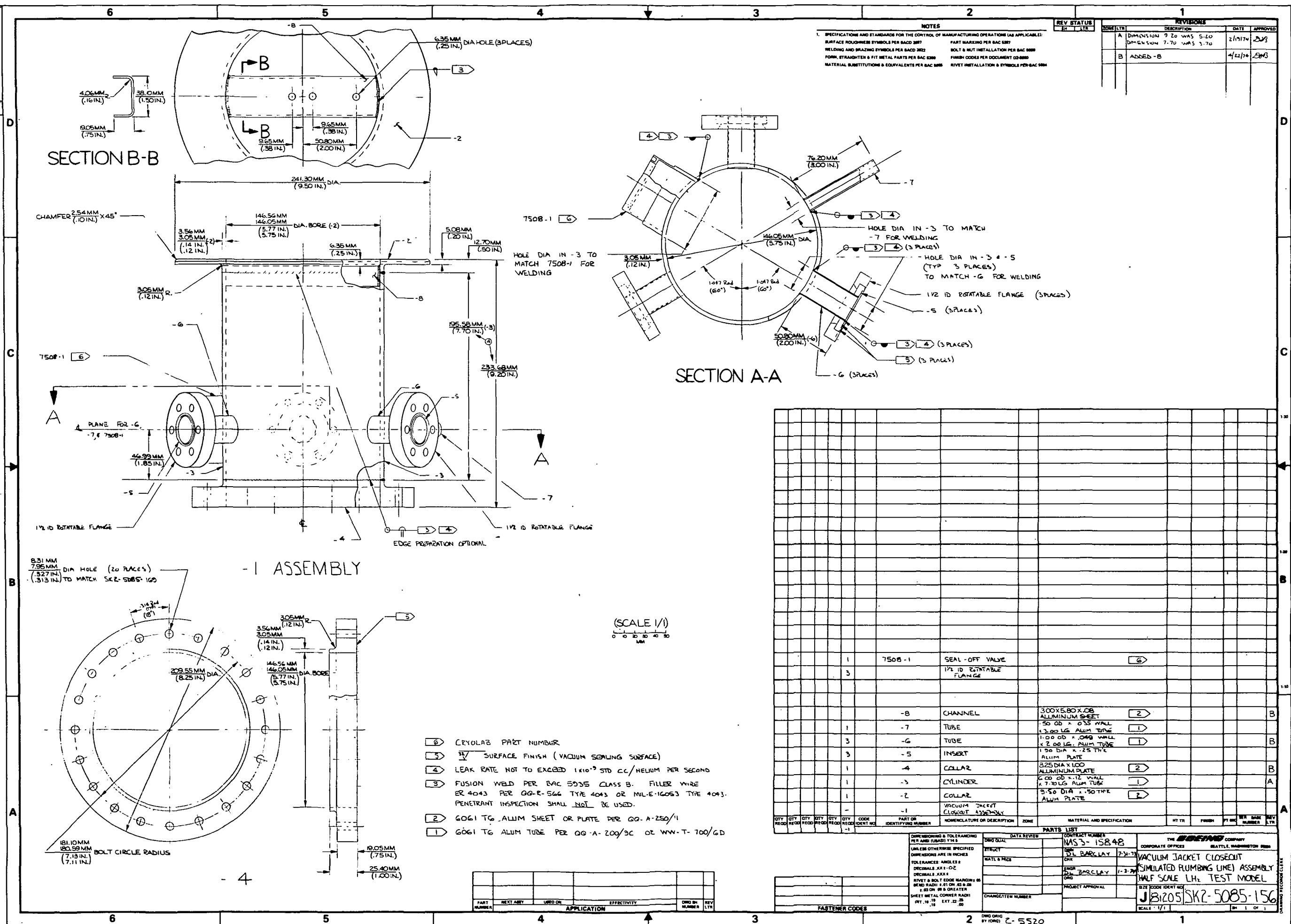
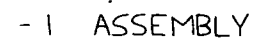



Figure D-26





(SCALE 1/1)



0 10 20 30 40 50  
mm

- ① 321, 302, OR 304 CRES BAR PER QQ-S-763
- ② THREAD UNF-3B 1/2 -20 RIGHT HAND TO CENTER
- ③ THREAD UNF-3B 1/2 -20 LEFT HAND TO CENTER
- ④ THREAD UNF-3A 1/2 -20 RIGHT HAND
- ⑤ THREAD UNF-3A 1/2 -20 LEFT HAND

REV STATUS		REVISION				
BY	DATE	NO	DATE	DESCRIPTION	DATE	APPROVED
		A		ALTERED DIMENSIONS -2 2.00 WAS 3.00 1.00 WAS 1.50 -1.0.4 2.00 WAS 2.75 1.20 WAS 1.55	4/17/74	200

[illegible]

PART NUMBER	REXT ARMY	USED ON	EFFECTIVITY	ONG ON NUMBER
APPLICATION				

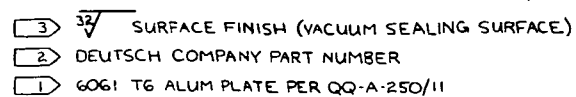
	DIMENSIONS & TOLERANCING PER AMB (USASI) Y14.5	REVIEW & APPROVAL	PARTS	CONTRACT NUMBER	THE <b>ASTM</b> COMPANY
	UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES	DRAWN BY		NAS3-15848	CORPORATE OFFICES SEATTLE, WASHINGTON 98108
	TOLERANCES: ANGLES ± DECIMALS .005 ± DECIMALS .003 ± O.T.O.	STRUCTURE		SILCARPENTIER 10-575	
	RIVET & BOLT EDGE MARKINGS AS SHOWN RADIUS & IT OR .03 & .06 ± .03 ON .06 & GREATER	MATERIAL & PRICE		LIBR <i>Lib. Material 10-575</i>	TURNBUCKLE ASSEMBLY HALF SCALE LH <sub>2</sub> TEST MODEL
	SHEET METAL CORNER RADII INT .16 EXT .26 .16 EXT .00	CHANGE/ITER NUMBER		PROJECT APPROVAL	BASIC CODE IDENT NO. J81205 SK2-5085-163
FASTER CODES					SCALE 1/1" = 1' UNLESS NOTED

SK2-5085-163 A

2K5-2082-1P3

81

Figure D-37

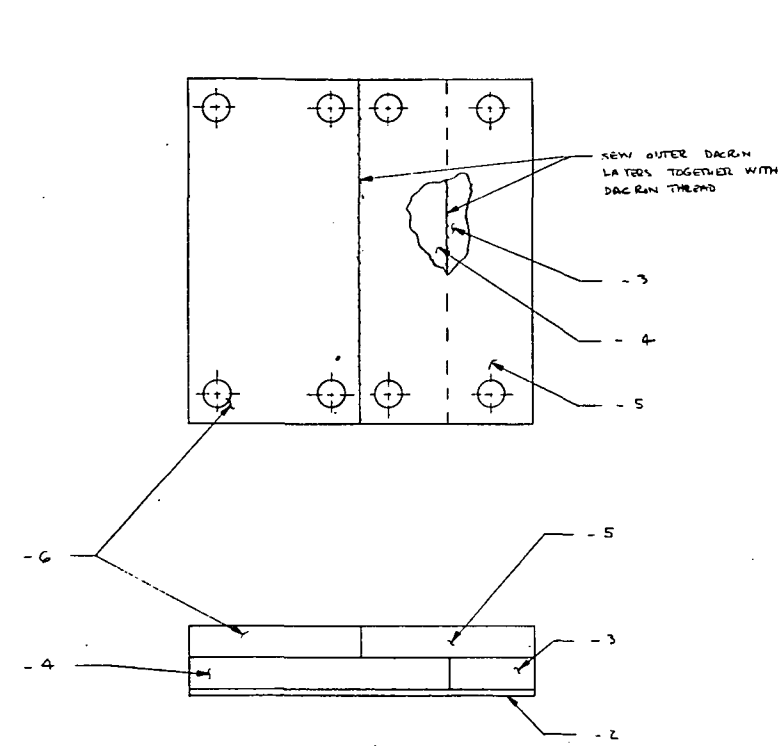


PART NUMBER	NEXT ARMY	USED ON	EFFECTIVITY	CHRG BN NUMBER	PLT NUMBER

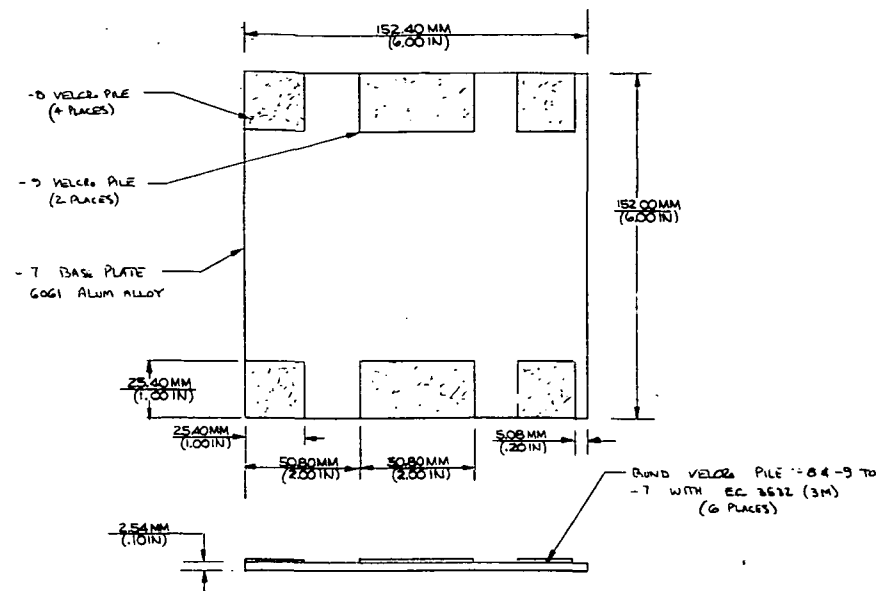
DIMENSIONS AND TOLERANCING PER ANSI (FABRI) Y14.5  UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES  TOLERANCES: ANGLES ± DECIMALS .012 DECIMALS .005  RIVETS & BOLT EDGE MOUNTING: IS REQ. RADII 1.01 ON .005 IS 1.00 ON .005 & GREATER  SHEET METAL CORNER RADII INT .10 EXT .00  EASTMAN CODES	DRAWING NO. 100
---	---

2K5-2082-1e2

Figure D-38



-1 ASSEMBLY

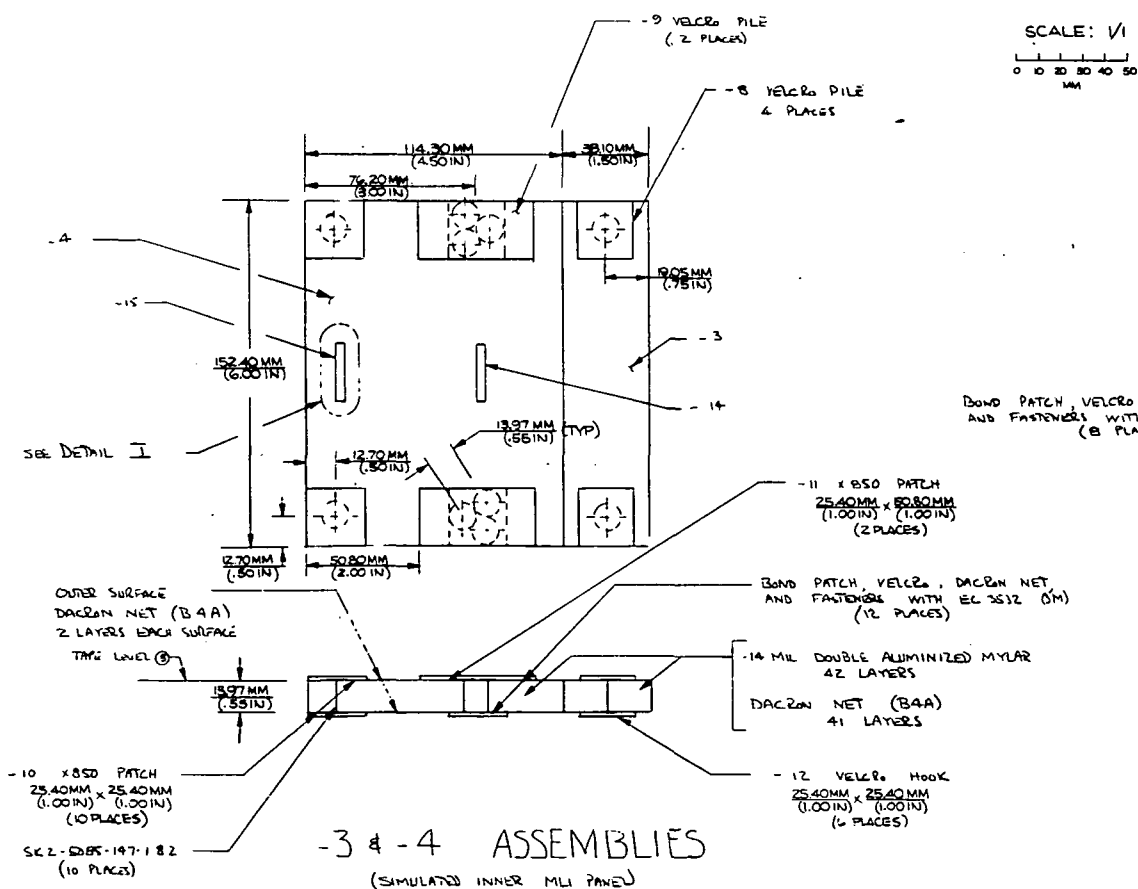


-2 ASSEMBLY  
(SIMULATED PRESSURE VESSEL)

-14 (5 PLACES)  
-15 (5 PLACES)

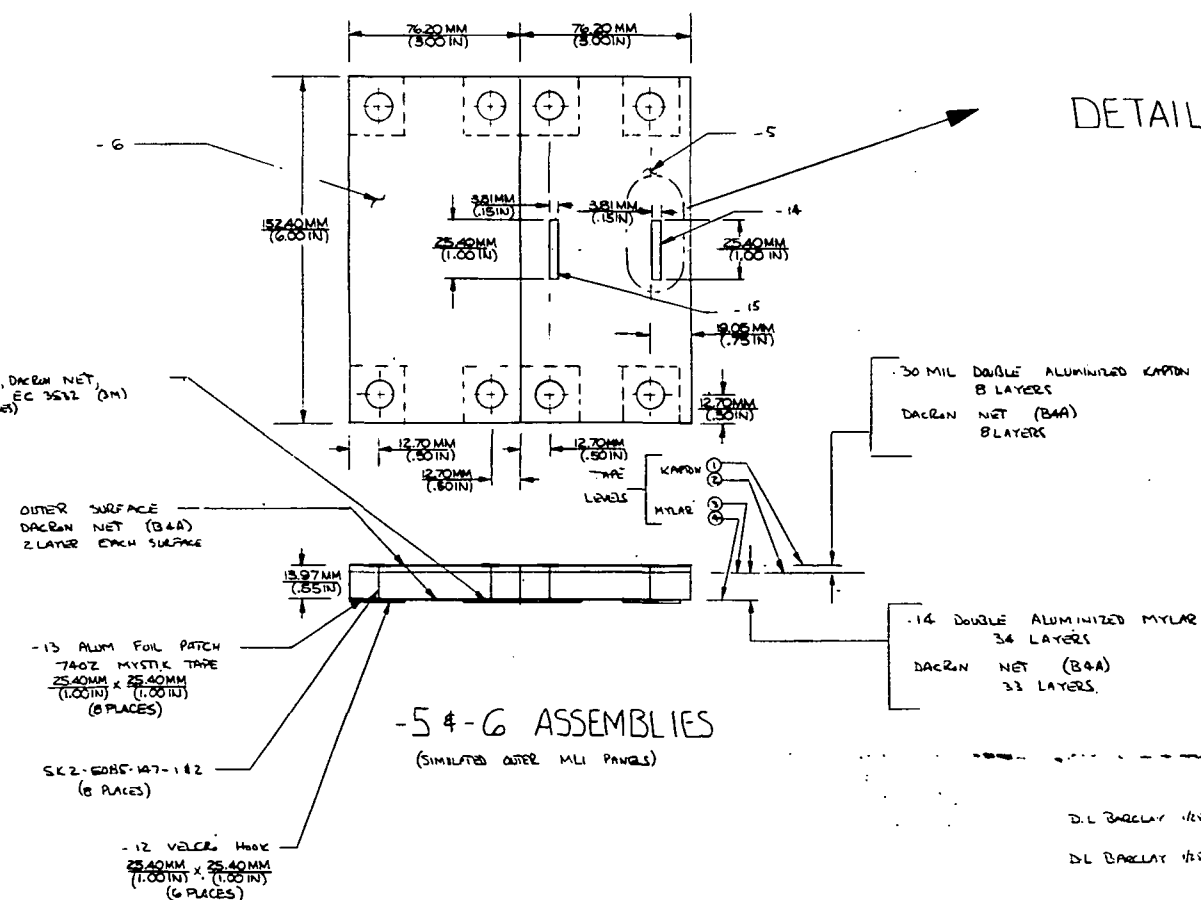
ALUMINIZED MYLAR  
OR KAPTON FOLD AS  
SHOWN

FOLD OUTER KAPTON LAYERS  
ONE EACH AT LEVELS ① & ②  
AND OUTER MYLAR LAYERS  
ONE EACH AT LEVELS ③, ④ & ⑤  
Tape WITH -14 ALUM FOIL 7402 MYSTIC TAPE  
AND -15 6600 PERMACEL TAPE, ONE EACH AT  
EACH LEVEL



-3 & -4 ASSEMBLIES  
(SIMULATED INNER MLI PANEL)

SCALE: 1/1  
0 10 20 30 40 50  
MM



-5 & -6 ASSEMBLIES  
(SIMULATED OUTER MLI PANELS)

DETAIL I

Figure D-39

D.L. BARCLAY 1/17/77 MLI PANEL  
D.L. BARCLAY 1/17/77 THERMAL TEST  
SPECIMEN

81205 SKZ-5085-172



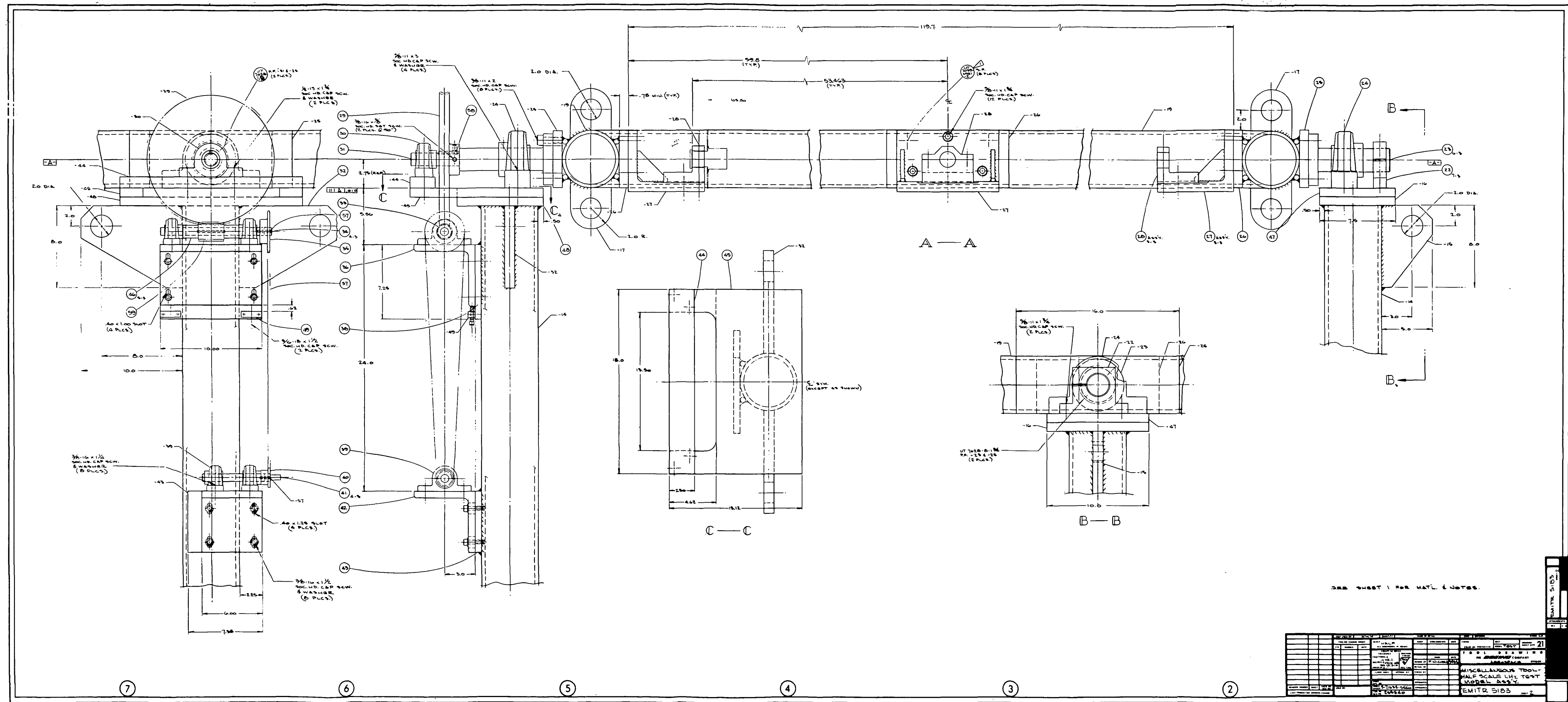
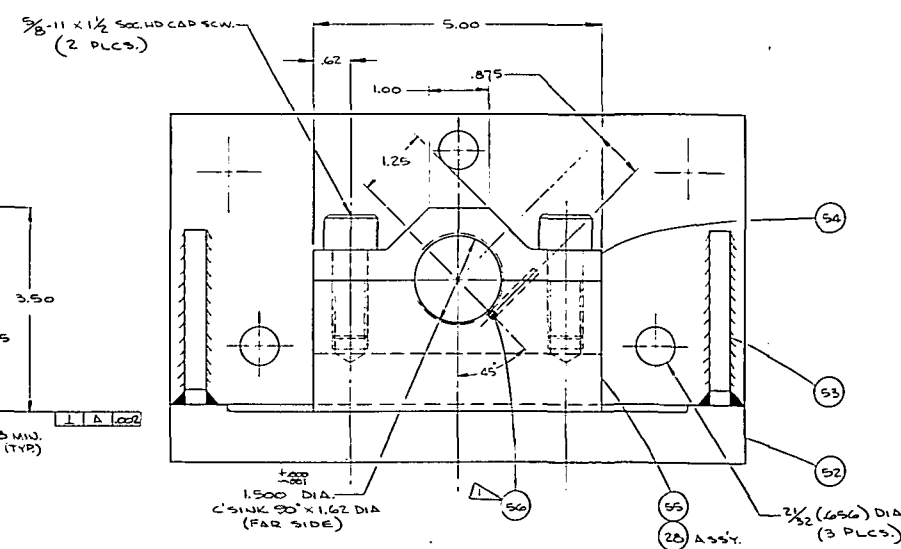
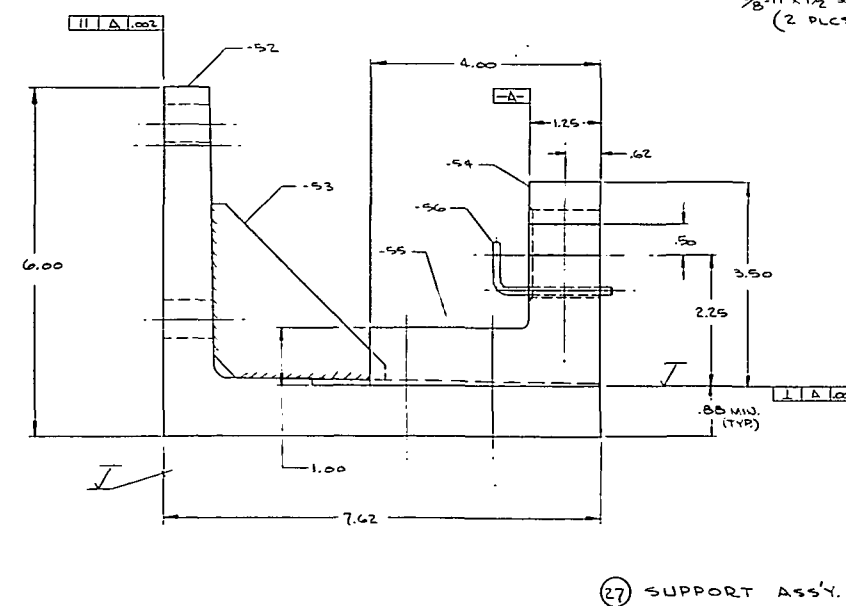
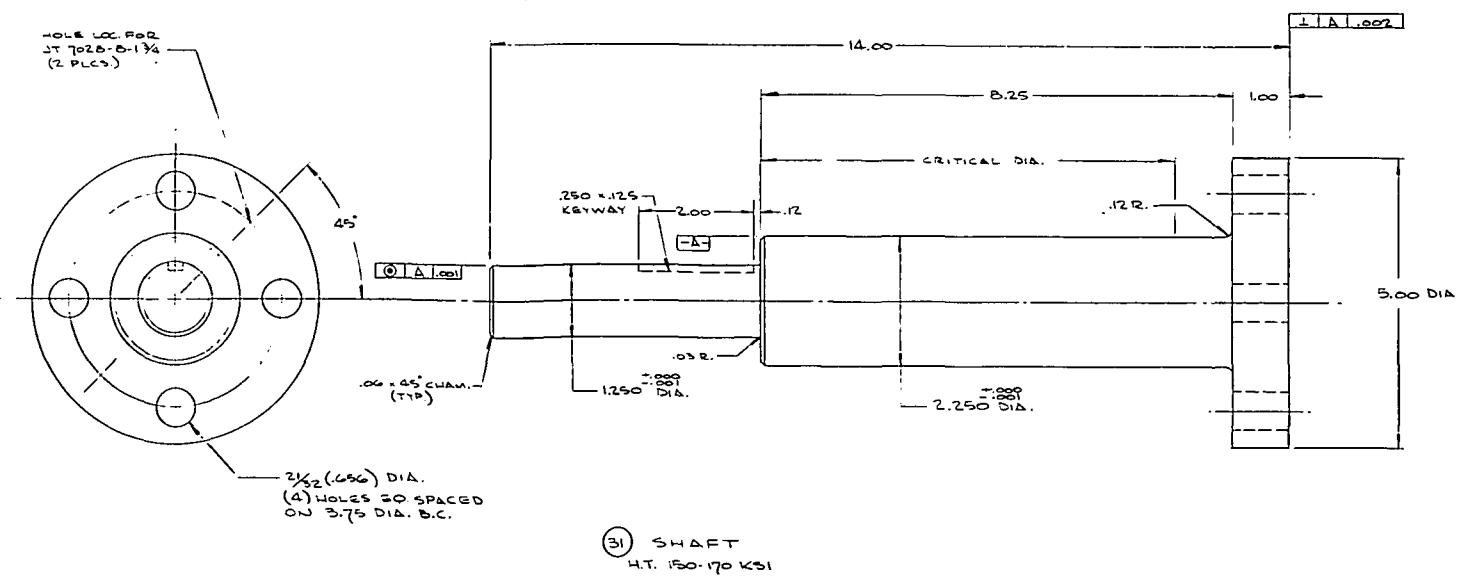
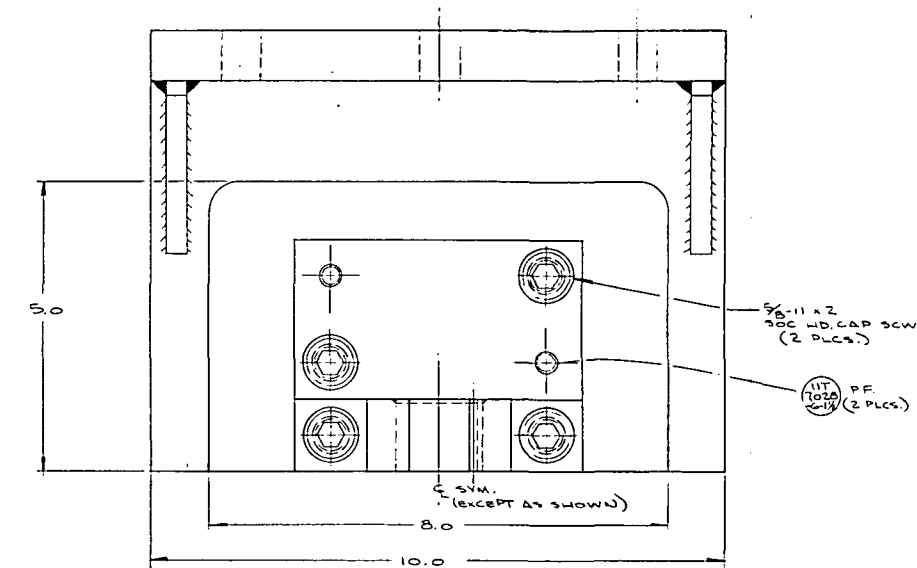
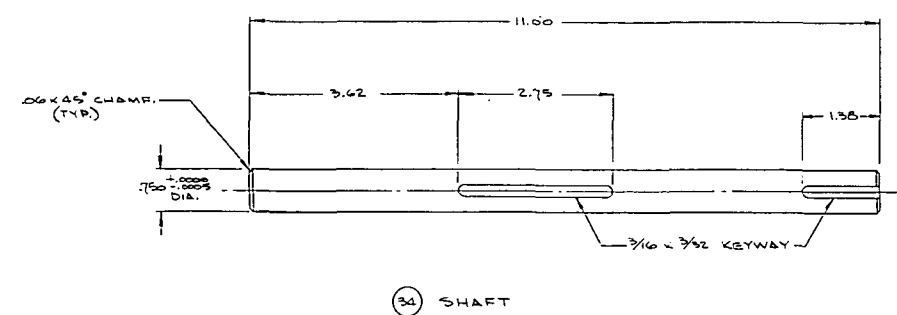
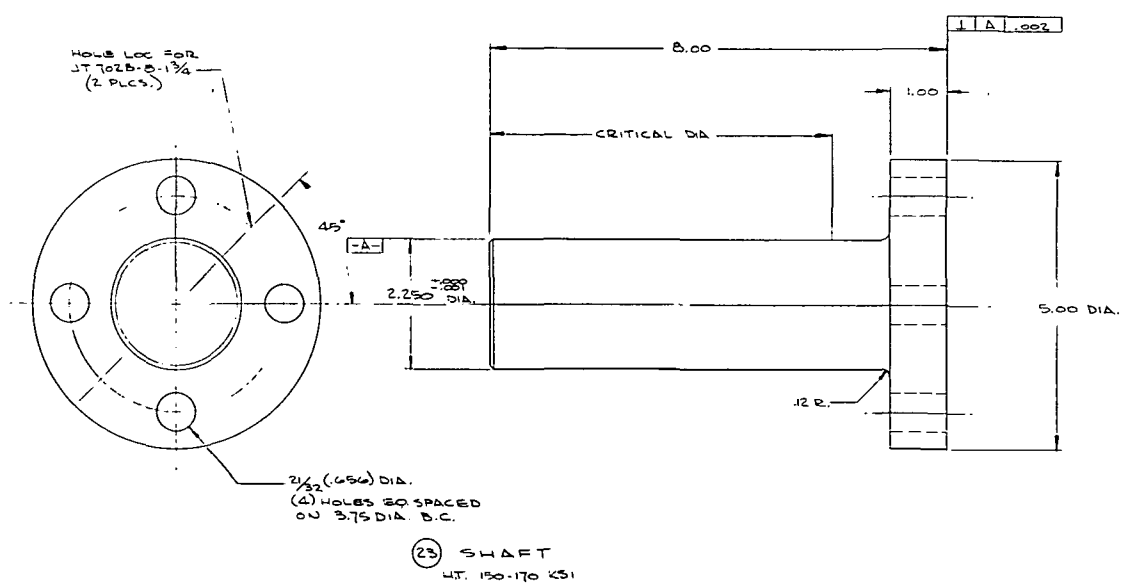
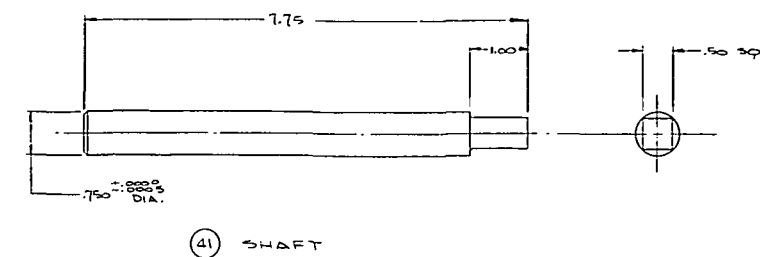
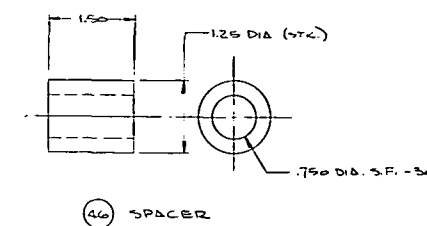
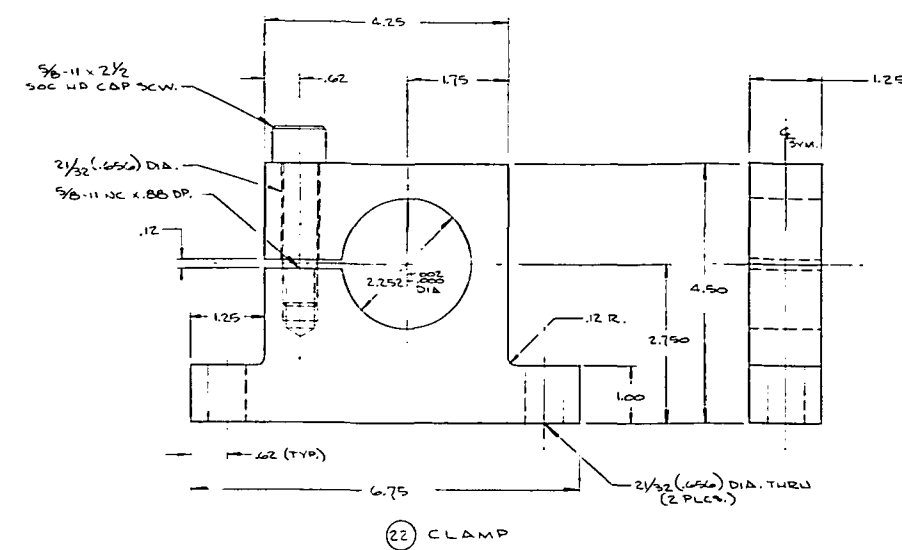


Figure D-41



SEE SHEET 1 FOR MAT'L & NOTES

REV		DATE	BY	CHKD	APP'D	DESCRIPTION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REVISION	DATE	REV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APPENDIX E

THERMAL INSTRUMENTATION TEST PLAN  
FOR THE LH<sub>2</sub> TEST MODEL

APPENDIX E  
THERMAL INSTRUMENTATION TEST PLAN FOR  
THE LH<sub>2</sub> TEST MODEL

The thermal instrumentation plan was developed from the Contract NAS 3-15848 Statement of Work instrumentation requirements. Thermocouple locations and wire routing are shown in Appendix D, Figures D-22 through D-26.

Table E-1 is a list of all thermocouples. The thermocouple entries in the list are grouped by the type of wire required and the lengths shown were used in determination of total quantities of wire required for the test. Four types of wire were used: Teflon insulated Chromel-Constantan for thermocouples in the evacuated volume; fiberglass insulated Chromel-Constantan for exterior locations; vinyl insulation Chromel-Constantan for hydrogen filled spaces; and vinyl insulated copper wire for LH<sub>2</sub> reference junctions and leads. The computations for required wire lengths included added length margins to facilitate component assembly and disassembly.

Table E-2 lists those thermocouples which comply with the Statement of Work requirements. In some of the areas specified, a greater number of thermocouples than the minimum required were installed; also, a number of thermocouples were installed in areas not mentioned in the reference.

A practical limitation on the number of thermocouples located in the evacuated volume exists by virtue of the available space and cost for vacuum tight connectors. Four Deutsch Series DM-5623 connectors, located in the cover plate of the simulated plumbing penetration assembly housing, as shown in Figure D-38 was used. These connectors contain 37 contracts each, providing circuit connectors for 74 thermo-couples. These 74 thermo-couples are listed as the first group in Table E-1.



The Deutsch connectors incorporate 18 Constantan and 19 Chromel pins each. Therefore, of the 74 thermocouples, 2 thermocouple circuits must use Chromel pins to connect their Constantan lead wires. Any temperature differences between the resulting superfluous Chromel-Constantan junctions will generate extraneous voltages, causing errors in the indicated temperatures. In order to minimize these possible errors, thermocouples in high temperature areas, i.e., thermocouples whose unaltered output voltage will be relatively high, were selected for routing through the all-Chromel connector pins. These thermocouples are T59 and T60 on the inner surface of the vacuum jacket. Any of the other thermocouples on this surface would have been suitable, except T56 or T62, for which accuracy is somewhat more important due to their association with thermocouple sets in the MIL.

Three thermocouples, T72, T73, and T74 were to be immersed in the  $\text{GH}_2$  boil-off flow in the vent tube in an attempt to measure the temperature rise in the gas as it travels up the tube. The possibility of bonding small fins to these junctions to increase their response to changes in gas temperature was explored. Because of the need for maximum accuracy for these temperatures, relative to the saturated vapor temperature, these thermocouple circuits were to employ reference junctions within the  $\text{LH}_2$  in the test tank rather than reference junctions in the cold box like all other thermocouples.

Thermocouple locations and wire routings were planned to achieve the best combination of accurate, useful data and minimum disruption. In some areas, these requirements conflicted and compromises were necessary. All thermocouples on or within the MLI were attached to shield layers with an aluminized Mylar tape having reflective properties very close to those of the shields themselves. When possible, the first 6 inches of wire leading to the thermocouple junction followed expected isotherm contours in order to minimize errors due to heat flow in the wires. On tubular components and support straps, loads made at least 2 turns around the component being routed away from the location of the junction.

Location and routings were planned to minimize junction and wire overlaps, avoiding the MLI thickness disruption that would result from excessive superposition. This is the reason for the rosette patterns for thermocouples T1 through T6, T7 through T12, and T25 through T30, and the staggering of groups T13 through T18 and T19 through T24.

# THERMOCOUPLE LIST

T/C IDENT.	COMPONENT	LOCATION ON COMPONENT	WIRE TYPE	LENGTH "A"	LENGTH "B"	LENGTH " "	NOTES
				(in)	(in)	(in)	
T1	MLI PANEL	-17	SHIELD 14	CHP-COH, TEFLON	151.7	187.7	(1) (2) (3)
T2			↓ 28		198.7	184.7	(1)
T3			↓ OUT. SURF.		132.7	179.7	
T4		-30	SHIELD 14		165.4	201.4	(1)
T5			↓ 28		169.4	205.4	(1)
T6			↓ OUT. SURF.		153.7	189.7	
T7		-10	SHIELD 14		78.0	114.0	(1)
T8			↓ 28		79.0	110.0	(1)
T9			↓ OUT. SURF.		69.9	100.9	
T10		-23	SHIELD 14		91.6	127.6	(1)
T11			↓ 28		95.6	131.6	(1)
T12			↓ OUT SURF.		80.0	116.0	
T13		-35	SHIELD 14		111.6	147.6	(1)
T14			↓ 28		116.6	152.6	(1)
T15			↓ OUT. SURF.		110.5	146.5	
T16		-10	SHIELD 14		102.4	144.4	(1)
T17			↓ 28		111.5	147.5	(1)
T18			↓ OUT. SURF.		109.0	145.0	
T19		-9	OUT. SURF.		86.9	122.9	
T20			SHIELD 28		94.4	130.4	(1)
T21	↓	↓	↓ 14		95.4	131.4	(1)
T22	LOWER TEN-SION STRAP		↓ OUT. SURF.		104.3	140.3	
T23	MLI PANEL	-23	SHIELD 28	↓	99.4	135.4	(1) ↓ ↓

	INITIALS	DATE	REV. BY INITIALS	DATE	TITLE	MODEL
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# THERMOCOUPLE LIST

T/C IDENT.	COMPONENT	LOCATION ON COMPONENT	WIRE TYPE	LENGTH "A"	LENGTH "B"	LENGTH "C"	NOTES
				(in)	(in)	(in)	
T 24	MLT PANEL	-23	SHIELD 14	CHR-COI, TEFLOH	98.4	134.4	(1) (2) (3)
T 25		-8	OUT. SURF.		29.5	65.5	
T 26			SHIELD 28		42.0	78.0	(1)
T 27			↓ 14		41.6	77.6	(1)
T 28		-6	OUT. SURF.		39.5	75.5	
T 29			SHIELD 28		39.7	75.7	(1)
T 30			↓ 14		40.8	76.8	(1)
T 31	LOWER TENSION STRAP	E, OUT. SURF.			105.8	141.8	
T 32					102.8	138.8	
T 33					101.3	127.3	
T 34					99.8	135.8	
T 35	UPPER TENSION STRAP				121.6	157.6	
T 36					120.1	156.1	
T 37					118.6	154.6	
T 38					117.1	153.1	
T 39					115.6	151.6	
T 42	GIRTH RING	IN. SURF.			94.8	130.8	
T 43					94.8	130.8	
T 56	VAC. JACKET UPPER HEAD	IN. SURF., OVER T1-T6			137.2	173.2	
T 57		IN. SURF., 60° FROM T56			137.2	173.2	
T 58		IN. SURF., 120° FROM T56			137.2	173.2	
T 59		IN. SURF., 180° FROM T56			137.2	173.2	
T 60		IN. SURF., 240° FROM T56			137.2	173.2	

	INITIALS	DATE	REV. BY INITIALS	DATE	TITLE	MODEL
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# THERMOCOUPLE LIST

T/C IDENT.	COMPONENT	LOCATION ON COMPONENT	WIRE TYPE	LENGTH "A"	LENGTH "B"	LENGTH "C"	NOTES
				(in)	(in)	(in)	
✓ T61	Y/C. JACKET UPPER HEAD	IN. SURF., 300° FROM T56	CHR-COH, TEFLO	137.2	173.2		(2) (3)
✓ T62	VAC JACKET LOWER HEAD	IN. SURF., OVER T7-T12		57.9	93.9		
✓ T63		IN. SURF., 60° FROM T62		57.9	93.9		
✓ T64		IN. SURF., 120° FROM T62		57.9	93.9		
✓ T65		IN. SURF., 180° FROM T62		57.9	93.9		
T66		IN. SURF., 240° FROM T62		57.9	93.9		
✓ T67		IN. SURF., 300° FROM T62		57.9	93.9		
✓ T68	MLI PANEL -7	SHIELD 34		182.7	218.7		(1)
T69	INLET TUBE	OUT. SURF.		209.3	245.3		
✓ T70				205.3	241.3		
✓ T71	↓	↓		201.3	237.3		
✓ T75	SIM. PLUMB. TUBE	OUT. SURF.		16.8	52.8		
✓ T76				15.3	51.3		
✓ T77	↓	↓		13.8	49.8		
✓ T78	SIM. PLUMB. ASSY	END OF FG INS. RETAINER		16.0	52.0		
T86		TUBE INSUL. OUT. SURF.		30.6	66.6		
✓ T87				29.1	65.1		
✓ T88	↓	↓		27.6	63.6		
✓ T89	MLI PANEL -6	OUT. SURF.		40.3	76.3		
✓ T90		-8		23.3	59.3		
✓ T91		-6		36.3	72.6		
✓ T92	↓	-8	↓	27.3	63.6		
✓ T96	PRESS. VESSEL	OUT. SURF. UNDER T68	✓	182.7	218.7		✓ ✓

	INITIALS	DATE	REV. BY INITIALS	DATE	TITLE	MODEL
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# THERMOCOUPLE LIST

T/C IDENT.	COMPONENT		LOCATION ON COMPONENT		WIRE TYPE	LENGTH "A"	LENGTH "B"	LENGTH " "	NOTES		
						(in)	(in)	(in)			
T 97	MLT PANEL	-9	OUT. SURF.		CHR-CON, TEFLON	83.2	119.2			(2)(3)	
T 98		-9				81.3	117.3				
T 99		-23				92.9	128.9				
T 102	PRESS. VESSEL OUT. SURF.		UNDER T1-T6			167.4	203.4				
T 103			UNDER T7-T12			93.6	129.6				
T											
T											
TOTAL	74	T/C IN VACUUM SPACE:				7030.2	9699.2				
T						=585.25'	=807.85'				
T											
TOTAL	CHR-CON, TEFLON WIRE REQ'D:						808'				
T											
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# THERMOCOUPLE LIST

T/C IDENT.	COMPONENT	LOCATION ON COMPONENT	WIRE TYPE	LENGTH "G"	LENGTH "H"	LENGTH "I"	NOTES
				(in)	(in)	(in)	
T 40	GIRTH RING	OUT. SURF.	CRP-COM, VINYL	96.1	246.1		(8) (9)
T 41	↓	↓		96.1	246.1		
T 44	VAC. JACKET UPPER HEAD	OUT. SURF. OVER T1-T6		139.2	289.2		
T 45		OUT. SURF. 60° FROM T44		139.2	289.2		
T 46		OUT. SURF. 120° FROM T44		139.2	289.2		
T 47		OUT. SURF. 180° FROM T44		139.2	289.2		
T 48		OUT. SURF. 240° FROM T44		139.2	289.2		
T 49	↓	OUT. SURF. 300° FROM T44		139.2	289.2		
T 50	VAC. JACKET LOWER HEAD	OUT. SURF. OVER T7-T12		59.0	209.0		
T 51		OUT. SURF. 60° FROM T50		59.0	209.0		
T 52		OUT. SURF. 120° FROM T50		59.0	209.0		
T 53		OUT. SURF. 180° FROM T50		59.0	209.0		
T 54		OUT. SURF. 240° FROM T50		59.0	209.0		
T 55	↓	OUT. SURF. 300° FROM T50		59.0	209.0		
T 79	SIM. PLUMB. ASSY.	HOUSING CLOSEOUT PLATE		0.	150.0		
T 80	VAC. JACKET UPPER HEAD	NEAR VENT SAME MERIDIAN		187.0	337.0		
T 81	↓	AS T68, T96		182.0	332.0		
T 82	VENT HOUSING	CLOSEOUT EXTENSION		225.3	375.3		
T 83	SIM. PLUMB. ASSY.	HOUSING BASE FLANGE OUT. SURF.		17.2	167.2		
T 100	VAC. JACKET LOWER HEAD	NEAR SIM. PLUMB. PENETRATION		21.2	171.2		
T 101	↓	SAME MERIDIAN AS T91, T92	↓	26.7	176.7		↓ ↓
TOTAL, 21 T/C ON ASSEMBLY EXTERIOR:				2040.8	5190.8		
T				170.07	432.6		

	INITIALS	DATE	REV. BY INITIALS	DATE	TITLE	MODEL
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# THERMOCOUPLE LIST

T/C IDENT.	COMPONENT	LOCATION OF COMPONENT	WIRE TYPE	LENGTH "D"	LENGTH "E"	LENGTH " "	NOTES
				(in)	(in)	(in)	
T72	VENT TUBE	FREE STANDING IN VENT GAS	CHR-CON, VINYL	10.0	16.0		⑤ ⑥ ⑩
T73				20.0	26.0		
T74				29.0	35.0		
T93	FILL LINE	OUT. SURF., IN VENT TUBE		28.5	34.5		
T94				24.5	30.5		
T95				20.5	26.5		
T							
TOTAL	6 T/C IN H <sub>2</sub> VENT SPACE :			132.5	168.5		
T				=11.04'	=14.04'		
T							
T							
T	EXTENSION WIRE FOR T/C's				LENGTH "C"		
T	IN VACUUM SPACE, EA. :				150.0		④
TOTAL	EXT. WIRE, 74 T/C's :				1100.0		
T					=925.0'		
T							
T							
TOTAL	CHR-CON, VINYL WIRE REQ'D :						
T	FOR 21 EXTERIOR T/C's				432.6'		
T	FOR EXTENSIONS FOR 74 VAC. T/C's :				925.0'		
T	FOR 6 T/C's IN H <sub>2</sub> VENT SPACE :				19.0'		
T					1371.6'		
T							

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# THERMOCOUPLE LIST

T/C IDENT.	COMPONENT		LOCATION ON COMPONENT	WIRE TYPE	LENGTH "J"	LENGTH "K"	LENGTH " "	NOTES		
					(in)	(in)	(in)			
TREF72			IMMERSED	Cu	63.6	69.6		⑩	⑪	⑫
TREF73			IN LH <sub>2</sub> ,		63.6	69.6				
TREF74			10" BELOW		63.6	69.6				
TREF93			PRESS. VESSEL		63.6	69.6				
TREF94			VENT OUTLET		63.6	69.6				
TREF95			↓		63.6	69.6				
T										
TOTAL	, 6 REF. JUNCTION LEADS					417.6				
T						=34.8'				
T										
T							LENGTH "F"			
T	EXTENSION WIRE FOR REF.									
T	JUNCTION LEADS, EA :					310.0		⑦	⑩	
TOTAL	, CU EXTENSION WIRE					1860.0				
T						=155.0'				
T										
T										
TOTAL	, CU WIRE REQ'D :					34.8'				
T						155.0'				
T						189.8'				
T										
T										
T										

	INITIALS	DATE	REV. BY INITIALS	DATE	TITLE	MODEL
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## THERMOCOUPLE LIST NOTES

- ① Shields are numbered counting from inner surface.
- ② Length "A" is calculated min. length from T/C to Deutsch vacuum-tight connectors on closure plate of simulated plumbing housing at bottom of vac. jacket.
- ③ Length "B" = Length "A" plus 36.0".
- ④ Length "C" is length of extension required from Deutsch connectors to terminals at reference box.
- ⑤ Length "D" is calculated min. length of Chromel-Constantan wire from T/C to reference junction in LH<sub>2</sub>.
- ⑥ Length "E" = Length "D" plus 6.0".
- ⑦ Length "F" = Length of Cu wire extension from H<sub>2</sub>-tight connector on vent plenum to terminals at reference box.
- ⑧ Length "G" is calculated length from T/C to plane of closure plate of simulated plumbing housing at bottom of vac. jacket.
- ⑨ Length "H" = Length "G" plus 150.0 in. additional required from plane of closure plate to terminals at reference box.
- ⑩ To be installed by Tulalip Test Site personnel.
- ⑪ Length "J" is calculated min. length of Cu wire from reference junctions to H<sub>2</sub>-tight connector.
- ⑫ Length "K" = Length "J" plus 6.0".

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TABLE E-2  
THERMOCOUPLE LOCATIONS FOR WORK STATEMENT COMPLIANCE

Work Statement Location Description	Thermocouples
"... six different depths in the MLI ... locations as follows:	
(a) Forward hemisphere on a line 45° from the centerline of the hemisphere.	T1,T2,T3,T4,T5,T6
(b) Aft hemisphere on a line 45° from the centerline of the hemisphere.	T7,T8,T9,T10,T11 T12
(c) Mid point of the cylindrical section.	T13,T14,T15,T16,T17,T18
(d) At a point where an inner tank support passes through the insulation at a constant radial distance from the support.	T19,T20,T21,T22 T23,T24
(e) The insulation within the simulated manhole access area."	T25,T26,T27,T28, T29,T30
"Twelve thermocouples ... on both the exterior of the vacuum jacket and the inner face shin of the vacuum jacket."	T44 through T55 T56 through T67
"Two thermocouples ... bonded to both exterior and interior surfaces of the girth ring."	T40,T41 T42,T43
"Two of the inner vessel supports ... (a minimum of four on each support)."	T31,T22,T32,T33,T34 T35,T36,T37,T38,T39
"The thermally modeled manhole and fluid line penetrations ... (a minimum of four on each penetration)."	T75,T76,T77,T78, T86,T87,T88: T69,T70,T71,T82, T93,T94,T95.
"The ullage gas temperature shall be measured at the outlet of the pressure vessel."	T72

## APPENDIX F

### SYSTEM EVALUATION TEST PLAN

TEST PLAN

LIGHTWEIGHT VACUUM JACKET FOR CRYOGENIC INSULATION

Distribution:	J. Allbee	4-1720	1E-11
	D. Barclay	2-5520	8C-45
	R. Hanna	2-5610	86-09
	R. Lominick	2-2405	1E-11
	F. Olsen	2-1833	4A-20

Test Engineer: H. M. Olden Date: 11-12-73  
H. M. Olden

Test Supervisor: W. A. Smith Date: 11/12/73  
W. A. Smith

TTS Test No.1 7308

Test Title: Lightweight Vacuum Jacket for Cryogenic Insulation

Contract No.: NASA Contract NAS3-15848

Test Location: Area 41, Pad 3

Reference:

- (1) Procedures Manual, Tulalip Test Site, D2-121703-1
- (2) Engineering Laboratories Safety Manual, D2-121354-1
- (3) SOP- 72K
- (4) Engineering Laboratories Test Practices Manual, D180-14822-1
- (5) EWA No. 25540, Task IV

1.0 PURPOSE

The purpose of this test is to demonstrate system performance of a half scale model of the Shuttle Orbiter OMS fuel tank with an evacuated multi-layer insulation (MLI) system. The performance of this system will be carefully evaluated in conjunction with the rigorous cyclic operational requirements of the orbiter.

This test plan describes in sufficient detail the test conditions, instrumentation, facilities, equipment, and safety and operating procedures to insure meeting the program test objectives in a safe and efficient manner.

2.0 OBJECTIVE

Testing to be accomplished under this test plan will furnish data that will be used to verify the feasibility of the evacuated MLI system design for the OMS fuel tank.

### 3.0 SCOPE

A half scale model of the Shuttle Orbiter OMS fuel tank will be designed, fabricated and shipped to Tulalip Test Site for testing on Pad 3, Area 41. The tank will be installed into an existing LH<sub>2</sub> supply, vent and disposal system with some modifications required. A radiant heat shroud will be fabricated and installed around the tank to provide temperature conditioning. A boiloff test system will be assembled and installed for determining heat flow into the tank. Three boiloff tests will be performed. The first one will be considered the baseline boil off rate. The second one will be performed after subjecting the tank to 100 external pressure cycles of 0 to 200,000 ft. equivalent altitude pressure and 100 thermal cycles of 100 to 350°F. The third boiloff test will be conducted while simulating a specified hydrogen leak into the vacuum annulus of the tanks insulation system. The test data will provide information to determine if any degradation in performance of the insulation system has occurred as a result of these operational environments.

### 4.0 TEST CONDITIONS

4.1 There will be three boiloff tests and two cyclic tests performed under this test plan in the sequence as follows:

- Baseline boil-off test
- Load cycling (altitude cycling) test - Area 34 .
- Thermal cycling test
- Post cycling boil-off test
- Constant GH<sub>2</sub> leak boil-off test.

4.2 The test conditions for the three boiloff tests will be as follows:

- Tank internal pressure 17.0 PSIA
- Tank vacuum annulus pressure  $\leq 2.0 \times 10^{-4}$  Torr
- Tank external pressure ambient
- Tank external temperature  $100^{\circ}\text{F} \pm 5^{\circ}\text{F}$
- Tank liquid hydrogen level  $\leq 3.0$  in. from Top

4.3 The test conditions for the load cycling (altitude cycling) test will be as follows:

- Tank internal pressure Same as external
- Tank vacuum annulus pressure  $\leq 2.0 \times 10^{-4}$  Torr
- Tank external pressure ambient to  $1.6 \times 10^{-1}$  Torr
- Tank external temperature ambient

## 4.3 continued

- Tank liquid hydrogen level No hydrogen
- Number of cycles 100 cycles

## 4.4 The test conditions for the thermal cycling test will be as follows:

- Tank internal pressure 15.0 PSIA
- Tank vacuum annulus pressure  $< 2.0 \times 10^{-4}$  Torr
- Tank external pressure ambient
- Tank external temperature 100°F to 350°F
- Tank liquid hydrogen level  $< 14.0$  in. from bottom
- Number of cycles 100 cycles

4.5 The half scale tank model will be constructed of two 90.0 inch diameter hemispheres connected to an 18.0 inch long cylindrical section. This tank will hold 2150 gallons of LH<sub>2</sub> when full. The inner vessel will have an operating pressure of 17.0 psia, a proof pressure of 40.0 psia, and a design burst pressure of 70.0 psia.5.0 FACILITIES

All testing will be conducted at the Boeing Company's Tulalip Test Site located approximately 45 miles north of Boeing's main Plant II headquarters. This facility has several well developed permanent test areas designed to handle a wide variety of hazardous and high noise level type tests. The cryogenic structural test facility, Test Area 41, is one of these specialized areas and will be used for this program. This area is devoted to structural materials testing which involve hazardous environments. These environments involve cryogenic propellants such as LH<sub>2</sub>, LO<sub>2</sub> and toxic propellants such as LF<sub>2</sub>, N<sub>2</sub>O<sub>4</sub> and Hydrazine.

In order to efficiently and safely handle this type of testing the area has three heavily reinforced concrete test cells dispersed on an arc located 130 ft. from the test control and data recording building. Each test pad is separated on this arc by 85 ft. Each pad has permanently installed wiring for remote control of the test setup and for remotely recording all test data. In addition, each pad has installed plumbing which provides He and N<sub>2</sub> pressurization and purge gases, plant air, water, and electrical power.

This facility has permanently installed LH<sub>2</sub> supply capacity of 14,000 gal. with a distribution and control system to each of the three test pads.

All testing for this program, except for altitude cycling at Area 34, will be performed on Pad 3 of Test Area 41.

## 6.0 TEST EQUIPMENT

A schematic of the test setup is shown in Figure F-1.

The LH<sub>2</sub> fill system will include a LH<sub>2</sub> conditioning cryostat mounted above the test tank. This vacuum insulated device will condition the LH<sub>2</sub> prior to entering the test tank so that the thermodynamic properties of the entering hydrogen are nearly the same as the hydrogen already in the test tank. This is to minimize the disturbance of thermal equilibrium during test tank refill operations. The pressure in this cryostat will be maintained at a constant differential pressure of 0.5 psi above the pressure in the test tank.

A radiant heat shroud will be fabricated and installed around the test tank so as to provide the required test tank external surface temperature. This shroud will be an aluminum sphere with quartz radiant heat lamps installed to the inside surface. Electrical power to the lamps will be regulated by a Research Incorporated ignitron electrical power control Unit. A closed loop temperature feedback system will be used to control the electrical power required to maintain the desired tank surface temperature. Redundant over-temperature safety circuits will be installed to prevent accidental overheating of the tank. Preheated gaseous nitrogen (GN<sub>2</sub>) will be used as a purge gas between the shroud and the external surface of the tank. This will aid in obtaining the desired surface temperature with minimum radiant heating power and also will exclude air, minimizing the chances of ignition of hydrogen.

The test tank internal pressure control system will consist of a closed loop pressure feedback servo valve system. The servo valve will control the flow of hydrogen boiloff gas from the tank so that the desired pressure in the tank will be maintained at a constant value. An Annin globe valve with the proper sized trim and an electro-hydraulic actuator will be used as the servo valve. The tank pressure feedback will be measured with a Baratron differential pressure transducer referenced to a very stable constant pressure source. This high-resolution differential pressure transducer will be used to sense very small positive or negative pressure variations in the cryogenic tank relative to a constant reference pressure. This reference pressure source will be a fixed volume of gas maintained at a constant temperature in an ice bath. A set point controller will be used to sense the electrical output signal of the high-resolution differential pressure transducer for control of the electrohydraulic pressure regulating servo valve. This complete system results in controlling the tank pressure to a constant value with respect to the reference pressure.

The flowrate of the hydrogen boiloff gas will be measured with a Hastings-Radist HF series mass flowmeter calibrated per documented procedures prepared by the Boeing Cal.-Cert. laboratories. A copper tube heat exchanger will be used to maintain the hydrogen boiloff gas at a constant temperature before entering the pressure regulating valve and flowmeter. In addition, the entire tank pressure regulating and boiloff flow measuring system will be contained in an insulated constant temperature controlled environmental box for thermal stability purposes.



## 6.0 continued

The gaseous hydrogen (GH<sub>2</sub>) leak system will consist of a needle valve, calibrated and installed to the test tank at the Kent Space Center, and a pure GH<sub>2</sub> pressure regulated supply system installed at Tulalip. The GH<sub>2</sub> leak rate needle valve will be calibrated at Kent by using helium leak rates corrected to yield the desired GH<sub>2</sub> leak rate. The calibrated leak will not be accessible when the LH<sub>2</sub> tank is loaded but final GH<sub>2</sub> leak rate adjustment will be provided by the remotely variable GH<sub>2</sub> supply pressure. Vacuum integrity will be insured by a remotely operated shut-off valve between the calibrated leak and the tank vacuum annulus. This valve will not be opened until the time of the constant GH<sub>2</sub> leak boil-off test. The section of plumbing between the calibrated leak and the vacuum shut-off valve will be evacuated prior to opening the annulus to the calibrated leak. This will purge this section of plumbing so as not to contaminate the tank annulus.

7.0 INSTRUMENTATION

An instrumentation list showing parameter, range, and accuracy is shown in Figure F-2. A block diagram of the instrumentation data acquisition system is shown in Figure F-3. All instrumentation equipment will have up-to-date calibration certifications issued by Boeing Quality Control Organization. These periodic calibrations are designated and performed per existing Q.C. documented standards. All calibrations are traceable to the National Bureau of Standards.

All pressures will be measured with strain gage pressure transducers. These will be located so that they will always be at ambient temperature thus requiring only an ambient calibration.

Test vessel temperatures above 0°F will be measured with chromel-constantan thermocouples with 150°F reference junction. Temperatures below 0°F will use a LN<sub>2</sub> reference junction. Temperatures measured in the vent tube will be set up as differential thermocouples with the LH<sub>2</sub> in the test tank as the reference temperature.

Hydrogen liquid level is determined by the use of carbon resistors which give a point indication as to the presence of liquid. In this application, the carbon resistor will be electrically driven so as to slightly heat up while in the presence of cold hydrogen gas. The increased heat dissipated as a result of submergence in the liquid hydrogen will cause a resistance change due to temperature which will be detected through a bridge circuit. This in turn will actuate a relay turning on an indicator light. The carbon resistor will be normally installed in a still-well (shield) to minimize false indications due to boiling liquid surface irregularities. Sufficient resistors will be installed in the test tank to adequately monitor liquid levels during thermal stabilization and boil-off testing. Power to carbon resistors will be turned off during stability period when boil-off data is taken.

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7.0 continued

The hydrogen boil-off gas flow rate will be measured with a Hastings-Radist, HF series, mass flowmeter. This will provide a continuous direct recording of mass flow rate from the tank. This meter with its laminar flow element will be calibrated per Boeing Cal-Cert laboratories documented procedures using hydrogen gas. The calibration will provide data as to actual standard cubic feet per minute of hydrogen gas vs millivolts output.

A MKS Baratron transducer will be used to monitor the pressure variations in the test tank. This device is a very high resolution differential pressure transducer which will be referenced to a stable constant pressure source.

The test tank vacuum annulus pressure will be monitored with the vac-ion pump current. The vac-ion pumps will be in continuous operation throughout the test program. A thermocouple gage and readout will be installed in the GH<sub>2</sub> leak system which could be opened to the annulus in case the annulus pressure increased beyond the effective range of the vac-ion pumps. This would provide an emergency means of monitoring the vacuum annulus pressure.

Test data will be recorded with a Non-Linear Systems (NLS) digital data recording system. This is a highly reliable, 200-channel recorder which records millivolt signal data on magnetic tape that can be listed or plotted on IBM-compatible computers. The system is designed to operate continuously 24 hours per day, seven days per week. The sample rate is 10 samples per second. System accuracy is considered to be 40 microvolts of 100 millivolts full scale. The data tape will be removed from the recording system at the end of each boil-off test for computer processing. The data and calculated values will be tabulated and/or plotted as required for each boil-off test performed.

Some of the data parameters will be displayed and recorded on potentiometer type strip chart recorders. This will provide continuous quick-look information for ease in determining system stability and satisfactory performance.

A test progress report, consisting of the test log, reduced data, brief test summary of any test deviations or anomalies, and photographs will be prepared after each of the three boil-off tests.

8.0 TEST PROCEDURES

- 8.1 Lift test tank/fixture assembly from transportation trailer with crane and place it on Pad 3, Area 41. This assembly will remain as a complete unit throughout the Task IV test program.
- 8.2 Move test tank/fixture assembly into test position and connect to LH<sub>2</sub> fill and conditioning cryostat.
- 8.3 Connect vac-ion pump power to facility source and monitor annulus vacuum.
- 8.4 Connect tank thermocouples to test facility instrumentation wiring.
- 8.5 Leak check tank connection by pressurizing with gaseous helium and using a helium mass spectrometer to sniff joint.
- 8.6 Install four sections of the spherical heat shroud around test tank and connect all power and control cables.
- 8.7 Connect gaseous hydrogen supply to the calibrated leak system, and purge system with gaseous hydrogen. R
- 8.8 Adjust all the test tank and cryogenic system instrumentation recorders for their proper zeros and calibration pulses with the system vent valves VH 351 and VH 354 open.
- 8.9 Purge LH<sub>2</sub> conditioning cryostat by flowing helium through VHe 303 and out VH 354 for 2 min.
- 8.10 Purge test tank by continuing flow as in 8.9 but open VH 363 and VH 351 and close VH 354 causing helium flow out the vent for 20 min.
- 8.11 While purging, set up thermal conditions by gradually energizing the heat shroud radiant lamps and nitrogen gas flow to slowly increase vacuum jacket external temperature to 100°F ± 5°F. Monitor vacuum annulus pressure while temperature is being increased and for a period of time after reaching final temperature until pressure is stable or decreasing. If vacuum annulus pressure exceeds  $2.0 \times 10^{-4}$  Torr discontinue temperature increase until pressure decreases. If vacuum annulus external surface temperature of 100°F ± 5°F cannot be reached without exceeding annulus pressure of  $2.0 \times 10^{-4}$  Torr, hold temperature at intermediate level until after completing tank LH<sub>2</sub> fill in paragraph 8.17.
- 8.12 Turn Pad 3 area condition lights to "RED" and notify Fire Department at completion of purging and after attaining external surface temperature conditions.
- 8.13 Set up the facility liquid hydrogen supply system per standard operating procedures for delivery of LH<sub>2</sub> to Pad 3.
- 8.14 Fill conditioning cryostat by opening vent VH 354 and pad LH<sub>2</sub> supply VH 301. The automatic liquid level control system will operate VH 301 as needed to maintain the cryostat full of LH<sub>2</sub>.
- 8.15 Set conditioning cryostat pressure control VH 365 to automatic and close VH 354 which will hold its pressure to 17.5 psia.

- 8.16 Fill test tank by opening VH 351 and then slowly opening VH 363 keeping the test tank pressure below 17.0 psia.
- 8.17 Stop test tank fill by closing VH 363.
- 8.18 Monitor test tank vacuum annulus pressure during and after filling with LH<sub>2</sub>. It should decrease during filling but should not be greater than  $2.0 \times 10^{-4}$  Torr. Complete temperature conditioning per para. 8.11.
- 8.19 Continue to refill test tank per 8.13 through 8.17 as tank liquid level decreases to 6.0 inches from top.
- 8.20 When time between refills approaches 6.0 hrs. then set VH 382 to automatic operation and close VH 351 which will hold test tank pressure at 17.0 psia.
- 8.21 Continue to monitor all data parameters and refilling tank per 8.19 until all data appears stabilized indicating thermal equilibrium has been obtained.
- 8.22 Boil-off data will be sampled at decreasing time intervals until considered stabilized for boil-off measurement data. At this time, 4 hrs of concentrated data will be taken. R
- 8.23 Stop conditioning cryostat fill by closing VH 301 and open vent VH 354 and drain VH 366.
- 8.24 Empty test tank by opening VH 363 and maintaining 17 to 20 psia in tank by closing VH 382 and intermittently opening helium purge valve VHE 352.
- 8.25 Test tank will be empty when LL1 light goes out and temperature TS1 begins to rise sharply.
- 8.26 Close VHE 352 and VH 363 and open VH 351. R
- 8.27 Empty conditioning cryostat by opening VH 366 and closing VH 354. It will be empty when LL9 light goes out and temperature TS2 begins rising.
- 8.28 Open VH 363 and close VH 366 and VH 354 to prepare system for helium purge and initial warmup. R
- 8.29 Open VHE 355 to purge system for 10 min. and then turn helium heater on to initiate system warmup.
- 8.30 When TS1 indicates -250°F switch purge gas supply from helium to nitrogen.
- 8.31 Continue gaseous nitrogen warmup until TS1 reaches 50°F. Then turn off heater and GN<sub>2</sub> and return test pad to green condition. Leave a slight positive pressure in system. R
- 8.32 Turn off radiant heat shroud and remove shroud from tank.
- 8.33 Disconnect instrumentation from test tank.
- 8.34 Disconnect vac-ion power from facility and put on portable supply.

- 8.35 Disconnect test tank/fixture assembly from conditioning cryostat and move out of test pad 3.
- 8.36 Load test tank/fixture assembly on transport trailer and move to Area 34 for altitude testing.
- 8.37 Lift test tank/fixture assembly from transport trailer at Area 34 and place on altitude chamber dolly.
- 8.38 Roll test tank/fixture assembly into altitude chamber and connect vac-ion pumps to facility power supply.
- 8.39 Close altitude chamber and evacuate to an equivalent altitude pressure of 200,000 ft. and return to ambient pressure.
- 8.40 Monitor test tank vacuum annulus pressure all during the pressure cycle. If the vacuum annulus pressure rises above the vac-ion pump capability (approximately  $2.4 \times 10^{-4}$  Torr) the test will be stopped. Procedures to reactivate the test will be developed at that time. R
- 8.41 Repeat this pressure cycle 100 times.
- 8.42 Open altitude chamber and remove test tank/fixture assembly for transport back to Area 41.
- 8.43 At Area 41 repeat steps 8.1 through 8.18 stopping test tank fill when liquid level reaches 14.0 inches from bottom of tank by closing VH 363.
- 8.44 Maintain this liquid level all during the succeeding thermal cycling test by opening VH 363 to refill test tank as needed. The thermal cycling will be interrupted during each refill operation.
- 8.45 Heating to 350°F must be approached cautiously. The possibility of out-gassing beyond the pumping capability of the vac-ion pumps exists. Heating should be controlled so that when the limitation of the vac-ion pump ( $\approx 2.4 \times 10^{-4}$  Torr) is approached the heating system is immediately shut off and cold nitrogen gas is introduced in the shroud area to restrict further out-gassing if possible. If the vacuum annulus pressure rises above the vac-ion pump capability, the test would be stopped. Procedures to reactivate the test would be developed at that time. R
- 8.46 Slowly apply heat manually for the first 100°F to 350°F temperature cycle while observing the precautions of para. 8.45.
- 8.47 Turn heat shroud controller to thermal cycle programmer and automatically cycle test tank surface temperature from 100°F to 350°F for a total of 100 cycles. Continually monitor test tank vacuum annulus pressure for any signs of degradation or system failure.
- 8.48 At end of thermal cycle test return heat shroud control to set point of 100°F for the following post cycling boil-off test.
- 8.49 Perform the post cycling boil-off test as in Section 8.16 through 8.22.
- 8.50 Evacuate the CH<sub>2</sub> leak system by opening VV 360 and pumping until P<sub>4</sub> stabilizes in pressure.
- 8.51 Close VV 360 and open VV 357 which will meter a calibrated flow of hydrogen gas into the vacuum annulus.

- 8.52 Set hydrogen supply pressure regulator VH 386 so that  $P_3$  is at a value TBD psia.
- 8.53 The flow rate of  $\text{GH}_2$  leak gas will be trimmed by adjusting supply pressure,  $P_3$ . Gradually increase  $P_3$  in steps, monitoring vacuum annulus pressure. Final  $P_3$  setting will be that resulting in a stable vacuum annulus pressure of TBD Torr.
- 8.54 Perform the constant  $\text{GH}_2$  leak boil-off test per section 8.19 through 8.23.
- 8.55 Close VV 357 to stop  $\text{GH}_2$  leak into vacuum annulus. Continue to refill test tank with  $\text{LH}_2$  per section 8.19 until vacuum annulus pressure returns to TBD Torr or for 3 days, whichever occurs first. Monitor vacuum annulus pressure. No boil-off rate or temperature data required.
- 8.56 Empty, purge and warm up test tank per 8.24 through 8.31.
- 8.57 Prepare tank for shipment per 8.32 through 8.35.
- 8.58 Load test tank/fixture assembly on transport trailer for shipment to the Kent Space Center.

## 9.0 SAFETY CONSIDERATIONS

The  $\text{LH}_2$  system to be used for this test poses no abnormal safety problem. The majority of the equipment, i.e., transfer lines, cryostats, vents and valves have all been used in past testing for several years. This specific test configuration has been used in nearly identical form for two previous test programs. The major difference for this program is in the size of the test tank. This tank will hold 2,150 gal. (1,270 lbs.) of  $\text{LH}_2$ .

Due to the integrity of the tank design, i.e., design, proof and operating pressures, failure of the test tank is considered highly improbable as long as overpressure conditions do not occur. Therefore, redundant pressure safety devices will be installed on the test tank so that maximum pressures will not be exceeded. Also, the vent system of the test tank has been sized such that in the event the vacuum fails in the insulation annulus the resultant higher boil-off rate will not cause the tank pressure to exceed a safe value.

In order to minimize the possible ignition of hydrogen gas that may accidentally be released, all spark producing equipment will either be explosion proof or be inert-gas purged. The space between the radiant heat lamp shroud and the external surface of the test tank will also be purged with nitrogen gas in order to eliminate any possible ignition in that area.

The radiant heat system will have a redundant temperature shutoff safety system. One system will be within the temperature control unit and the backup system will be a completely independent unit and will shut off all electrical power to the heating system in the event tank temperatures exceed a safe value.

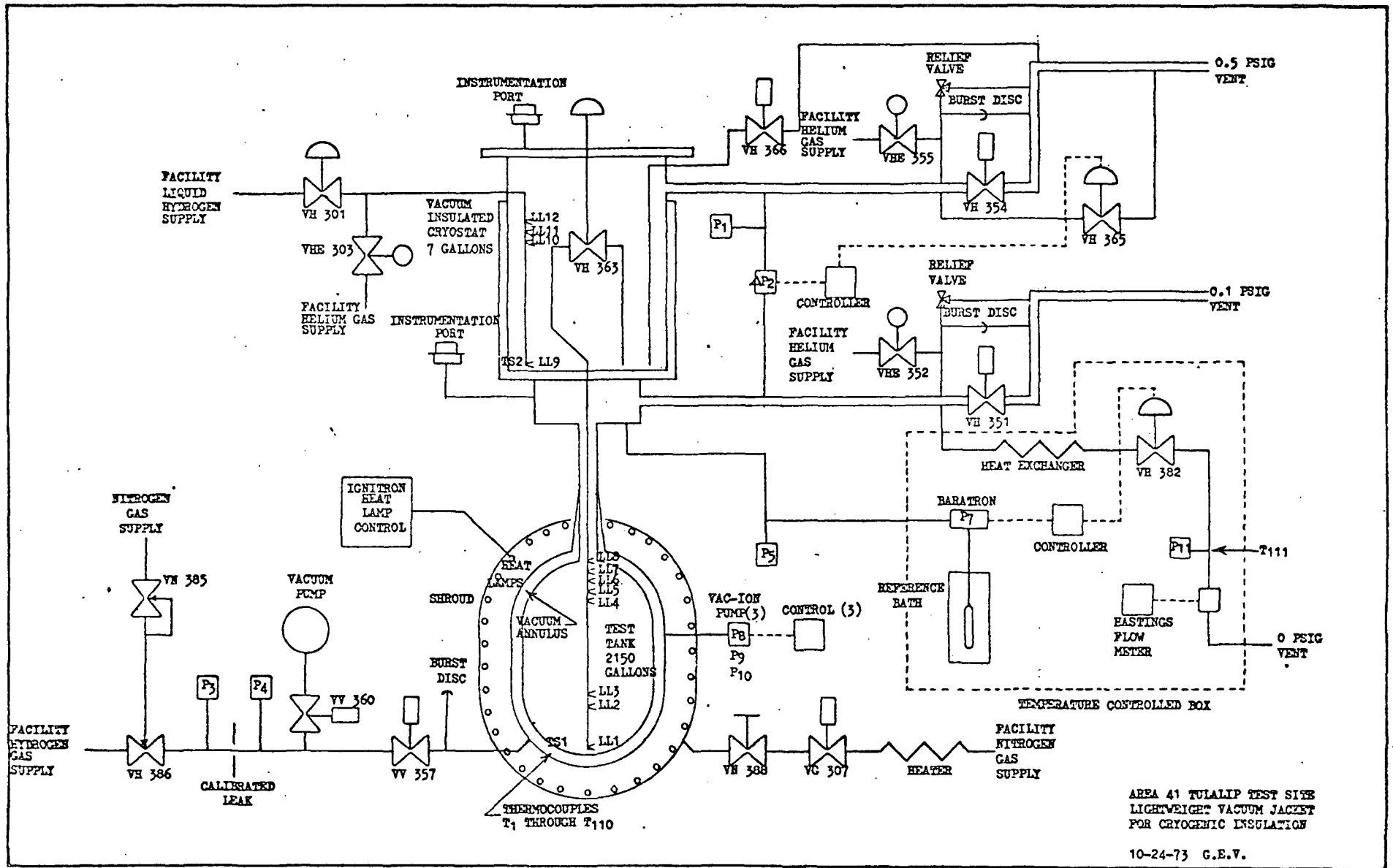
9.0 continued

In order to eliminate possible conflicts or confusion, and to increase the safety of operations, the assigned Test Area Engineer will have overall control of the test including safety procedures, cryogenic test procedures, malfunction procedures, and control of all personnel in the test area. During hydrogen test operations, all of the area of Test Area 41, east of the east end of the control house, will be excluded of all personnel.

10.0 MALFUNCTION PROCEDURES

A malfunction will be defined as any damage of equipment as a direct result of test operations conducted per this test plan. This could be caused by rupture and/or fire of/or from the test tank.

In the event of the above all hydrogen supply valves will be closed and the test tank will be emptied per 8.24 through 8.31, if possible, and the LH<sub>2</sub> conditioning cryostat emptied immediately after the test tank. The Fire Department will be notified and instructed to proceed to Test Area 41 and coordinate with the Test Area Engineer for further action. Helium purges of both tanks will then be turned on and maintained to warm up the test tanks and eliminate hydrogen. As soon as the hydrogen has been expelled and no over-pressurization of a tank exists the Fire Department may then be admitted to the pad area to expel any secondary fires that may exist and to cool down the test area. All main electrical power to the pad will also be turned off to eliminate electrical hazards. If a malfunction occurs the Test Area Engineer will immediately notify the Test Site Manager and the Ordnance Engineer. A written report will subsequently be prepared giving the details surrounding the incident. Corrective action will be taken before any further testing per this test plan is resumed.



REV LTR \_\_\_\_\_

BOEING NO. FIGURE F-1 Test No. 7308

Sh 12



PARAMETER			ACCURACY	RANGE		CALIBRATION			J-BOX	BALANCE PANEL	MAC PANEL	DATA SYSTEM CHAN			RECORDER CALIBRATION		
DESCRIPTION	NUMBER	S/N		RATED	USING	%	DIV	PSI									
PRESSURE																	
Cryostat	P1		1% P.S.		5 psig												
Cryostat vs Test Tank	P2		1% P.S.		5 psid												
Hydrogen Gas Supply	P3		1% P.S.		50 psig												
Hydrogen Gas Vacuum	P4		1% P.S.		$1 \times 10^{-5}$ Torr												
Test Tank	P5		1% P.S.		5 psig												
Barometric Pressure	P6		1% Reading		0-15 psia												
Test Tank vs Reference	P7		$\pm 0.0002$ psi		2 psid												
Vacuum Appliance	P8		1% P.S.		$1 \times 10^{-5}$ Torr												
VACUUM METER	P9		1% P.S.		$1 \times 10^{-5}$ Torr												
TEMPERATURE																	
Bottom Test Tank	TS 1		Indication		-423°P												
Cryostat	TS 2		"		"												
LIQUID LEVEL																	
Bottom Test Tank	LL 1		.1 Inch		On/Off												
Top Test Tank	LL 2		↑		↑												
Top Test Tank	LL 3																
Top Test Tank	LL 4																
Top Test Tank	LL 5																
Top Test Tank	LL 6																
Top Test Tank	LL 7																
Top Test Tank	LL 8																
Bottom Cryostat	LL 9																
Top Cryostat	LL 10																
Top Cryostat	LL 11		↓		↓												
Top Cryostat	LL 12		.1 Inch		On/Off												
FLOW METER																	
			3% of Reading		2.5 SCFM												

	INITIALS	DATE	REV BY INITIALS	DATE	TITLE	MODEL
CALC.	GEV	10/26/68			INSTRUMENTATION LIST AREA 41 LIGHTWEIGHT VACUUM JACKET FOR CRYOGENIC INSULATION	
CHECK						
APPD						
APPD						

PARAMETER		ACCURACY	RANGE		CALIBRATION	J-BOX	MAC PANEL	DATA SYSTEM CRAN		RECORDER CALIBRATION	
DESCRIPTION	NUMBER				REFERENCE						
<b>THERMOCOUPLE</b>											
Surface &	T 1	± 2°F	-300 to	+350°F							
Buried MLI	T 2										
Upper Heml	T 3										
	T 4										
	T 5										
	T 6										
Surface &	T 7	± 2°F	-300 to	+350°F							
Buried MLI	T 8										
Lower Heml	T 9										
	T 10										
	T 11										
	T 12										
Surface &	T 13	± 2°F	-300 to	+350°F							
Buried MLI	T 14										
Blanket	T 15										
Girth	T 16										
	T 17										
	T 18										
Surface &	T 19	± 2°F	-300 to	+350°F							
Buried MLI	T 20										
Upper Strap	T 21										
	T 22										
	T 23										
	T 24										
Surface &	T 25	± 2°F	-300 to	+350°F							
Buried MLI	T 26										
Lower Polar	T 27										
Cap	T 28										
	T 29										
	T 30										

	INITIALS	DATE	REV BY	DATE	TITLE	MODEL
CALC.	GEV	10/26/8			INSTRUMENTATION LIST AREA 41 LIGHTWEIGHT VACUUM JACKET FOR CRYOGENIC INSULATION	
CHECK						
APPD						
APPD						

REV LTR \_\_\_\_\_  
 US 4011 1000 REV. 2/65

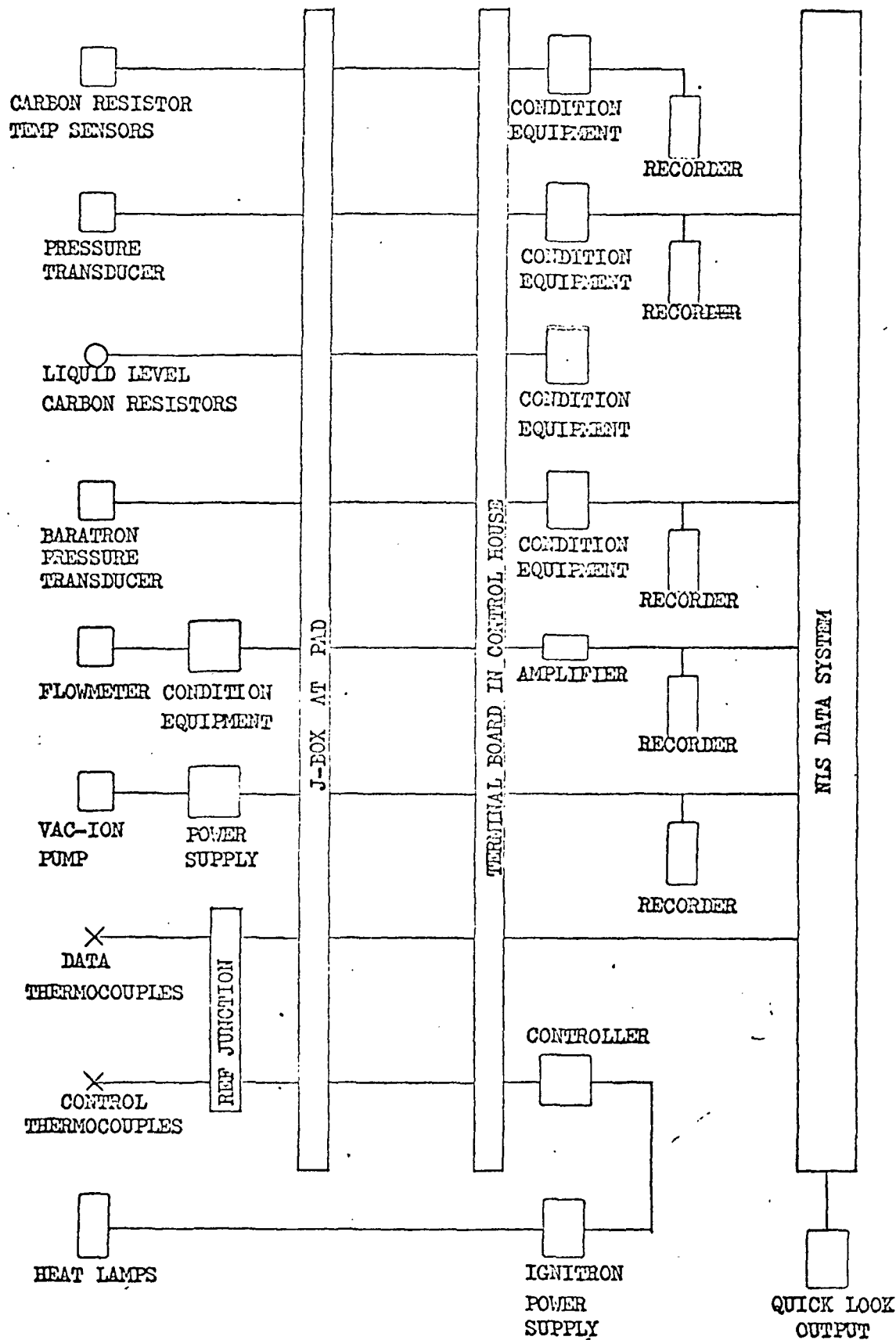
**BOEING** NO. FIGURE 2 - Test No. 7308  
 Sh 14

PARAMETER		ACCURACY	RANGE	CALIBRATION	J-BOX	MAC PANEL	DATA SYSTEM CHAN	RECORDER CALIBRATION
DESCRIPTION	NUMBER			REFERENCE				
THERMOCOUPLE								
Support Strip	T31	$\pm 2^{\circ}\text{F}$	-300° to +350°F					
	T32							
	T33							
	T34							
Left Strip	T35	$\pm 2^{\circ}\text{F}$	-300° to +350°F					
	T36							
	T37							
	T38							
	T39							
Surface Ex	T40	$\pm 2^{\circ}\text{F}$	+60° to +350°F					
Girth Ring	T41							
Surface In	T42	$\pm 2^{\circ}\text{F}$	+60° to +350°F					
Girth Ring	T43							
Vac Jac	T44	$\pm 2^{\circ}\text{F}$	+60° to +350°F					
Exterior	T45							
	T46							
	T47							
	T48							
	T49							
	T50							
	T51							
	T52							
	T53							
	T54							
	T55							
Vac Jac	T56	$\pm 2^{\circ}\text{F}$	+60° to +350°F					
Interior	T57							
	T58							
	T59							
	T60							

	INITIALS	DATE	REV BY	INITIALS	DATE	TITLE	MODEL
CALC.	GSY	10/26/6				INSTRUMENTATION LIST AREA 41 LIGHTWEIGHT VACUUM JACKET FOR CRYOGENIC INSULATION	
CHECK							
APPD							
APPD							

REV LTR _____	<b>DOEINC</b> NO. <b>FIGURE F-2 - Test No. 7508</b>
US 6011 3000 REV. 2/65	SH 10





AREA 41 TULALIP TEST SITE  
LIGHTWEIGHT VACUUM JACKET  
FOR CRYOGENIC INSULATION

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A Division of Rockwell International  
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- 1 Space Division  
A Division of Rockwell International  
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Downey, California  
  
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- 1 Rocket Research Corporation  
Willow Road At 116th Street  
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Attn: Library
- 1 Stanford Research Institute  
333 Ravenswood Avenue  
Menlo Park, California 94025  
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- 1 Susquehanna Corporation  
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Shirley Highway & Edsall Road  
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1 Thiokol Chemical Corporation  
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TRW Systems Inc.  
1 Space Park  
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1 TRW  
- TAPCO Division  
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1 United Aircraft Corporation  
Corporation Library  
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1 United Aircraft Corporation  
United Technology Center  
P. O. Box 358  
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Pratt & Whitney Division  
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P. O. Box 2691  
West Palm Beach, Florida 33402

1 Attn: Library  
1 G. B. Cox

1 Vickers Incorporated  
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Troy, Michigan  
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1 Vought Astronautics  
Box 5907  
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